

ENERGY CONSERVATION USING
A SOIL HEAT EXCHANGER-STORAGE SYSTEM
IN A COMMERCIAL TYPE GREENHOUSE

by

Hervé Bernier

A thesis submitted to the Faculty of Graduate
Studies and Research in partial fulfillment
of the requirements for the degree of
Master Science

Department of Agricultural Engineering,
Macdonald College,
McGill University,
Montreal, Canada.

© December 1987

ENERGY CONSERVATION USING A SOIL HEAT EXCHANGER-STORAGE SYSTEM

ABSTRACT

Because of the low thermal mass of a typical commercial greenhouse, the surplus heat captured by a greenhouse during the day is mostly ventilated to the outside, while during the night, auxiliary heat is required to maintain proper air temperature.

The soil within a greenhouse represents an important thermal mass, which is underused. By providing a suitable heat exchange surface, soil can become a relatively low cost storage material. Further, it has been shown that a higher root zone temperature will lead to a beneficial effect with regard to crop yield, while reducing the energy consumption. Therefore, soil heat storage is advantageous for soil grown crops.

A heat exchanger-storage system, using soil as a storage medium, was designed and built in a NORDIC type greenhouse. The greenhouse oriented east-west, is located at the Institut de technologie agro-alimentaire in La Pocatière, Québec, Canada.

The heat exchanger-storage system made of 26 non-perforated, corrugated plastic drainage pipes, 102 mm in diameter, was buried in the soil. Two rows of 13 pipes, 12 m long, were buried at 450 mm and 750 mm respectively. The pipes run parallel to the longitudinal axis of the greenhouse, and are spaced 450 mm apart. A 0.75 kW blower was provided to circulate hot air collected in the greenhouse,

through the pipes at a flowrate of $0.91 \text{ m}^3/\text{s}$. The heat stored was recovered both by convection at the soil surface and by forced convection in the exchanger pipes.

The storage has a seasonal temperature fluctuation of 10°C . The system performance seems to be more influenced by greenhouse air temperature than by incident solar radiation. Values for the average coefficient of performance and pipe convective heat transfer coefficient were 3.6 and $12 \text{ W/m}^2\cdot\text{K}$ respectively. Approximately 30 % of the heat recovered from the storage is exchanged by convection at the soil surface. Results indicate that solar energy contributed to 58 % of the heating requirements from February to June and from September to December 1986. This contribution represents 33 % energy conservation. Further, the system seems to have a beneficial effect on crop growth and yield. For two consecutive growing seasons, soil grown tomato plants have produced yields of 25 kg/m^2 ; by comparison, average yields of 15 kg/m^2 are encountered in commercial greenhouses in Quebec. The payback period for the system is estimated at two years, depending on the cost and crop management practice.

RESUME

La faible masse thermique des serres de type commercial pose un problème de gestion efficace de l'énergie thermique qu'elles consomment. Pendant le jour, les surplus d'énergie solaire interceptés sont évacués à l'extérieur alors que pendant la nuit, un apport énergétique externe doit être fourni.

Le sol d'une serre constitue une masse thermique importante mais sous utilisée. En augmentant la surface d'échange de chaleur, le sol pourrait constituer un matériau de stockage de chaleur relativement peu coûteux. De plus, il a été démontré qu'en chauffant le sol, il est possible d'augmenter les rendements des plantes cultivées à même le sol tout en diminuant la consommation d'énergie; le stockage de chaleur dans le sol pourrait permettre de profiter de ce double avantage.

Un système combiné d'échange et de stockage de chaleur a été mis en place dans une serre de type "NORDIQUE", orientée est-ouest et construite à l'Institut de technologie agro-alimentaire de La Pocatière. Le système comprend 26 tuyaux de plastique ondulé, non perforé, de 10 cm de diamètre, enfouis à 45 et 75 cm de la surface respectivement; l'espacement latéral est de 45 cm entre les tuyaux. L'air chaud est recueilli au faite de la serre et est poussé à travers les tuyaux par un ventilateur ayant un débit de $0.91 \text{ m}^3/\text{s}$. La chaleur accumulée dans le sol peut être récupérée.

par conduction vers la surface et par convection forcée dans les tuyaux.

Des fluctuations saisonnières de l'ordre de 10°C ont été enregistrées au niveau de l'ensemble de la masse thermique. La performance du système semble plus influencée par la température de l'air que par le rayonnement solaire. Le coefficient de performance moyen associé au système et le coefficient moyen d'échange convectif dans l'échangeur ont été respectivement de 3.6 et $12 \text{ W/m}^2\text{K}$. Environ 30 % de l'énergie est récupérée directement à la surface du sol. Les résultats indiquent que l'énergie solaire a contribué à 58 % des besoins thermiques de la serre de février à juin et de septembre à décembre 1986. L'économie d'énergie réalisée serait de 33 % pour cette même période. De plus, le système pourrait avoir un effet bénéfique sur les rendements des cultures. Des rendements de l'ordre de 25 kg/m^2 ont été obtenus pour une production de tomates effectuée sur deux saisons consécutives; par comparaison, des rendements moyens de 15 kg/m^2 sont obtenus chez les producteurs en serre du Québec. Le délai de récupération de l'investissement pour ce système pourrait être inférieur à deux ans suivant les coûts et le type de régie de production.

ACKNOWLEDGEMENT

The author wishes to express his gratitude to:

the Institut de technologie agro-alimentaire de La Pocatière, the Ministère de l'Agriculture, des Pêcheries et de l'Alimentation du Québec, and Employment and Immigration Canada for their financial support;

Dr. G.S.V. Raghavan, Professor of Agricultural Engineering at McGill University and Dr. Jean Paris, Professor of Chemical Engineering at the Ecole Polytechnique, for their advice, guidance and positive criticism in conducting the research work and in the preparation of this manuscript;

the Corporation des Agronomes (La Pocatière region) and the Fondation François Pilote Inc. for their help and support during the different phases of this project;

Nicole Gagné, Lynda Tremblay, Claire Dubé, Raymond Létourneau, Richard Levesque, André Tremblay, Gilles Lapierre, Denis Bellerose, Yvon Blanchet, François Chalifour, Sylvain Bourgelas, Ghislain Massé, André Dupéré, Alain Lebel, André Pelletier and Wilbrod Pelletier, for the technical help provided during the research phase, and finally to my wife, Angèle Bernier for typing the manuscript, her understanding and her precious collaboration.

TABLE OF CONTENTS

	PAGE
ABSTRACT.....	ii
RESUME.....	iv
ACKNOWLEDGEMENT.....	vi
LIST OF TABLES.....	x
LIST OF FIGURES.....	xi
LIST OF SYMBOLS AND ABBREVIATIONS.....	xiv
CHAPTER I. INTRODUCTION.....	1
1.1 Problem Description.....	1
1.2 Objectives.....	2
1.3 Scope of the Study.....	3
CHAPTER II. REVIEW OF LITERATURE.....	4
2.1 Review.....	4
2.2 Summary.....	12
CHAPTER III. THEORETICAL CONSIDERATIONS.....	13
CHAPTER IV. MATERIALS AND METHODS.....	24
4.1 Experimental Setup.....	24
4.2 Procedures.....	36
4.2.1 Soil Physical Properties.....	36
4.2.2 Air Velocity.....	36
4.2.3 Overall Heat Exchange Coefficients.....	39

4.2.3.1 Heat Loss Coefficients.....	39
4.2.3.2 Convective Heat Transfer Coefficient	39
4.2.4 Electrical Energy Used.....	41
4.2.5 Daily Performance.....	41
4.2.6 Operating Modes.....	43
4.2.6.1 Alternate cycles of active storage and recovery.....	43
4.2.6.2 Active storage and passive recovery	43
4.2.6.3 Continuous Operation.....	43
4.2.7 Observations on the crops.....	43
4.2.8 Payback Estimation.....	44
CHAPTER V. RESULTS AND DISCUSSION.....	49
5.1 Soil Physical properties.....	49
5.1.1 Soil Texture.....	49
5.1.2 Soil Wet Bulk Density.....	49
5.1.3 Soil Specific Heat.....	49
5.1.4 Soil Water Content.....	52
5.2 Air Velocity.....	52
5.2.1 Heat Exchanger.....	52
5.2.2 Heater Outlets.....	52
5.3 Heat Exchange Coefficients.....	56
5.3.1 Greenhouse Heat Loss Coefficients.....	56
5.3.2 Convective Heat Transfer Coefficient.....	58
5.4 Electrical Energy Used.....	59
5.5 System Performance.....	59
5.5.1 Heat Storage and Recovery.....	60
5.5.2 Dehumidification-humidification of the air.	78

5.5.3 Heat Exchanger.....	80
5.5.4 Greenhouse Energy Consumption.....	90
5.6 Tomato Crop.....	97
5.7 Payback Period.....	99
CHAPTER VI. CONCLUSIONS.....	101
CHAPTER VII. RECOMMENDATIONS.....	106
REFERENCES.....	107
APPENDICES.....	112
A. Linear regression for the estimation of the average air temperature at the heater outlets.....	113
B. Data and results obtained from a heat loss test...	116
C. Computer program for the calculation of the overall heat loss coefficient.....	120
D. Daily Results.....	122
E. Example of raw, treated and compiled data.....	131
F. Computer Program.....	138

LIST OF TABLES

	PAGE
Table 4.1 Experimental greenhouse characteristics.....	25
Table 4.2 Heat exchanger pipe characteristics.....	26
Table 4.3 Data acquisition and control system characteristics.....	30
Table 4.4 Schedule of events for equipment operation..	43
Table 4.5 Cost estimates for the payback period.....	46
Table 4.6 Payback period scenarios.....	47
Table 5.1 Texture of the soil bottom layer.....	50
Table 5.2 Texture of the soil top layer.....	51
Table 5.3 Soil wet bulk density.....	51
Table 5.4 Soil specific heat.....	53
Table 5.5 Soil water content.....	53
Table 5.6 Heat exchanger air velocity and flowrate....	54
Table 5.7 Heater outlets air velocity and flowrate....	54
Table 5.8 Voltage, current and electrical power.....	59
Table 5.9 Fluctuation of the heat storage and recovery parameters.....	77
Table 5.10 Effect of the temperature differential on the COP and exchanger efficiency.....	83
Table 5.11 Payback period estimation.....	100
Appendix	
Table A.1 Air temperatures at the heater outlets.....	114
Table A.2 SAS linear regression program for the estimation of the heater outlet mean air temperature.....	115
Table A.3 Summarized SAS regression output for the estimation of the heater outlet mean air temperature.....	115

LIST OF FIGURES

	PAGE
Figure 4.1 Greenhouse heat exchanger-storage system...	27
Figure 4.2 Greenhouse main equipments.....	28
Figure 4.3 Microcomputer system installed in the experimental greenhouse.....	31
Figure 4.4 Temperature sensors installed in the northeast quadrant of the greenhouse.....	33
Figure 4.5 Temperature sensors installed in the western part of the greenhouse.....	34
Figure 4.6 Analog Device AD-590 temperature sensor....	35
Figure 4.7 Stratagauge Trowler model no. 3411B.....	37
Figure 4.8 Pitot tube apparatus.....	38
Figure 4.9 Flowchart for the computation of the overall heat loss coefficient.....	40
Figure 4.10 Flowchart for the computation of the daily performance.....	42
Figure 4.11 Layout of tomato plants in the greenhouse..	48
Figure 5.1 Wind effect on the overall heat loss coefficient.....	57
Figure 5.2 Average thermal mass mean temperature fluctuation for the periods of September through November 1985 and 1986, and February through July 1986.....	61
Figure 5.3 Soil daily temperature variation during the storage phase in the spring of 1986.....	62
Figure 5.4 Soil daily temperature variation in the fall of 1985 with passive heat recovery.....	64
Figure 5.5 Soil longitudinal temperature profile for mid-spring.....	65
Figure 5.6 Soil longitudinal temperature profile at the beginning of summer.....	66
Figure 5.7 Soil longitudinal temperature profile during heat recovery in the fall of 1986.....	68

Figure 5.8	Soil longitudinal temperature profile at the end of fall.....	69
Figure 5.9	Daily variation of the temperature differential between the heat exchanger inlet and outlet.....	70
Figure 5.10	Air longitudinal temperature profile in the exchanger pipes, after 4 hours of heat storage.....	72
Figure 5.11	Air longitudinal temperature profile in the exchanger pipes, after 8 hours of heat storage.....	73
Figure 5.12	Air longitudinal temperature profile in the exchanger pipes after 8 hours of heat recovery.....	74
Figure 5.13	Soil-air temperature relationship.....	75
Figure 5.14	Air relative humidity daily variation.....	79
Figure 5.15	Daily variation of the coefficient of performance.....	81
Figure 5.16	Effect of the temperature differential between the air at the blower inlet and the soil, on the COP.....	84
Figure 5.17	Effect of the temperature differential between the soil and the greenhouse air, on the COP.....	85
Figure 5.18	Effect of the temperature differential between the air at the blower inlet and the soil, on the efficiency.....	87
Figure 5.19	Effect of the temperature differential between the soil and the greenhouse air on the efficiency.....	88
Figure 5.20	Ratio of the heat recovered passively to the total heat recovered for the periods of February through July 1986 and September through November 1986.....	89
Figure 5.21	Variation of the heat exchanger efficiency for the periods of September through November 1985 and 1986, and February through July 1986.....	91

Figure 5.22	Effect of solar radiation on the solar contribution to the greenhouse heat load.....	92
Figure 5.23	Effect of the greenhouse air temperature on the solar contribution to the greenhouse heat load.....	94
Figure 5.24	Effect of the soil temperature on the solar contribution to the greenhouse heat load....	95
Figure 5.25	Variation in the contribution to the greenhouse heat load.....	96
Figure 5.26	Tomato crop grown in the experimental greenhouse.....	98

LIST OF SYMBOLS AND ABBREVIATIONS

A	: soil surface absorptivity (0.82, Kreith and Kreider, 1978)
AF	: greenhouse floor surface area (71.49 m ²)
AG	: greenhouse roof and walls surface area (174.25 m ²)
AI	: heater air inlet, cross sectionnal area (0.162 m ²)
AHU1	: absolute humidity difference between the air at the exchanger inlet and outlet for the first row of pipes (kg)
AHU2	: absolute humidity difference between the air at the exchanger inlet and outlet for the second row of pipes (kg)
AHUC	: absolute humidity difference between the air at the exchanger inlet and outlet cumulated over 24 hours (kg)
AHUI	: air absolute humidity at the exchanger inlet (kg vapor/m ³ humid air)
AHUD1	: air absolute humidity at the exchanger outlet for the first pipe row (kg vapor/m ³ humid air)
AHU02	: air absolute humidity at the exchanger outlet for the second pipe row (kg vapor/m ³ humid air)
ASHRAE	: American Society of Heating, Refrigerating and Air-Conditioning Engineers
AT	: surface area of a pipe (5.31 m ²)
c	: regression constant
CAF	: air specific heat (kJ/kg·°C)
CDS ₁	: dry soil specific heat (kJ/kg·°C) (CDS ₁ = 0.7678, CDS ₂ = 0.6815)
COP	: Coefficient of performance
COPM	: coefficient of performance (average over 24 hours)
COPR	: coefficient of performance for the recovery phase
COPS	: coefficient of performance for the storage phase

C_p : fluid specific heat ($\text{kJ/kg}\cdot^\circ\text{C}$)
 ($C_{p_s} = 1.0465$ $C_{p_v} = 1.84$)

C_{PTI} : air specific heat at the exchanger inlet
 ($\text{kJ/kg}\cdot^\circ\text{C}$)

C_{PTO1} : air specific heat at the exchanger outlet for the
 first row pipes ($\text{kJ/kg}\cdot^\circ\text{C}$)

C_{PTO2} : air specific heat at the exchanger outlet for the
 second row of pipes ($\text{kJ/kg}\cdot^\circ\text{C}$)

C_{PTM1} : air mean specific heat in the exchanger first row
 of pipes ($\text{kJ/kg}\cdot^\circ\text{C}$)

C_{PTM2} : air mean specific heat in the exchanger second row
 of pipes ($\text{kJ/kg}\cdot^\circ\text{C}$)

C_{PVH} : specific heat of the evacuated humid air
 ($\text{kJ/kg}\cdot^\circ\text{C}$)

C_{PVQ} : Conseil des productions végétales du Québec

C_s : humid air specific heat ($\text{kJ/kg}\cdot^\circ\text{C}$)

C_{s_i} : moist soil specific heat ($\text{kJ/kg}\cdot^\circ\text{C}$)

CW : water specific heat ($4.179 \text{ kJ/kg}\cdot^\circ\text{C}$)

DA : density of humid air in the greenhouse (kg/m^3)

DAF : density of humid air in the heater (kg/m^3)

DAV : density of humid air evacuated (kg/m^3)

$DATI$: density of humid air at the exchanger inlet
 (kg/m^3)

$DATO1$: density of humid air at the exchanger outlet for
 the first row of pipes (kg/m^3)

$DATO2$: density of humid air at the exchanger outlet for
 the second row of pipes (kg/m^3)

$DATM1$: mean density of humid air in the exchanger first
 row of pipes (kg/m^3)

$DATM2$: density of humid air at the exchanger outlet for
 the second row of pipes (kg/m^3)

$DHU1$: absolute humidity removed from the air in the ex-
 changer first row of pipes (kg)

DHU2 : absolute humidity removed from the air in the ex-
 changer second row of pipes (kg)
 DHUC : absolute humidity removed from the air circulated
 in the exchanger, cumulated over 24 hours (kg)
 DIF. : differential
 DSS_i : soil layer density (kg/m³)
 DSS₁ = 1340, DSS₂ = 1736
 EA : vapor partial pressure (kPa)
 ECC : thermal curtain energy conservation factor (0.72)
 ERE : heat exchanger efficiency in the recovery mode
 EREM : mean heat exchanger efficiency in the recovery
 mode
 ES : saturated vapor pressure (kPa)
 ESAF : saturated vapor pressure of the air circulated in
 the heater (kPa)
 ESAV : saturated vapor pressure of the air evacuated
 (kPa)
 ESE : heat exchanger efficiency in the storage mode
 ESEM : mean heat exchanger efficiency in the storage mode
 ESTI : saturated vapor pressure at the exchanger inlet
 (kPa)
 ESTO1 : saturated vapor pressure at the exchanger outlet
 for the first row of pipes (kPa)
 ESTO2 : saturated vapor pressure at the exchanger outlet
 for the second row of pipes (kPa)
 FF : heater contribution to the greenhouse heat load
 FG : top soil surface area (64.4 m²)
 FRT1 : air flowrate in the exchanger first row of pipes
 (0.454 m³/s)
 FRT2 : air flowrate in the exchanger second row of pipes
 (0.464 m³/s)
 FRH : air flowrate in the heater (0.804 m³/s)

HCE : convective heat transfer coefficient at the soil surface ($0.0054 \text{ kW/m}^2 \cdot ^\circ\text{C}$)
HCT : heat exchanger convective heat transfer coefficient ($\text{kW/m}^2 \cdot ^\circ\text{C}$)
HCTM : mean heat exchanger convective heat transfer coefficient ($\text{kW/m}^2 \cdot ^\circ\text{C}$)
HT : air enthalpy (kJ/kg)
HTI : air enthalpy at the exchanger inlet (kJ/kg)
HTO1 : air enthalpy at the exchanger outlet for the first row of pipes (kJ/kg)
HTO2 : air enthalpy at the exchanger outlet for the second row of pipes (kJ/kg)
HU : air absolute humidity (kg water vapor/kg dry air)
HUC : absolute humidity of the air cumulated over 24 hours (kg water vapor/kg dry air)
HUF : absolute humidity of the air circulated through the heater (kg water vapor/kg dry air)
HUV : absolute humidity of the air evacuated to the outside (kg water vapor/kg dry air)
HUM1 : absolute humidity added to the air passing in the exchanger first row of pipes (kg)
HUM2 : absolute humidity added to the air passing in the exchanger second row of pipes (kg)
HUTI : air absolute humidity at the exchanger inlet (kg/kg dry air)
HUTO1 : air absolute humidity at the exchanger outlet for the first row of pipes (kg/kg dry air)
HUTO2 : air absolute humidity at the exchanger outlet for the second row of pipes (kg/kg dry air)
IBR : current used by the burner (5.9 A)
IC : current used by the blower (4.9 A)
ISC : solar radiation flux (mW/cm^2)
ISH : incident solar radiation (kJ/m^2)

ISHC : incident solar radiation cumulated over 24 hours
 (kJ/m^2)

IV : current used by the ventilator in low speed mode
 (3.2 A)

LF : artificial lighting system contribution to the
 greenhouse heat load

LH : water latent heat of vaporization (2501 kJ/kg)

M_{air} : mass of air (kg)

MA : air molecular weight (0.028966 kg/mole)

MW : water molecular weight (0.018016 kg/mole)

P : atmospheric pressure (101.325 kPa)

PE : greenhouse perimeter (37.2 m)

PF : heater power averaged on an hourly basis (kW)

PFM : rated heater power (36 kW)

PL : artificial lighting system power (1.9 kW)

QAUX : auxiliary heat used by the greenhouse (kJ)

QBR : electrical energy not used by the burner (kJ)

QBRC : electrical energy not used by the burner cumulated
 over 24 hours (kJ)

QC : electrical energy used by the blower (kJ)

QCC : electrical energy used by the blower cumulated
 over 24 hours (kJ)

QCF : convective heat transfer at the soil surface (kJ)

QES : sensible and latent heat exchanged in the pipes
 during storage (kJ)

QES1 : sensible and latent heat exchanged in the first
 row of pipes during storage (kJ)

QES2 : sensible and latent heat exchanged in the second
 row of pipes during storage (kJ)

QEST : sensible and latent heat exchanged in both row of
 pipes during storage (kJ)

QUESTC : sensible and latent heat exchanged in the pipes
 during storage cumulated over 24 hours (kJ)

QET1 : sensible heat exchanged in the first row of pipes
 (kJ)

QET2 : sensible heat exchanged in the second row of pipes
 (kJ)

QETR1 : sensible heat recovered from the exchanger first
 row of pipes (kJ)

QETR2 : sensible heat recovered from the exchanger second
 row of pipes (kJ)

QETRT : sensible heat recovered from the exchanger (kJ)

QETRTC : sensible heat recovered from the exchanger cumula-
 ted over 24 hours (kJ)

QETS1 : sensible heat lost from the air in the exchanger
 first row of pipes (kJ)

QETS2 : sensible heat lost from the air in the exchanger
 second row of pipes (kJ)

QETST : sensible heat lost from the air in the exchanger
 (kJ)

QETSTC : sensible heat lost from the air in the exchanger
 cumulated over 24 hours (kJ)

QEX : sensible and latent heat exchanged in the pipes
 (kJ)

QEX1 : sensible and latent heat exchanged in the first
 row of pipes (kJ)

QEX2 : sensible and latent heat exchanged in the second
 row of pipes (kJ)

QF : heat supplied by the heater (kJ)

QFC : heat supplied by the heater cumulated over 24
 hours (kJ)

QGF : heat gained from the soil by convection at the
 soil surface (kJ)

QGFC : heat gained from the soil by convection at the
 soil surface cumulated over 24 hours (kJ)

QGHL : greenhouse structure heat loss (kJ)
 QIR : heat gained by the soil from the irrigation water (kJ)
 QL : effective heat produced by the artificial lighting system (kJ)
 QLE : excess heat produced by the artificial lighting system (kJ)
 QLU : usable heat produced by the artificial lighting system (kJ)
 QLUC : usable heat produced by the artificial lighting system cumulated over 24 hours (kJ)
 QRF : heat recovered by convection at the soil surface (kJ)
 QRFC : heat recovered by convection at the soil surface cumulated over 24 hours (kJ)
 QRS : direct solar heat gain at the soil surface (kJ)
 QRSC : direct solar heat gain at the soil surface cumulated over 24 hours (kJ)
 QRT : sensible and latent heat recovered by the air passing in the exchanger pipes (kJ)
 QRT1 : sensible and latent heat recovered by the air passing in the exchanger first row of pipes (kJ)
 QRT2 : sensible and latent heat recovered by the air passing in the exchanger second row of pipes (kJ)
 QRTT : sensible and latent heat recovered by the air passing in both pipe rows of the exchanger (kJ)
 QRTTC : sensible and latent heat recovered by the air passing in the exchanger cumulated over 24 hours (kJ)
 QSHL : greenhouse soil heat loss to the outside (kJ)
 QSS : soil heat exchanged (kJ)
 QSS1 : heat exchanged by the soil surface (kJ)
 QSS2 : heat exchanged by the soil top layer (kJ)
 QSS3 : heat exchanged by the soil bottom layer (kJ)

QSST : total heat exchanged by the soil thermal mass (kJ)
 QSSTG : total heat gain by the soil thermal mass (kJ)
 QSSTL : total heat lost by the soil thermal mass (kJ)
 QTHL : greenhouse total heat loss (kJ)
 QTHLC : greenhouse total heat loss cumulated over 24 hours (kJ)
 QV : electrical energy not used by the ventilators (kJ)
 QVC : electrical energy not used by the ventilators cumulated over 24 hours (kJ)
 QVHL : sensible heat loss by ventilation (kJ)
 R : universal gas constant (0.008314 kJ/mole·K)
 RE : heat recovery efficiency
 RH : air relative humidity (%)
 RSR : ratio of the heat recovered by convection at the soil surface to the total heat recovered based on sensible heat
 SAS : Statistical Analysis System
 SE : heat storage efficiency
 SF : solar contribution to the greenhouse heat load
 SPT : static pressure in the exchanger (0.78 kPa)
 SS : soil surface area (64.4 m²)
 T : temperature (°C)
 TASD : temperature differential between the soil and the air inside the greenhouse (°C)
 TB : time interval between data reports (s)
 TC : circulator operating time (s)
 TCC : cumulated circulator operating time (s)
 TCE : air temperature at the heater central outlet (°C)
 TCSD : temperature differential between the air at exchanger inlet and the soil (°C)

TCT : thermal curtain operating time (s)
 TCTC : cumulated thermal curtain operating time (s)
 TEMP. : temperature ($^{\circ}\text{C}$)
 TF : heater operating time (s)
 TFC : cumulated heater operating time (s)
 TFD : temperature differential between the heater inlet and outlet ($^{\circ}\text{C}$)
 TFM : average temperature between the heater inlet and outlet ($^{\circ}\text{C}$)
 TIM : inside daily average temperature ($^{\circ}\text{C}$)
 TIG : air inside temperature ($^{\circ}\text{C}$)
 TIH : air temperature at the heater inlet ($^{\circ}\text{C}$)
 TIOD : temperature differential between the inside and the outside of the greenhouse ($^{\circ}\text{C}$)
 TIOM : average temperature between the inside and the outside of the greenhouse ($^{\circ}\text{C}$)
 TL : lighting system operating time (s)
 TLC : cumulated lighting system operating time (s)
 TNO : temperature at the heater north side outlet ($^{\circ}\text{C}$)
 TOC : average temperature at the heater outlet ($^{\circ}\text{C}$)
 TOG : outside air temperature ($^{\circ}\text{C}$)
 TOHC : corrected air average temperature at the heater outlet ($^{\circ}\text{C}$)
 TOM : air daily average temperature outside the greenhouse ($^{\circ}\text{C}$)
 TOS : soil temperature outside the greenhouse ($^{\circ}\text{C}$)
 TOSM : mean soil temperature outside the greenhouse ($^{\circ}\text{C}$)
 TP : ventilation air inlet operating time (s)
 TPC : ventilation air inlet cumulated operating time (s)
 TS_{if} : final temperature of the *i*th soil layer ($^{\circ}\text{C}$)

TS_{ii} : initial temperature of the i th soil layer ($^{\circ}C$)
 $TSIC$: initial warm soil temperature ($25^{\circ}C$)
 $TSIF$: initial cold soil temperature ($12^{\circ}C$)
 TSF : operating time saved on the heater (s)
 TSS : soil temperature inside the greenhouse ($^{\circ}C$)
 $TSS1$: temperature at the soil surface ($^{\circ}C$)
 $TSS2$: temperature of the soil top layer ($^{\circ}C$)
 $TSS3$: temperature of the soil bottom layer ($^{\circ}C$)
 $TSSG$: average soil temperature ($^{\circ}C$)
 $TSSGM$: daily average soil temperature ($^{\circ}C$)
 TSU : temperature at the heater south side outlet ($^{\circ}C$)
 TSV : operating time saved on ventilation (s)
 TTD : temperature differential between the air passing through and the exchanger wall ($^{\circ}C$)
 TV : operating time of the ventilators (s)
 $TVGC$: cumulated operating time of the ventilators in high speed (s)
 $TVBC$: cumulated operating time of the ventilators in low speed (s)
 TVH : humidity controlled ventilator operating time (s)
 $TVHC$: humidity controlled ventilator cumulated operating time (s)
 UG : overall heat loss coefficient ($kW/m^2 \cdot ^{\circ}C$)
 UGM : mean overall heat loss coefficient ($kW/m^2 \cdot ^{\circ}C$)
 UGO : overall heat loss coefficient without wind ($kW/m^2 \cdot ^{\circ}C$)
 UGC : overall heat loss coefficient with the thermal curtain ($kW/m^2 \cdot ^{\circ}C$)
 UGH : overall heat loss coefficient with hoods on ventilation outlets ($kW/m^2 \cdot ^{\circ}C$)

UGCH : overall heat loss coefficient with thermal curtain and hoods ($\text{kW/m}^2 \cdot ^\circ\text{C}$)
 USM : greenhouse perimeter heat loss coefficient ($0.00112 \text{ kW/m} \cdot ^\circ\text{C}$, Lawand et al., 1985)
 V : wind speed (km/h)
 v : air velocity over the soil surface inside the greenhouse (0.045 m/s)
 VAH : air velocity at the heater inlet (4.96 m/s)
 VBR : burner supply voltage (120 V)
 VC : blower supply voltage (240 V)
 VG : greenhouse volume (242.7 m^3)
 VM : average wind velocity during a test (km/h)
 VS_i : soil layer volume (m^3)
 ($\text{VS}_0 = 9.88$ $\text{VS}_1 = 19.76$, $\text{VS}_2 = 42.15$)
 VV : ventilators supply voltage (122 V)
 WS_i : soil water content (kg of water/kg of soil)
 ($\text{WS}_1 = 0.2371$, $\text{WS}_2 = 0.0879$)
 x : soil fraction exposed to solar radiation (0.5)

Subscripts:

0	: at 0°C or at the soil surface	s	: dry air
1	: soil top layer	v	: vapor
2	: soil bottom layer	W	: wall
f	: final		
FL	: fluid		
i	: ith layer or initial		
I	: inlet		
n	: number of soil layers (2)		
O	: outlet		

I INTRODUCTION

1.1 Problem Description

The high cost of energy has forced greenhouse owners and researchers to look for more efficient means of production. During the seventies solar energy was seen as a possible alternative for reducing fossil fuel consumption (Short et al., 1976; Maghsood, 1976; Lawand et al., 1974; Ingratta, 1978; Agriculture Canada, 1979; Ingratta, 1979; and others).

Since conventional greenhouses are not efficient in collecting and storing solar energy, attempts were made to optimize the structure for collection and to develop techniques for the storage of the excess solar heat entering a greenhouse (Lawand et al., 1974; Candura et Gusman, 1977; Wilson et al., 1977; Maes, 1978; Baird and Water, 1979). Excessive heat input situation due to the greenhouse effect, are often encountered in a greenhouse during daytime. Ventilation has to be used to evacuate the excess heat and maintain adequate temperatures, while at night, heat has to be supplied to the greenhouse. The storage of excess heat might represent a short term solution for the greenhouse industry's high heating costs, while the development of energy efficient structures might be seen as a long term goal.

Water, crushed stones and phase change materials have been proposed to store solar energy and have been used in experimental setup with limited success (Roberts and Mears, 1977; Staley et al., 1982; Audet and Paris, 1985), mainly due to the high capital cost involved for the storage ins-

tallation, resulting into long payback periods. Furthermore, water, crushed stones and phase change materials have to be contained and imported into the greenhouse. However, soil a readily available material in greenhouses, is now considered as a possible heat storage media. The thermal properties are interesting and the costs involved are relatively low. The concept has been and is being evaluated in greenhouses located in mild climates (Portales et al., 1982; Staley et al., 1983; and Mori, 1978) and so far the results seem to be promising. Furthermore, the storage of heat in the soil of a greenhouse might have a beneficial effect on crops (Gosselin and Trudel, 1983).

The present study investigates the potential of a wet soil heat exchanger storage system for a commercial type greenhouse, in a cold climatic region.

1.2 Objectives

The objectives of the present study are:

- a) To determine the efficiency of a soil heat exchanger storage system for different operating conditions.
- b) To establish the contribution of solar energy, artificial lights and auxiliary heater to the total heat load of a commercial type greenhouse equipped with a wet soil heat exchanger-storage system.
- c) To observe the impact of the system on tomato crops.
- d) To estimate the payback period of the system for different situations.

1.3 Scope of the Study

The results of this study apply to a Nordic type greenhouse manufactured by Les Industries Harnois. The greenhouse is located in La Pocatière, Québec, Canada, at 47°21' north latitude, 70°02' west longitude and 30.5 m of altitude.

From 1951 to 1980 La Pocatière has received an average of 1952 hours of sunshine annually (Services de l'environnement atmosphérique, 1982). Corresponding to approximately the same period, the average annual number of heating degree-days below 18°C, were 5036 (Services de l'environnement atmosphérique, 1982). The annual mean daily global solar radiation is approximately 12.5 MJ/m² (Phillips, D.W. and D. Aston, 1980).

Care should therefore be taken in applying the results to different greenhouse structures, locations and climates.

II REVIEW OF LITERATURE

2.1 Review

The use of soil as a thermal buffer to cool or warm the air ventilated from agricultural buildings has been investigated over many years.

At the beginning, the attention was focussed on animal housing. Scott, et al. (1965) have studied the thermal and economic potential of a soil heat exchanger, made of buried pipes. These authors mentioned that such a system was first built in 1875; other systems were built thereafter, but the efficiency and the thermal behavior of these heat exchangers were not known or clearly understood. Their study permitted to establish and quantify the heat transfer mechanisms that takes place in the system.

Over the years, the technique has evolved and the Midwest Plan Service (1984) now offers a design guide for soil heat exchanger for energy conservation related to the ventilation of swine housing.

Recently, Puri (1986) has developed a mathematical model to simulate the dynamic response of soil in terms of heat and mass (moisture) transfer, in relation to a soil heat exchanger system. His study has shown that the soil moisture content is not significantly affected by the system; on the other hand, most of the heat transfer takes place over a pipe length equivalent to 50 diameters, and ra-

dial heat transfer takes place over a distance equivalent to 4 diameters.

From a general point of view, while Eckert (1976) has shown the potential of using soil as a seasonal heat storage material, Givoni (1977) has outlined the pro's and con's of doing so. Shelton (1975) has pointed out that only a small fraction of heat is lost by conduction from non insulated storage system buried in soil, this seems to favor the use of soil as a storage material. However, water infiltration into the storage due to a high water table can have a disastrous effect on the performance of such a storage system (Paris and Jacqmain, 1983).

The thermal behavior of a seasonal heat storage system made of vertically buried pipes in soil was studied by Mustacchi et al. (1981). They found that the heat front moved over two meters after six months of storage. However, conclusions drawn on the thermal behavior of seasonal storage system do not necessarily apply to short term storage system.

Sinha et al. (1981) investigated the use of different diameters and types of pipe such as plastic and galvanized steel for soil heat exchanger. It appears that the pipe material has little effect on the heat transfer. This conclusion is very important since pipe material affects directly the cost of the system. The heat exchange was more efficient in a 30 cm diameter pipe than in a 46 cm one, most likely due to the greater turbulence encountered for the same air

flowrate. The authors pointed out that heat transfer takes place over a maximum pipe length of 15 m and that pipe spacing should be equivalent to four or five pipe diameters.

Eckoff and Okos (1980) have studied different types of heat storage systems based on crushed stone, phase change material and soil. They concluded that the soil heat storage system is the less efficient, however, it is the most cost effective. Furthermore, it seems that a soil heat storage system is more efficient at low air velocity (3 m/s).

Walker and Buxton (1977) have conducted a theoretical study on the potential use of a soil heat exchanger system for greenhouse application. They concluded that such a system has no potential anywhere in the U.S.A., but the system under investigation has only one 30 m long pipe, buried outside the greenhouse perimeter, having its air intake located outside the greenhouse; therefore, forced convection through that pipe was the only means of exchanging heat with the greenhouse and conduction heat lost by the soil could not be recovered by the greenhouse.

Dale and Hammer (1983) have designed, built and tested a solar greenhouse equipped with a soil thermal storage system within the foundations of the greenhouse, and an external solar collector. The study's most interesting conclusion is that the external solar collector is unnecessary since the greenhouse structure plays that role without additional cost.

Richard et al. (1981) have simulated the thermal behavior of a greenhouse equipped with a soil heat exchanger system under the climatic conditions of Nova Scotia. The various test runs indicated a 25 % energy conservation and a payback period of approximately one year, but the model used was simple, and the various assumptions used are not always encountered in a greenhouse.

Sibley and Raghavan (1984) have developed an empirical equation to determine forced convection heat transfer coefficient within non-perforated, corrugated plastic pipes, for various diameters. The following equation:

$$Nu = 0.230 Re^{0.56} Pr^{0.3} \quad (2.1)$$

is said to be valid for the conditions listed below:

Reynolds number (Re) > 2300

Prandtl number (Pr) : 0.71

diameter : 102 mm

$2.2 \leq$ air velocity (m/s) ≤ 9.6

$2.0 \leq$ wall temperature ($^{\circ}\text{C}$) ≤ 15.0

$7.9 \leq$ air inlet temperature ($^{\circ}\text{C}$) ≤ 23.1

This relationship seems to be more adequate for design purposes of heat exchanger systems than the ones based on smooth pipes, like for example, the one proposed by Colburn-Reynolds reported by the authors. However, the convective heat transfer coefficient derived from the equation cannot predict the heat transfer when condensation occurs in the pipes; this phenomena is likely to happen with air coming

from a greenhouse. Nevertheless, the proposed relationship might be useful as a design tool.

Milburn and Aldrich (1979) have used the coefficient of performance (COP) concept related to heat pumps, in order to evaluate the performance of a heat storage system installed in a greenhouse and using crushed stones. They indicated that the system performance is more dependent on internal air temperature than on intercepted solar radiation. It seems interesting to use the COP concept in order to compare different thermal storage systems or different configurations or operating conditions of a soil heat exchanger-storage system.

By the end of 1981, more than 200 Japanese greenhouse owners had equipped their greenhouses with soil heat exchanger-storage systems (Takakura et al., 1982). Yamamoto (1973) designed a soil heat exchanger-storage system made of three rows of earthen pipes buried at 64, 100 and 136 cm deep respectively. The spacing between the pipes was 36 cm. The system was capable of maintaining an average temperature of 10°C at night in the greenhouse without auxiliary heat source.

Sasaki and Itagi (1979) have conducted a similar experiment using one and two rows of pipes buried in the soil. The amount of thermal energy provided by the soil heat exchanger-storage system was found to be equivalent to eight times the amount of electrical energy used for its own ope-

ration. The results obtained have led the authors to conclude that the system was efficient and cost effective.

Sasaki et al. (1980) have obtained energy conservation factors ranging from 40 to 60 % in a greenhouse equipped with thermal curtain. The same authors (1981) also reported that up to 60 % of the heat exchange was due to latent heat. Therefore, the mass transfer due to humidification and dehumidification, should not be neglected in a thermal balance calculation.

Takakura and Yamakawa (1981) and Takakura et al. (1982) have developed a simplified heat exchange model, in order to simulate the thermal response of soil heat exchanger-storage systems under different operating conditions. Approximately 20 to 30 % of the heat stored during the day is said to be recovered at the soil surface. Furthermore, the exchanger pipe thickness and the pipe material do not seem to have any significant effect on the exchanger performance; this confirms the conclusions of Sinha et al. (1981). According to this study, the optimum pipe diameter and horizontal/vertical spacing should be of 11 and 40 cm respectively.

The use of soil as a thermal buffer within greenhouses, was also studied in U.S.S.R. (Rabbimov et al., 1971; Vardiasvili and Kim, 1980; and others). Umarov et al. (1981) for example, obtained energy conservation factors ranging from 35 to 55 % depending on sunshine conditions. Since the technique is used in such a northern country effectively, it might be suitable for the Canadian greenhouse industry.

In Prince Edward Island, Caffell and Mackay (1981) have designed a seasonal heat storage system for a greenhouse, using 1080 m³ of wet soil. No experimental result is reported; but simulations indicated that such a system could not store all the excess heat generated at peak times, unless an intermediate level of storage is provided.

Although the intermediate storage concept might be interesting from a thermal efficiency point of view, this concept might not be cost effective and adds to the complexity of the system.

Inspired by the Japanese research works, Monk et al. (1983) and Staley et al. (1983) have built in southern B.C., a soil heat exchanger-storage system in a glass covered greenhouse equipped with a thermal curtain. Preliminary results show a 49 % energy conservation between April 23 and June 3. The authors noted that it was not necessary to reverse the air flow direction for heat recovery, since there was little horizontal temperature stratification in the thermal mass. For that same project, French (1987) estimated an overall energy conservation factor ranging from 20 to 25 %, which seems to be satisfactory for the prevailing conditions.

Further, an economic analysis done by Arcus Consulting Ltd. (1985) for Agriculture Canada, comparing different energy conservation techniques including soil heat storage, crushed stones heat storage, internal solar collector and thermal curtains, indicated that soil heat storage seems to

be the most cost effective technique having an internal rate of return of 55 % based on an oil heated greenhouse.

Lawand et al. (1983) have studied a similar system in order to produce a design guide. The report includes an extensive literature review covering the theoretical and practical aspects related to soil heat exchanger-storage systems. Tests performed on a small scale experimental setup seem to indicate that the heat transfer rates were similar for either sand or loam. A 40 cm spacing between the pipes is said to be optimum; this confirms the conclusions of studies cited previously.

In their design guide, Lawand et al. (1985) recommend pipe spacing ranging from 40 to 60 cm, and a maximum pipe length of 15 m. An optimum air velocity of 2.0 m/s is suggested by the authors; however, this value has not been verified experimentally. The system is said to have a dehumidifying effect on the air circulated in the exchanger; condensation rates between 0.256 and 0.383 l/h were obtained under the prevailing conditions of the experiment. The authors mentioned that by draining the condensate into the surrounding soil, heat transfer rates could be increased by 40 to 78 %. However, the use of perforated pipes could be detrimental to the system performance by draining excess irrigation water. Further, the use of perforated pipes could result in soil drying, which can be detrimental to soil grown crops and to heat transfer and storage. The authors acknowledge that further work should be carried out before any

recommendations can be provided with regard to condensate disposal. It should also be pointed out that the design guide has not been tested for a commercial type greenhouse application.

2.2 Summary

Much of the review cited previously lead to anticipate that a soil heat exchanger-storage system installed within the perimeter of a greenhouse, can be efficient and cost effective. Studies indicate that the maximum length of such exchanger should range from 10 to 15 m, and that pipe spacing (10 cm in diameter) should be of 40 cm. As far as optimum air velocity is concerned, the studies seem to favor values under 5 m/s, but no precise conclusions are available. The same holds true for air dehumidification. The capacity of such a system to dehumidify air is acknowledged; however, no precise and problem free method for the removal of condensate is recommended. Although a design guide for soil heat exchanger-storage system was produced, no result was obtained from a typical commercial type greenhouse in Canada or in the U.S.A. However preliminary results and simulation studies acknowledge the energy conservation potential of the system.

III THEORETICAL CONSIDERATIONS

The evaluation of the heat exchanger-storage performance and of its impact on the greenhouse thermal energy consumption, was based on energy balance and standard heat transfer equations. In order to simplify the data analysis, the following assumptions were made:

- 1 - The soil physical properties remain constant for the duration of the study,
- 2 - Each soil layer is homogeneous,
- 3 - Only water content can affect the soil thermal properties for the temperature interval involved,
- 4 - Heat distribution in the soil is symmetrical along a perpendicular to the greenhouse longitudinal axis,
- 5 - The temperature profile is identical in each pipe.

From an energy balance point of view, the contribution of solar energy to the heat load of a greenhouse, can be calculated by subtracting the amount of auxiliary heat used from the overall greenhouse heat requirements; therefore, a solar energy contribution ratio can be defined as:

$$SF = (Q_{THL} - Q_{AUX}) / Q_{THL} \quad (3.1)$$

The total heat loss of the greenhouse should include the aerial, soil and ventilation portions, as indicated by:

$$Q_{THL} = Q_{GHL} + Q_{SHL} + Q_{VHL} \quad (3.2)$$

To compensate for the heat losses, heat is supplied by one or more of the following sources:

- i) solar energy;
- ii) auxiliary heat by a heater;
- iii) irrigation water;
- iv) and artificial lights.

Therefore,

$$Q_{AUX} = Q_F + Q_{IR} + Q_{LU} \quad (3.3)$$

Among these heat sources, the heat provided by the irrigation water is much smaller compared to the heat provided by the heating (Portugais and Paris, 1983) and artificial lighting systems, and therefore can be neglected.

$$Q_{AUX} = Q_F + Q_{LU} \quad (3.4)$$

Similarly, the heater contribution can be defined by:

$$FF = Q_F / Q_{THL} \quad (3.5)$$

The thermal energy output of a heater can be expressed in terms of its power, therefore:

$$Q_F = P_F \cdot T_F \quad (3.6)$$

The average effective power of the heater can be estimated by:

$$PF = DAF \cdot VAH \cdot AI \cdot CAF \cdot (TOC - T_{IH}) \quad (3.7)$$

re-arranging:

$$QF = DAF \cdot VAH \cdot AI \cdot CAF \cdot (TOC - T_{IH}) \cdot TF \quad (3.8)$$

For artificial lighting, the contribution to the heat load is defined by:

$$LF = Q_{LU} / Q_{THL} \quad (3.9)$$

The heat released by the lamps can be evaluated by:

$$QL = PL \cdot TL \quad (3.10)$$

However, when the ventilation is in operation, part of thermal energy released by the lamps is in excess, this excess heat can be estimated by:

$$QLE = PL \cdot TP \quad (3.11)$$

provided that both the lamps and ventilation are operating simultaneously during the monitoring period. Therefore, the useful energy provided by the lamps is:

$$Q_{LU} = Q_L - Q_{LE} \quad (3.12)$$

At night, for steady state conditions, the heat losses of the greenhouse will be equal to the heat provided by the heating system, provided that the average top soil temperature is equal to the air temperature inside the greenhouse, in order to minimize the convective and radiative heat transfer between the soil and the air.

$$Q_{GHL} = U_G \cdot A_G \cdot (T_{IG} - T_{OG}) \cdot T_B = P_F \cdot T_F \quad (3.13)$$

Re-arranging the terms, the greenhouse overall heat loss coefficient for the aerial section can be obtained:

$$U_G = \frac{D_{AF} \cdot V_{AH} \cdot A_I \cdot C_{AF} \cdot (T_{OC} - T_{IH}) \cdot T_F}{A_G \cdot (T_{IG} - T_{OG}) \cdot T_B} \quad (3.14)$$

if $T_F = T_B$, then:

$$U_G = \frac{D_{AF} \cdot V_{AH} \cdot A_I \cdot C_{AF} \cdot (T_{OC} - T_{IH})}{A_G \cdot (T_{IG} - T_{OG})} \quad (3.15)$$

It has been shown that the heat loss coefficient is directly proportional to the wind velocity (Sheard, 1978); therefore, the following empirical relationship can be used to evaluate the average overall heat loss coefficient for different wind velocities:

$$UG = UG_0 + c \cdot V \quad (3.16)$$

The heat losses from the storage can be evaluated using a relationship outlined in ASHRAE (1968):

$$Q_{SHL} = USM \cdot PE \cdot (T_{SSG} - T_{OG}) \cdot TB \quad (3.17)$$

If the soil within the greenhouse is divided into n elemental volumes the amount of heat stored in the soil can be expressed by:

$$Q_{SS} = \sum_{i=1}^n VS_i \cdot DSS_i \cdot CS_i \cdot (TS_{if} - TS_{ii}) \quad (3.18)$$

where the specific heat of each soil elemental volume is estimated from Munn (1966):

$$CS_i = CW \cdot WS_i + CDS_i \cdot (1 - WS_i) \quad (3.19)$$

Part of the heat stored is gained through a buried pipe heat exchanger, in which air is circulated; the amount of heat transfer in the form of sensible and latent heat is:

$$Q_{ES} = DAT \cdot FR \cdot (HTI - HTO) \cdot TC \quad (3.20)$$

Some heat is also gained through solar radiation absorption by the soil surface, given by:

$$QRS = A \cdot x \cdot SS \cdot ISH \quad (3.21)$$

where the amount of solar radiation received is based on the operation of the thermal curtain. The thermal curtain also acts as a shading device.

$$ISH = ISC \cdot (TB - TCT) / 100 \quad (3.22)$$

Part of the heat stored in the soil, also comes from the convective heat transfer between the air and the soil surface, as defined by:

$$QGF = HCE \cdot AF \cdot (TIG - TSS) \cdot TB \quad (3.23)$$

The convective heat transfer coefficient at the soil surface is estimated using an empirical relationship presented by Cormany and Nicolas (1985):

$$HCE_s = (5.0 + 3.5 \cdot v) / 1000 \quad (3.24)$$

A thermal heat storage efficiency can be defined by:

$$SE = \frac{QSS}{QES + QRS + QGF} \quad (3.25)$$

In terms of heat recovery from the storage, the heat reclaimed by the exchanger can be evaluated by:

$$QRT = DAT \cdot FR \cdot (HTO - HTI) \cdot TC \quad (3.26)$$

Part of the stored heat is also recovered by air at the soil surface:

$$QRF = HCE \cdot AF \cdot (TSS - TIG) \cdot TB \quad (3.27)$$

Assuming that the heat transferred from the soil to the air by radiation is negligible, a thermal heat recovery efficiency can be defined by:

$$RE = \frac{QRT + QRF}{QSS} \quad (3.28)$$

The sensible heat exchange within the pipes of the exchanger can be evaluated by the following relationships:

$$QEX = AT \cdot HCT \cdot (TF - TW) \cdot TB = M_{air} \cdot Cs \cdot (TI - TO) \quad (3.29)$$

The convective heat transfer coefficient can be obtained by re-arranging the terms:

$$HCT = \frac{M_{air} \cdot Cs \cdot (TI - TO)}{AT \cdot (TF - TW) \cdot TB} \quad (3.30)$$

Since sensible heat is recovered passively at the soil surface and actively from the exchanger, the ratio of pas-

sive heat recovery to the total heat recovered is expressed by:

$$RSR = Q_{RF} / (Q_{EX} + Q_{RF}) \quad (3.31)$$

Chapman (1976), defined the efficiency of a heat exchanger when the fluid is heated by:

$$ERE = (T_O - T_I) / (T_{SIC} - T_I) \quad (3.32)$$

when the fluid is cooled by the exchanger, the efficiency is expressed by:

$$ESE = (T_I - T_O) / (T_I - T_{SIF}) \quad (3.33)$$

The heat exchanger-storage system can be considered as a heat pump. A coefficient of performance (COP) adjusted for the amount of electrical energy not used by the ventilators or the heater when the heat exchanger is in action, can be defined in terms of sensible heat by:

$$COPS = Q_{ES} / (Q_C - Q_V) \quad (3.34)$$

for heat storage, and by:

$$COPR = Q_{RT} / (Q_C - Q_{BR}) \quad (3.35)$$

for heat recovery.

The electrical energy used by the heat exchanger blower is evaluated by:

$$Q_C = V_C \cdot I_C \cdot T_C \quad (3.36)$$

The electrical energy not used by the heater burner is estimated from:

$$Q_{BR} = V_{BR} \cdot I_{BR} \cdot T_{SF} \quad (3.37)$$

Since the heat recovered will not have to be supplied by the heater, the operating time saved on the heater can be evaluated from:

$$Q_{RT} + Q_{RF} = Q_F = P_F \cdot T_{SF} \quad (3.38)$$

re-arranging the terms:

$$T_{SF} = (Q_{RT} + Q_{RF}) / P_F \quad (3.39)$$

$$Q_{BR} = V_{BR} \cdot I_{BR} \cdot (Q_{RT} + Q_{RF}) / P_F \quad (3.40)$$

Similarly, the electrical energy not used by the ventilators is:

$$Q_V = V_V \cdot I_V \cdot T_{SV} \quad (3.41)$$

Regarding ventilation, the operation of the buried pipes heat exchanger storage system is the first ventilation stage, therefore the operating time saved on the ventilators is equal to the difference between the system blower and ventilators operating time.

$$TSV = TC - TV \quad (3.42)$$

by re-arranging:

$$QV = VV \cdot IV \cdot (TC - TV) \quad (3.43)$$

Since the air relative humidity inside the greenhouse can highly fluctuate, the physical properties of moist air will also fluctuate and can significantly differ from the properties of dry air. Moist air density can be computed from Rosenberg et al. (1983):

$$DA = MA \cdot (P - 0.378 \cdot EA) / (R \cdot T) \quad (3.44)$$

The vapor partial pressure is obtained from the definition of relative humidity:

$$RH = EA / ES \cdot 100 \quad (3.45)$$

re-arranging:

$$EA = RH \cdot ES / 100 \quad (3.46)$$

where the saturated vapor pressure is obtained from:

$$ES = 0.61078 \exp (17.269 \cdot T / (T + 237.30)) \quad (3.47)$$

The absolute humidity is defined by:

$$HU = 0.622 \cdot (RH \cdot ES / 100) / (1 - RH \cdot ES / 100) \quad (3.48)$$

The specific heat of moist air is estimated by:

$$Cs = Cp_s + Cp_v \cdot HU \quad (3.49)$$

and the enthalpy of moist air is obtained from:

$$HT = Cs \cdot (T - T_0) + HV \cdot LH \quad (3.50)$$

These theoretical considerations were translated into two computer programs, one to characterize the greenhouse heat loss with respect to wind conditions, and one to generate the performance indicators for various operating conditions. These topics are covered in the next section.

MATERIALS AND METHODS

4.1 Experimental Setup

A heat exchanger-storage system made of 26 non perforated, corrugated plastic drainage pipes, 102 mm in diameter, buried in the soil of a NORDIC type greenhouse, is shown in Figure 4.1. The first row consisting of 13 pipes, was buried at 450 mm and the second row at 300 mm below. The pipes run parallel to the horizontal east-west axis of the greenhouse, and were spaced at 450 mm apart. The characteristics of the greenhouse and pipes are listed in Tables 4.1 and 4.2 respectively.

The 0.75 kW Delhi blower shown in Figure 4.2, is used to circulate the air from the greenhouse into the pipes at $0.91 \text{ m}^3/\text{s}$. The blower was placed over a $6230 \times 750 \times 1240 \text{ mm}$ plywood plenum on which the pipes were connected. The blower has its air intake over the thermal curtain near the ridge for daytime operation, and under the curtain for nighttime operation. The air is discharged into the greenhouse from the 26 pipe outlets that emerge from the soil.

The greenhouse is equipped with a thermal curtain (Table 4.1) that can be used as a light shading device during summer. Two 2-speed ventilators, 610 mm in diameter, having a maximum capacity of $2.12 \text{ m}^3/\text{s}$, were used to control the temperature when the soil heat exchanger storage system can no longer remove enough heat. An auxiliary ventilator, 335 mm having a capacity of $0.63 \text{ m}^3/\text{s}$, was used to control the air relative humidity. A standard oil heater 36 kW in

Table 4.1 Experimental greenhouse¹ characteristics

Parts	Dimensions	Covering	Insulation	Surface Area (m ²)
South Wall	12.1 x 1.83	Polycarbonate ² 6 mm	Thermal ³ Curtain	22.1
South Roof	12.1 x 3.53	Polycarbonate 6 mm	Thermal Curtain	42.7
North Roof	12.1 x 3.53	Polycarbonate 6 mm	Thermal Curtain	42.7
North Wall	12.1 x 1.83	Corrugated sheet metal	Polyurethane 76 mm	22.1
East Wall	- (top section)	Polycarbonate 6 mm	-	5.7
	0.95 x 6.50 (middle section)	Polypropylene ² 5 mm	-	6.2
	1.2 x 6.50 (bottom section)	Corrugated sheet metal	Polyurethane 76 mm	6.6
West Wall	-	Corrugated sheet metal	Polyurethane 38 mm	18.5
Floor	12.1 x 6.50	-	-	78.7
Foundation	1.4 x 36.3	Reinforced concrete	Polystyrene 76 mm	50.8

1 "NORDIC" model constructed by Harnois Industries, oriented east-west.

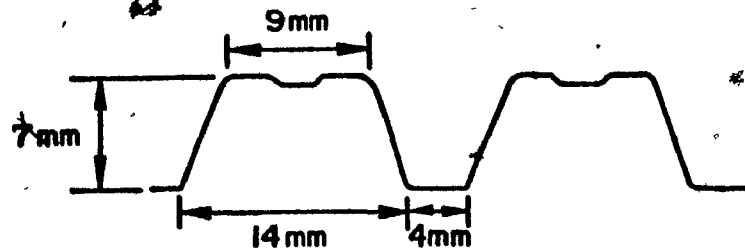
2 Hollow profile double skin plastic.

3 Single white diffusing plastic film.

Table 4.2 Heat exchanger pipe¹ characteristics

Thickness	:	1.0 mm
Inside diameter	:	102.0 mm
Outside diameter	:	110.0 mm
Mean inside diameter	:	105.5 mm
Corrugation height	:	7.0 mm
Base corrugation length	:	14.0 mm
Top corrugation total length	:	11.0 mm
Inter-corrugation length	:	4.0 mm
Top corrugation straight length	:	9.0 mm
Side corrugation length (estimated)	:	7.4 mm
Unit surface area	:	0.531 m ² /m

¹ Non-perforated, corrugated drainage pipe



(From Lawand et al., 1985)

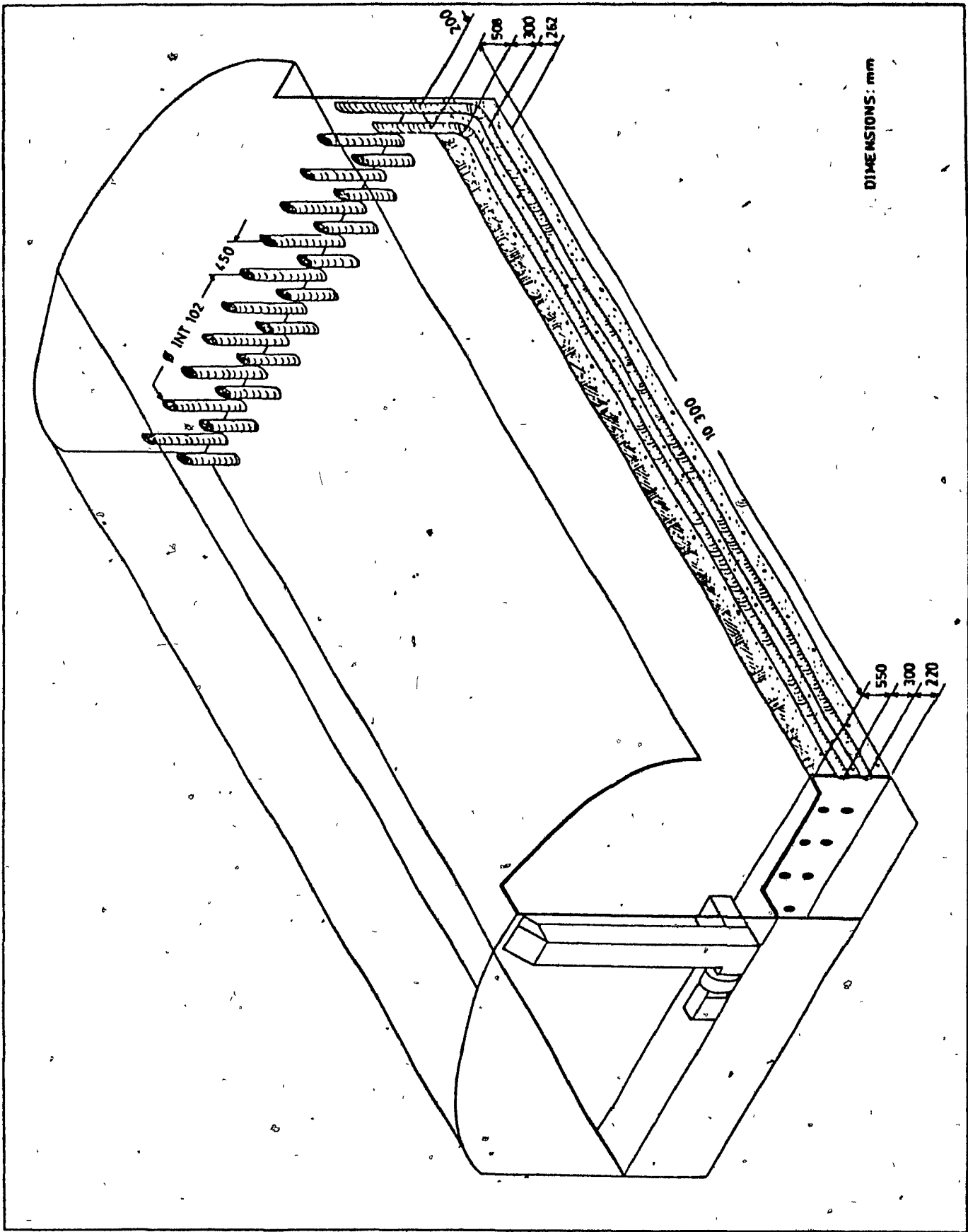
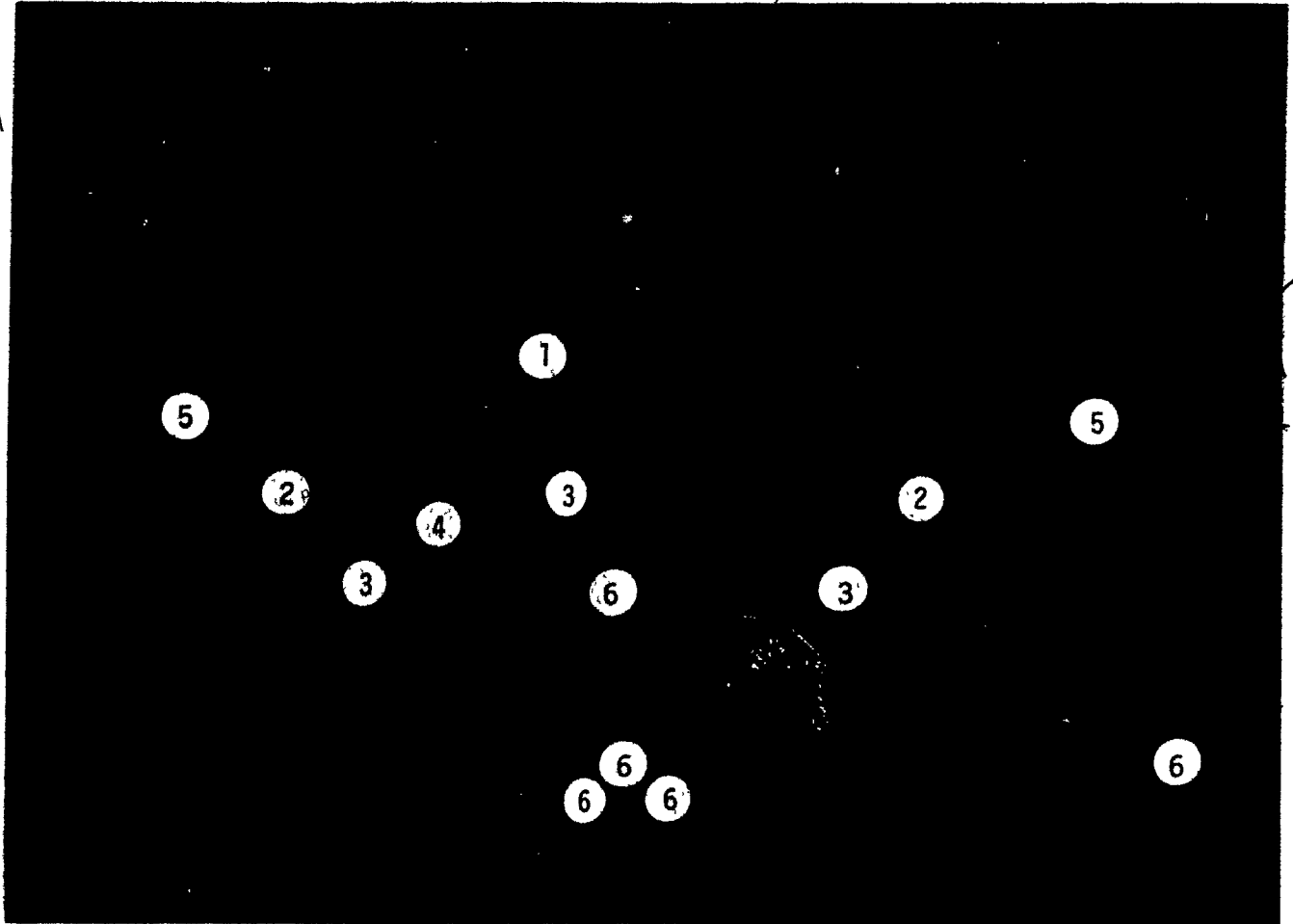


Figure 4.1 Greenhouse heat exchanger-storage system.



1: heat exchanger air inlet 2: ventilator 3: heater outlet
4: heater inlet 5: thermal curtain 6: support for sensors

Figure 4.2 Greenhouse main equipments.

capacity was used to heat the greenhouse and an adjacent service room. A drip irrigation system was used to irrigate and fertilize the crop, helping that way in maintaining a relatively constant humidity level in the soil. The system was controlled by three irrometers mounted in parallel; they activate a solenoid valve when the soil moisture level goes below a preset level. Supplemental lighting was provided in winter by four 400 W HPS lamps mounted under the thermal curtain in the greenhouse center along its longitudinal axis.

A microcomputer system was installed to control the greenhouse environment. The system shown in Figure 4.3, includes a Tandy microcomputer model # 26-3127 complete with Microsoft extended BASIC, 64 kilobytes of random access memory, a black and white video monitor, an audio cassette recorder for data storage, a Tandy dot matrix model # 26-1276 and a custom made data acquisition and control interface.

The system can report on the activity of 8 electrical equipments and perform data acquisition from 80 channels. The characteristics of the system are listed in Table 4.3. Hooked to the system are:

- one Lycor # LI-200SB pyranometer, mounted horizontally in the greenhouse center over the thermal curtain at 2.5 m above ground,
- one Trade Wind Instruments # C72131 anemometer, mounted outside at 4.2 m above ground,

Table 4.3 Data Acquisition and control system characteristics

analog digital converter resolution	:	15 bits
conversion time per channel	:	300 ms
temperature sensor channels	:	64
miscellaneous sensor channels	:	16
control channels	:	8
system watchdog	:	1
real time clock	:	date, day, hour, minute, second and millisecond
expansion	:	1 port 10 bits
channel scanning rate	:	0 to 34 s
report frequency	:	user defined
computation	:	differences air relative humidity from dry and wet bulb temperatures
report output on monitor, printer or cassette	:	date, day, hour, minute, second report number number of scanning maximum value per channel minimum value per channel mean value per channel standard deviation per channel time of operation for each electrical equipment

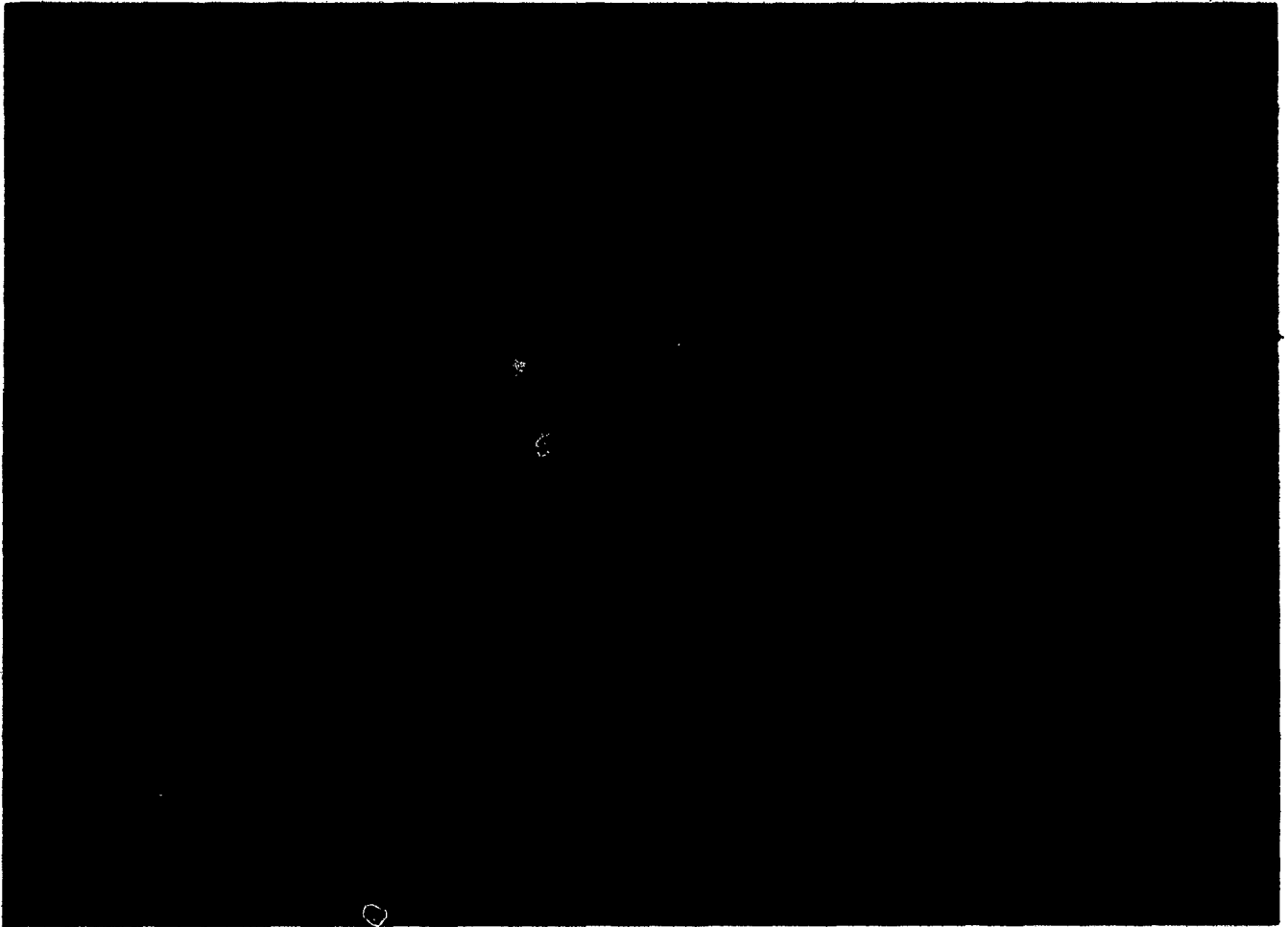


Figure 4.3 Microcomputer system installed in the experimental greenhouse.

- sixty Analog Device # AD-590, temperature sensors were installed as shown in Figures 4.4 and 4.5. A typical temperature sensor is shown in Figure 4.6.

The system was designed to use AD-590 converters as temperature sensors because of their linear response to temperature variation.

Not shown in Figures 4.4 and 4.5 are:

sensors T31 and T32H, measuring respectively the internal air dry and wet bulb temperatures; these sensors were installed in a ventilated box located in the greenhouse center, at 1.75 m above ground;

sensors T20, T47 and T48 measuring soil temperatures at one meter outside the foundations, were installed to correspond with depths identical to the ones shown in Figure 4.4 for T17, T18 and T19 respectively;

sensors T57 and T58 were located to measure air temperatures at the heater inlet and central outlet respectively;

sensors T28 and T29H were installed to obtain respectively the air dry and wet bulb temperatures at the exchanger inlet;

sensor T56 was used to measure the air temperature near the roof ridge in the western part of the greenhouse;

sensor T33 was located to measure the outside air temperature; this sensor was shielded from precipitation and sunshine and was located close to the anemometer.

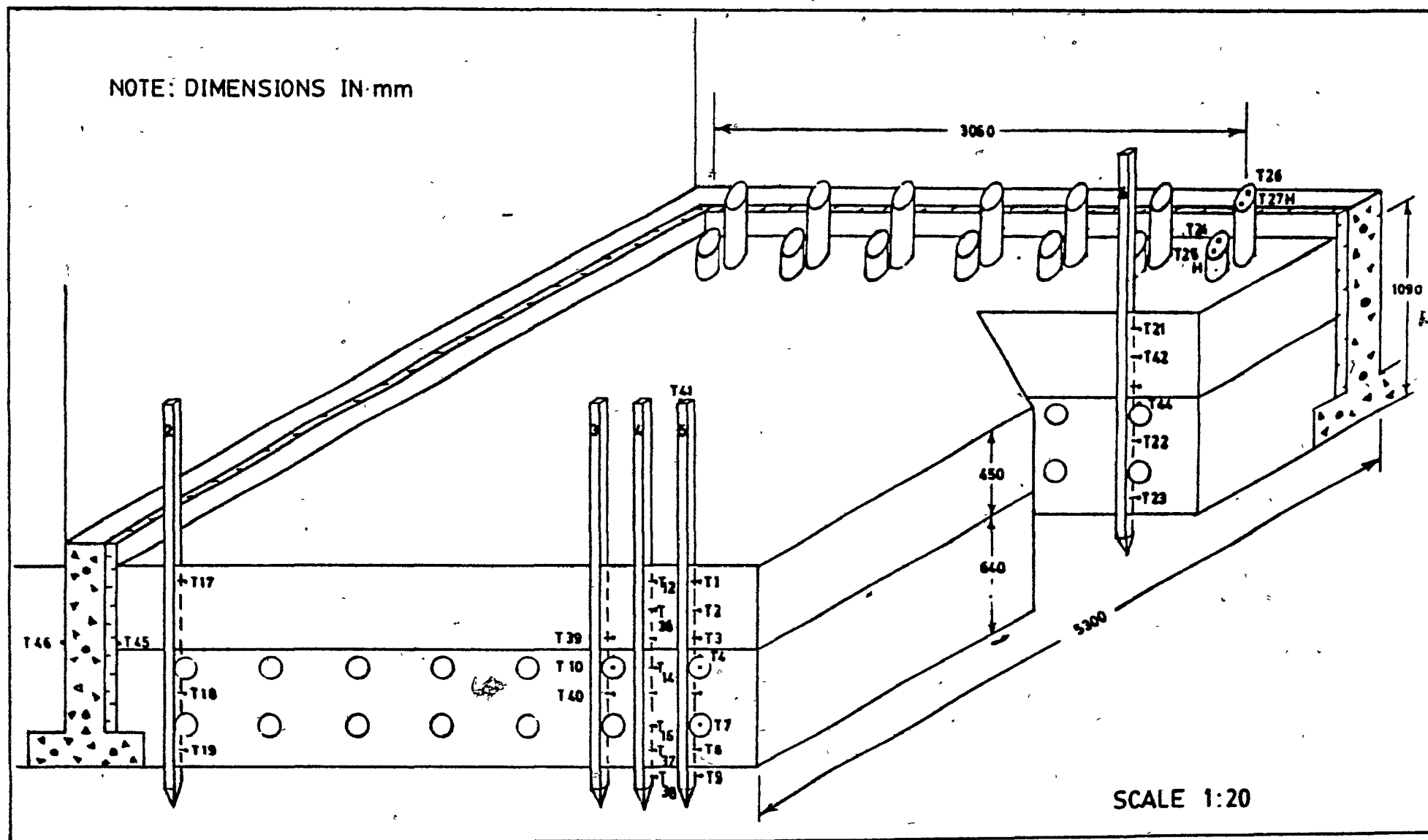


Figure 4.4 Temperature sensors installed in the northeast quadrant of the greenhouse.

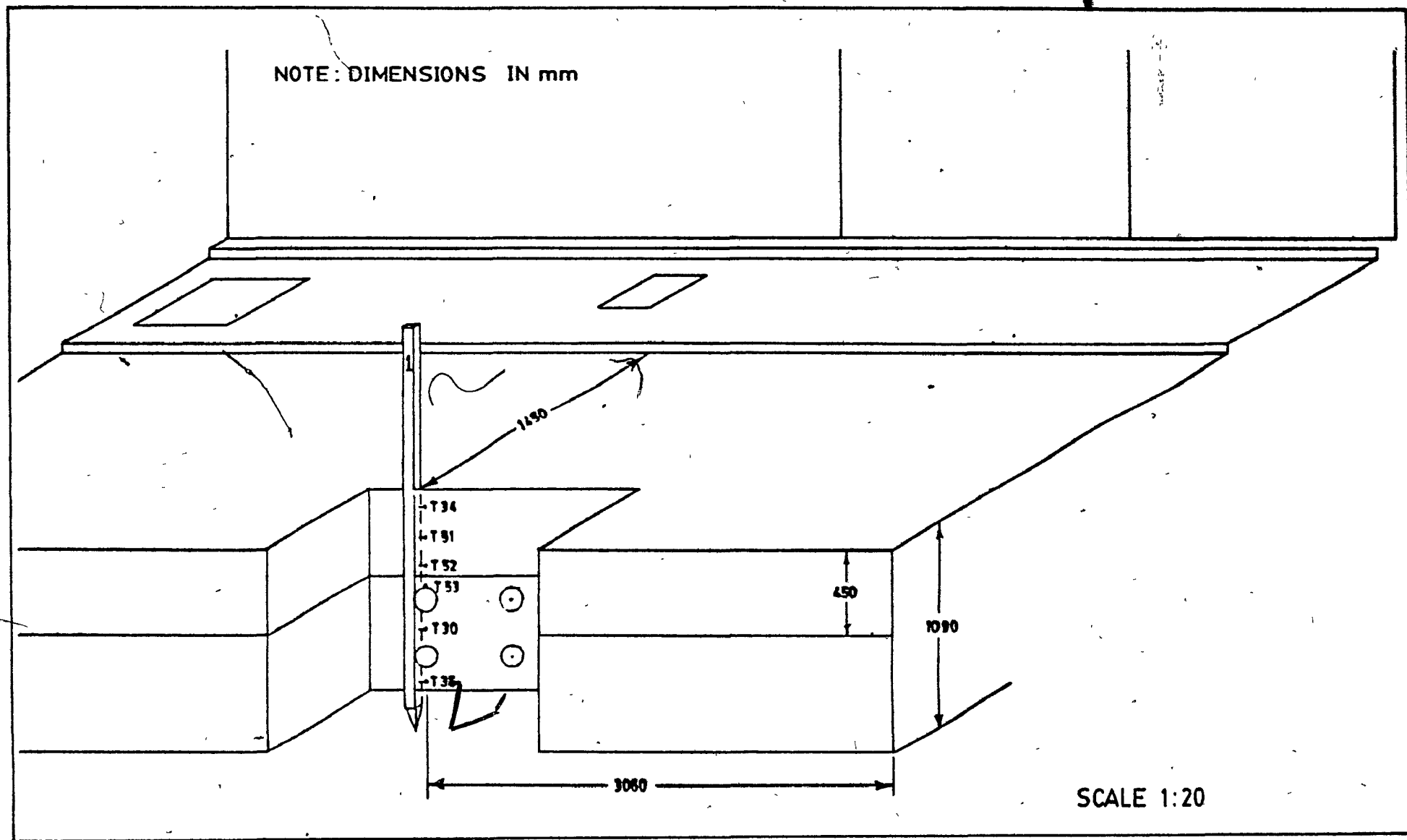


Figure 4.5 Temperature sensors installed in the western part of the greenhouse.

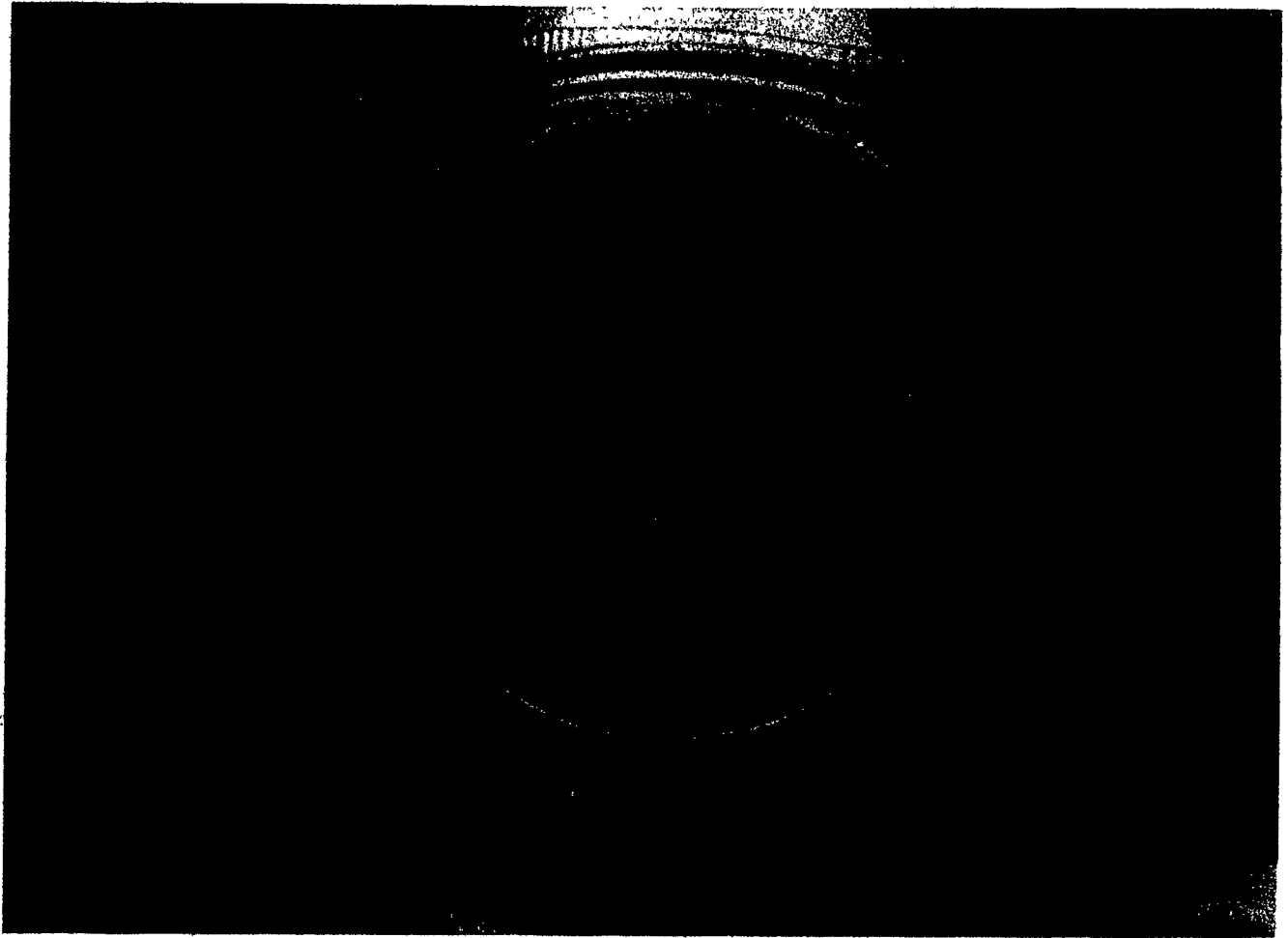


Figure 4.6 Analog Device AD-590 temperature sensor.

4.2 Procedure

4.2.1 Soil Physical Properties

The soil storage is made of two layers; the top layer consists of 450 mm of organic soil, whereas the bottom layer of 610 mm depth, is of mineral soil. Clay was encountered at approximately 230 mm below the bottom layer.

A soil texture analysis was done for the two layers. Soil water contents, wet and dry bulk densities were measured using a Stratagauge Troxler model # 3411B shown in Figure 4.7, and by taking core samples at 20 sites picked at random. For each site, samples were collected at depths of 150, 300 and 600 mm.

For each layer, thermal capacity tests were conducted on composite samples, using standard methods outlined by Taylor and Jackson (1965). Soil water content was monitored at several times during the test period.

4.2.2 Air velocity

The air flow rates were adjusted for each buried pipes using a velometer (Alnor model Jr). Restrictions were adjusted in order to equalize the flow of air in each tube. A Pitot tube (Figure 4.8) was used to measure air velocities at five points in a cross-section of each pipe, as indicated in bulletin no. H-11 by Dwyer Instruments Inc. (1984). The same procedure was used to adjust and measure air velocities at



Figure 4.7 Stratagauge Troxler model no. 3411B.

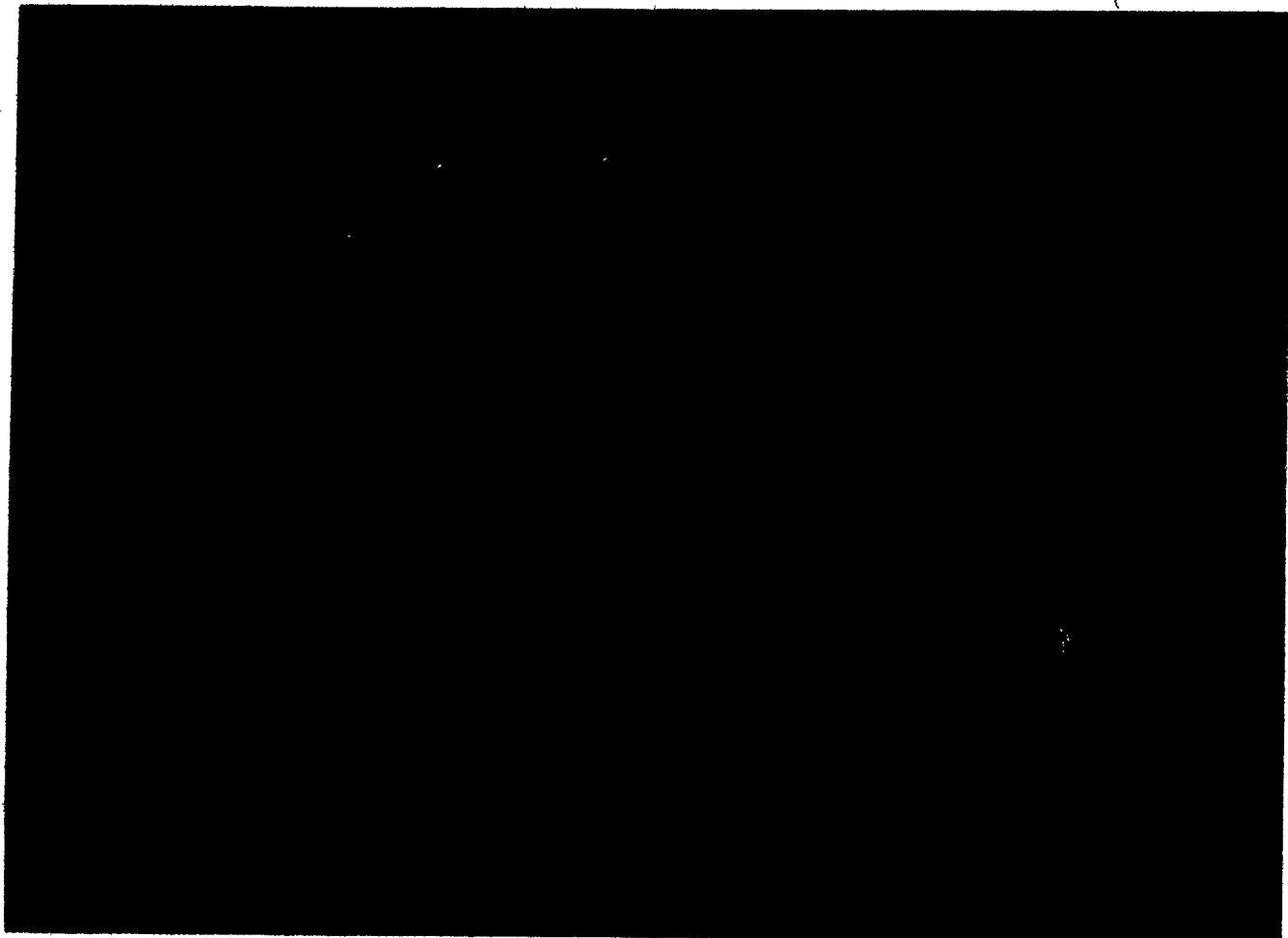


Figure 4.8 Pitot tube apparatus.

the heater outlet; twenty points in the outlet cross-section were measured.

4.2.3 Overall Heat Exchange Coefficients

4.2.3.1 Heat Loss Coefficients (UG)

Test runs were periodically conducted at night in order to evaluate the overall heat loss coefficients for the aerial portion of the greenhouse for different wind conditions.

Tests were also performed with and without hoods on ventilator outlets and thermal curtain. Inside and outside air temperatures, inside air relative humidity, air temperatures at the heater inlet and outlet and wind speed were measured over a period of a few hours, when steady state conditions were reached.

In order to minimize the heat transfer between the top soil and the surrounding air inside the greenhouse, the air temperature was maintained at the average temperature of the top soil. Figure 4.9 illustrates the computation process.

4.2.3.2 Convective Heat Transfer Coefficient

Over the test period, the air relative humidity and temperature measured at the exchanger inlet and outlets, and the air and wall temperatures in a pipe were continuously monitored. The estimation of the coefficient was based on sensible heat exchange, as indicated by equation 3.30.

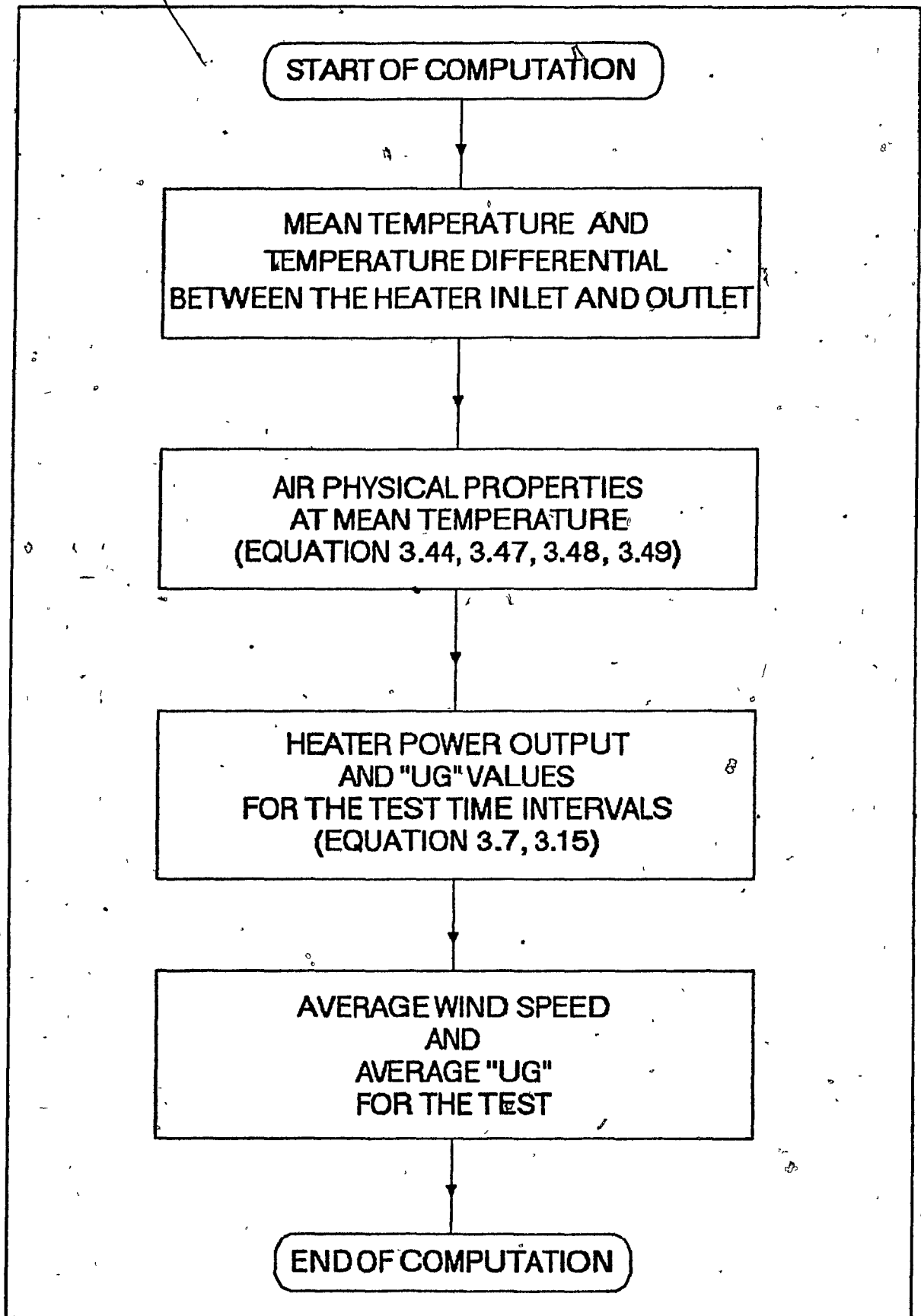


Figure 4.9 Flowchart for the computation of the overall heat loss coefficient.

4.2.4 Electrical Energy Used

The RMS voltages and currents were measured for each piece of equipment over a period of time, in order to evaluate the average power requirements.

The operating time was obtained from the computer reports on an hourly basis. The electrical energy used was calculated from these data, using equations 3.10, 3.11, 3.36, 3.40 and 3.43.

4.2.5 Daily performance

On an hourly basis, the following measurements were made to assess the thermal performance of the greenhouse and of the heat exchanger storage system:

- inside and outside soil temperatures,
- inside and outside air temperatures,
- temperatures at the inlet and outlet of the heater,
- inside air relative humidity,
- dry and wet bulb temperatures at the inlet and outlet of two pipes,
- inside pipe and pipe wall temperatures for one pipe,
- solar radiation inside the greenhouse measured on a horizontal plane,
- wind velocity,
- operating time for each piece of equipment.

The equipment inside the greenhouse were operated according to the schedule of events shown in Table 4.4, and the flowchart presented in Figure 4.10 illustrates the

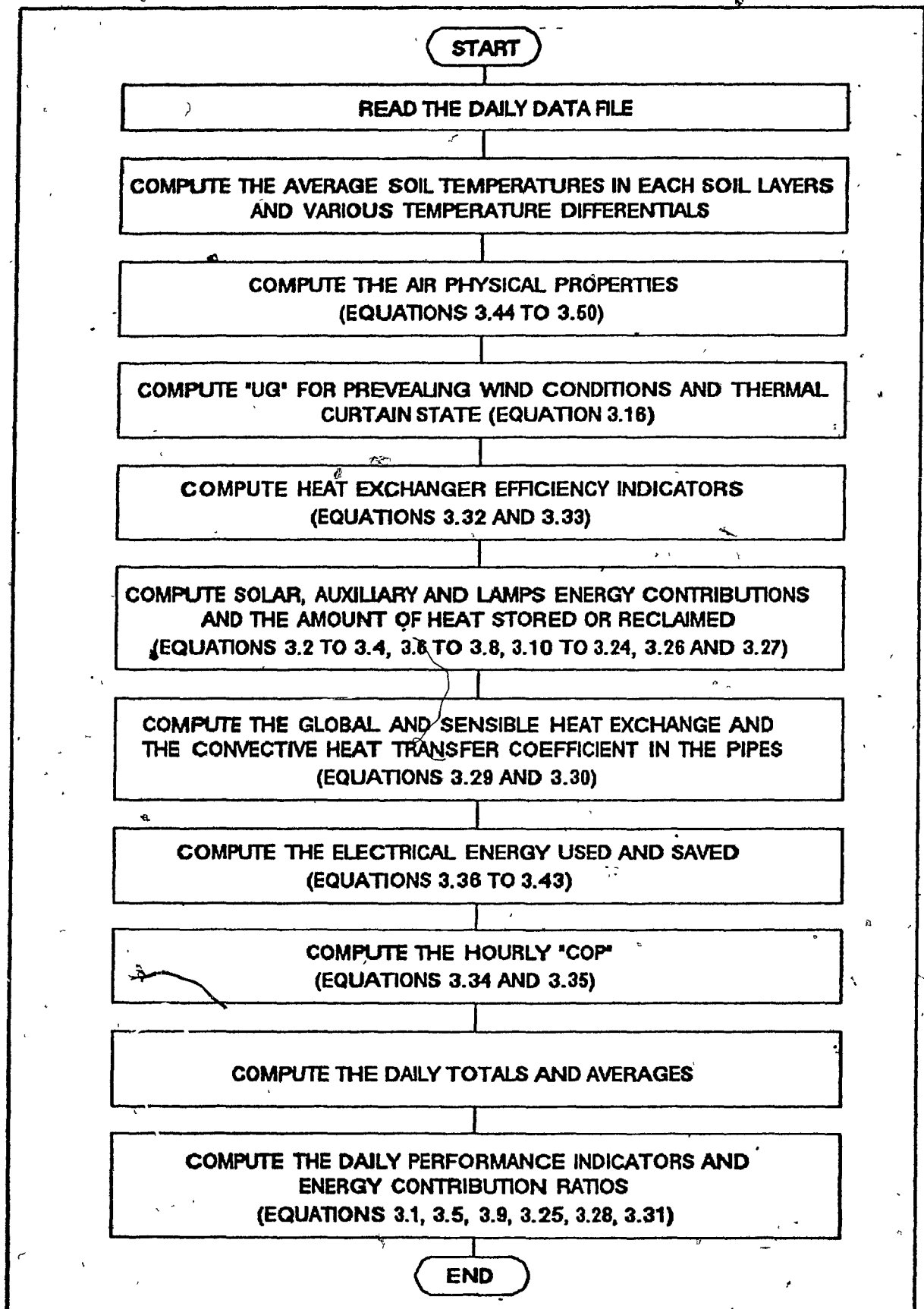


Figure 4.10 Flowchart for the computation of the daily performance.

different steps involved in the computation of the daily performance parameters.

Table 4.4 Schedule of events for equipment operation

Equipment	Controlling	Setpoint	Hysteresis	Test Period
Heater	TIG	15.0°C	1.0°C	Fall 1985
Heater	TIG	13.0°C	1.0°C	Spring 1986 Fall 1986
Blower	TCSD	different values	1.0°C	Fall 1985
Blower	TCSD	different values	1.0°C	Spring 1986 Fall 1986
Air Inlet	TIG	23.5°C	1.5°C	Entire
Air Exhaust (low speed)	TIG	25.0°C	1.0°C	Entire
Air Exhaust (high speed)	TIG	27.0°C	1.0°C	Entire
Auxiliary Fan	RH	80 %	5 %	Entire
Curtain	ISC	5.0	3.0	Entire
HPS Lamps	Time	6.00 to 22:00	-	Spring 1986 Fall 1986
Irrigation ¹	Irrrometer	200 mbars	100 mbars	Entire

¹ Operating time not logged by the data acquisition system.

4.2.6 Operating Modes

Different modes of operation were tested over periods ranging from a few days to several weeks. For each mode, the measurements mentioned in section 4.2.5 were made.

4.2.6.1 Alternate cycles of active storage and recovery

During the day, the heat exchanger-storage system stored excess heat, and during the night, part of the stored heat was recovered actively by the system. The exchanger operation was based on air-soil temperature differentials ($|T_{CSD}|$) ranging from 1 to 10°C.

4.2.6.2 Active storage and passive recovery

The system was put in operation during the day only to store excess heat. During the night, part of the stored heat was recovered passively by convection at the soil surface. The daytime operation was based on the temperature differentials (T_{CSD}) described above in section 4.2.6.1.

4.2.6.3 Continuous Operation

The system was operating 24 hours a day ($|T_{CSD}|=0^{\circ}\text{C}$).

4.2.7 Observations on the crops

In order to get a plant response indicator in connection with the heat exchanger-storage system and the greenhouse operation, tomato plants were grown, because it is the largest vegetable greenhouse production in Québec (C.P.V.Q.). The Vendor cultivar was selected because of its popularity among greenhouse growers, when this study was undertaken.

For every production stage, from seedling to fruit harvest, the crop was managed according to the C.P.V.Q.

(1984) recommendations. The fertilization schedule proposed in that same reference, was followed. Fertilizers were added directly to the irrigation system, and provided to each plant through a standard drip irrigation micro-tube. The fertilizer injector has a fixed injection proportion of one liter of liquid fertilizer for every 128 liters of irrigated water. Irrigation was controlled by the use of three irrometers mounted in parallel as described in section 4.1. The irrometers were preset to activate the irrigation system whenever the tension went below 200 mbars.

The plants were soil grown according to the plant layout presented in Figure 4.11; the rooting depth extended from 15 to 30 cm below the soil surface. Yields were measured on soil grown tomato plants. These yields were compared to the average yields obtained by Quebec greenhouse owners.

4.2.8 Payback Estimation

The payback period for various scenarios was estimated using the procedure outlined by Perry and Robertson (1980). These scenarios involved different initial capital costs, auxiliary heating costs in connection with different energy sources, operating costs and plant productivity, for a commercial type greenhouse having a surface area of 179 m², producing 3560 kg of tomato annually. An inflation rate of 4 % and an energy conservation factor of 33 % were used in the calculations. The costs are detailed in Table 4.5 and the scenarios are presented in Table 4.6.

Table 4.5 Cost estimates for the payback period

DESCRIPTION	COST ¹ (\$)
Material: 52 pipes twelve meters long, 0.70 \$/m	440
Plenum	160
Two ventilators	400
Two thermostats	200
Inlet pipe	100
Two dampers	200
Connection to the electrical circuit. (manpower: two hours)	100
Total	1600
Installation: Manpower for excavation 16 hours at 40 \$/hour	640
Manpower for installation 15 days at 80 \$/day	1200
Total	1840
Material plus Installation:	3440
Operation: Electricity, 3080 kWh at 0.0396 \$/kWh	122
Maintenance over a ten year period	100
Heating: Heat load 52320 kWh	
Heat source: heating oil at 0.037 \$/kWh	1936
natural gas at 0.029 \$/kWh	1507
wood chips at 0.008 \$/kWh	419
Productivity: 960 kg increase ² in yield at 3.50 \$/kg	3360

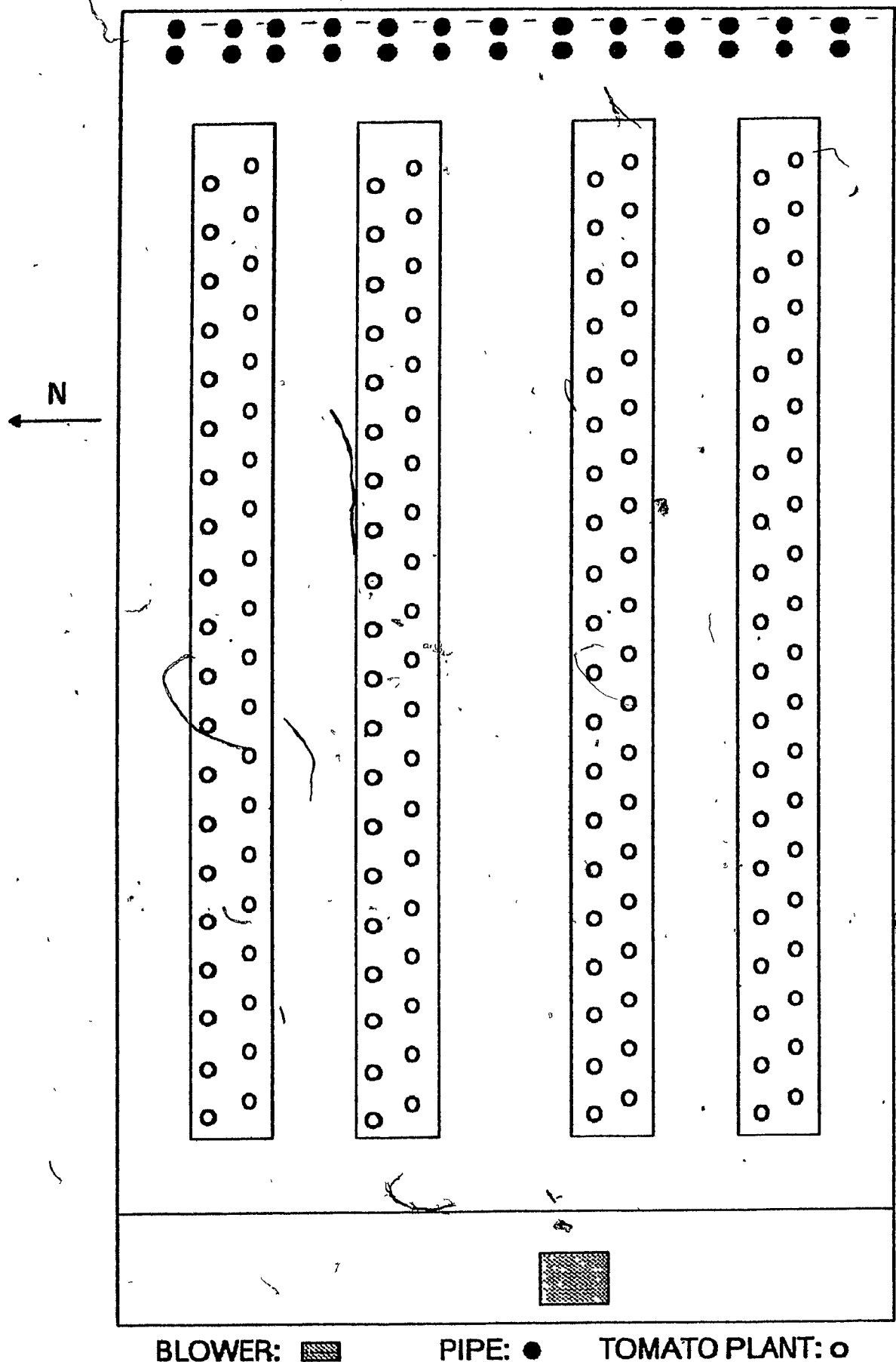
1 1986

2 Based on two tomato crops

Table 4.6 Payback period scenarios

SCENARIO #	AUXILIARY HEAT SOURCE	INITIAL CAPITAL COST (\$)	OPERATING COST (\$)	PLANT PRODUCTIVITY CONSIDERED
1	Heating oil	1593 ¹	0 ³	No
2	Heating oil	3433 ²	0	No
3	Heating oil	1593	222	No
4	Heating oil	3433	222	No
5	Heating oil	3433	222	Yes ⁴
6	Natural gas	3433	222	Yes
7	Wood chips	3433	222	Yes

- ¹ The greenhouse owner buys and installs the material required for the heat exchanger-storage system.
- ² The greenhouse owner buys the material and pays for the installation of the heat exchanger-storage system.
- ³ The heat exchanger-storage system blower replaces the standard fan jet normally found in a greenhouse.
- ⁴ The increase in plant yield is assumed to be 960 kg.



BLOWER: ■

PIPE: ●

TOMATO PLANT: ○

Figure 4.11 Layout of tomato plants in the greenhouse.

V RESULTS AND DISCUSSION

5.1 Soil Physical Properties

5.1.1 Soil Texture

A soil texture analysis was performed on the two soil layers forming the thermal mass; the bottom layer accounting for approximately 2/3 of the thermal, more samples were taken from this layer; sampling was done at 65 and 95 cm below the soil surface in the bottom layer and at 15 cm in the top layer. The results are shown in Tables 5.1 and 5.2. These results indicate that the bottom layer is made of sand and the top layer is made of sandy loam. Both layers are quite homogeneous.

5.1.2 Soil Wet Bulk Density

The average soil wet bulk densities obtained from different sampling are shown in Table 5.3. The top layer being more easily accessible, more samples were taken in this layer. The average density of the bottom layer is higher than that of the top layer, since the top layer was tilled and loosened prior to each crop planting. For both layers the density distribution is uniform.

5.1.3 Soil Specific Heat

The soil specific heat values obtained from dried composite samples coming from both soil layers, are shown in

Table 5.1 Texture of the soil bottom layer

Sample #	Sand (%)	Silt (%)	Clay (%)
1	93.0	4.4	2.6
2	93.2	3.2	3.6
3	93.3	3.2	3.5
4	92.0	4.0	4.0
5	94.0	2.5	3.5
6	91.8	3.6	4.5
7	96.7	0.8	2.5
8	95.7	1.7	2.6
9	97.3	0.2	2.5
10	96.4	0.0	3.6
11	96.9	0.6	2.5
12	96.5	1.0	2.5
Mean	94.7	2.1	3.2
S.D.	2.1	1.6	0.7
C.V. (%)	2.2	78.0	23.4

Table 5.2 Texture of the soil top layer

Sample #	Sand (%)	Silt (%)	Clay (%)
1	62.5	13.4	14.1
2	62.5	15.1	12.4
3	68.6	21.0	10.4
4	70.6	17.0	12.4
Mean	66.0	16.6	12.3
S.D.	4.2	3.3	1.5
C.V. (%)	6.3	19.6	12.3

Table 5.3 Soil wet bulk density

Layer	Number of samples	Average bulk density (kg/m ³)	Standard deviation	Coefficient of variation (%)
Sandy loam	47	1340	51	3.8
Sand	40	1736	40	2.3

Table 5.4. These results are in agreement with values generally encountered in the literature, for these types of soil (Baver, 1940; Van Wijk, 1963; Lawand et al., 1983).

5.1.4 Soil Water Content

Table 5.5 shows the average water content on a dry basis, for each soil layer. In this case, based on the coefficient of variation, the distribution is not as uniform as it was for soil wet bulk density, especially in the sandy layer. Because of its higher water retention capacity, the top layer had a higher water content.

5.2 Air Velocity

5.2.1 Heat Exchanger

The air flowrates in the exchanger were equalized prior to the velocity measurements; these measurements were taken before the start of the test period. The values of the average air velocity for each pipe are listed in Table 5.6. The air velocities were relatively uniform, except for one pipe (A11), for which the air flowrate could not be adjusted, most likely due to an obstruction or pipe deformation.

5.2.2 Heater Outlets

The heater was equipped with three air outlets; the air flowrates in these outlets could not be perfectly equalized. The average air velocities for each outlet are listed in Table 5.7. The hot air from the heater was distributed in

Table 5.4 Soil specific heat

Layer	Number of samples	Mean specific heat (kJ/kg·°C)	Standard deviation (kJ/kg·°C)	Coefficient of variation (%)
Sandy Loam	24	0.77	0.14	18
Sand	20	0.68	0.06	9

Table 5.5 Soil water content

Layer	Number of samples	Mean water contents (% weight)	Standard deviation (% weight)	Coefficient of variation (%)
Sandy Loam	24	23.7	4.3	18
Sand	24	8.8	4.1	47

Table 5.6 Heat exchanger air velocity and flowrate

Pipe #	Velocity (m/s)	Flowrate (m ³)	Pipe #	Velocity (m/s)	Flowrate (m ³ /s)
A 1	4.1	0.032	B 1	3.9	0.031
A 2	4.0	0.032	B 2	4.1	0.032
A 3	3.9	0.030	B 3	3.9	0.031
A 5	3.3	0.026	B 5	3.6	0.028
A 6	4.1	0.032	B 6	3.9	0.030
A 7	3.9	0.031	B 7	3.8	0.030
A 8	4.1	0.032	B 8	4.2	0.033
A 9	3.9	0.031	B 9	3.9	0.031
A10	3.9	0.031	B10	3.9	0.031
A11	2.7	0.021	B11	3.9	0.031
A13	3.9	0.031	B13	3.7	0.029
Mean	3.81	0.0349		3.90	0.0357
Standard deviation	0.40			0.21	
C.V.(%)	11			5.23	

Table 5.7 Heater outlet air velocity and flowrate

Outlet	Velocity (m/s)	Flowrate (m ³ /s)
South	2.8	0.29
Center	3.3	0.33
North	2.4	0.24

the greenhouse through perforated polyethylene tubes. One of these tubes was located over the soil along the north wall. The second one was located on the greenhouse center line. The third one was installed along the south wall.

In order to minimize the total number of sensors used, two temperature sensors were used for the evaluation of the amount of heat provided by the heater. One sensor was installed at the heater air inlet, the other was installed at the central outlet. Since temperature differences were found among the three air outlets, the air temperature measured in the central outlet could not be used directly in equation 3.8 to compute the amount of heat provided by the heater. Therefore, the air temperature measured in the central outlet had to be adjusted in order to be representative of the average air temperature coming from the three outlets. To do that, test runs were conducted and air temperatures from each outlet were measured. A linear regression equation was derived from temperature measurements obtained during the test runs.

$$TOC = 1.0101 + 0.9527 \cdot TCE \quad (5.1)$$

This empirical equation predicted the average temperature of the air coming from the heater, from the air temperature of the central outlet. The results of the statistical analysis are presented in Appendix A.

5.3 Heat Exchange Coefficients

5.3.1 Greenhouse Heat Loss Coefficients

Results from the heat loss test runs performed on selected nights throughout the study period, are shown in Figure 5.1; from the figure, it can be seen that the overall heat loss coefficient varies linearly with wind velocity. The use of the thermal curtain decreases the overall heat loss coefficient by 28 to 30 %, which results in an energy conservation factor of the same magnitude. The use of hoods on ventilator outlets results in a decrease of the overall heat loss coefficient by 16 to 28 %, likely by reducing air infiltration. An example of the data obtained during a heat loss test run and the computer program used to compute the data are listed in Appendix B and C respectively.

The parameters UG, UGH, UGC and UGHC stand respectively for the overall heat loss coefficients without the thermal curtain and hoods (equation 5.2), without the curtain but with hoods, with the thermal curtain but without hoods (equation 5.3), and with both the curtain and the hoods.

To simplify the computations when the thermal curtain was on, 72 % of the UG value corresponding to a given wind condition without the thermal curtain, was used; which yields a conservative estimate of the heat losses. Furthermore, four test runs made at different periods without wind, thermal curtain or hoods, have yielded a UG value of $0.0051 \text{ kW/m}^2 \cdot ^\circ\text{C}$, which is lower than the UG value predicted by the linear regression equation 5.2. It seems that the

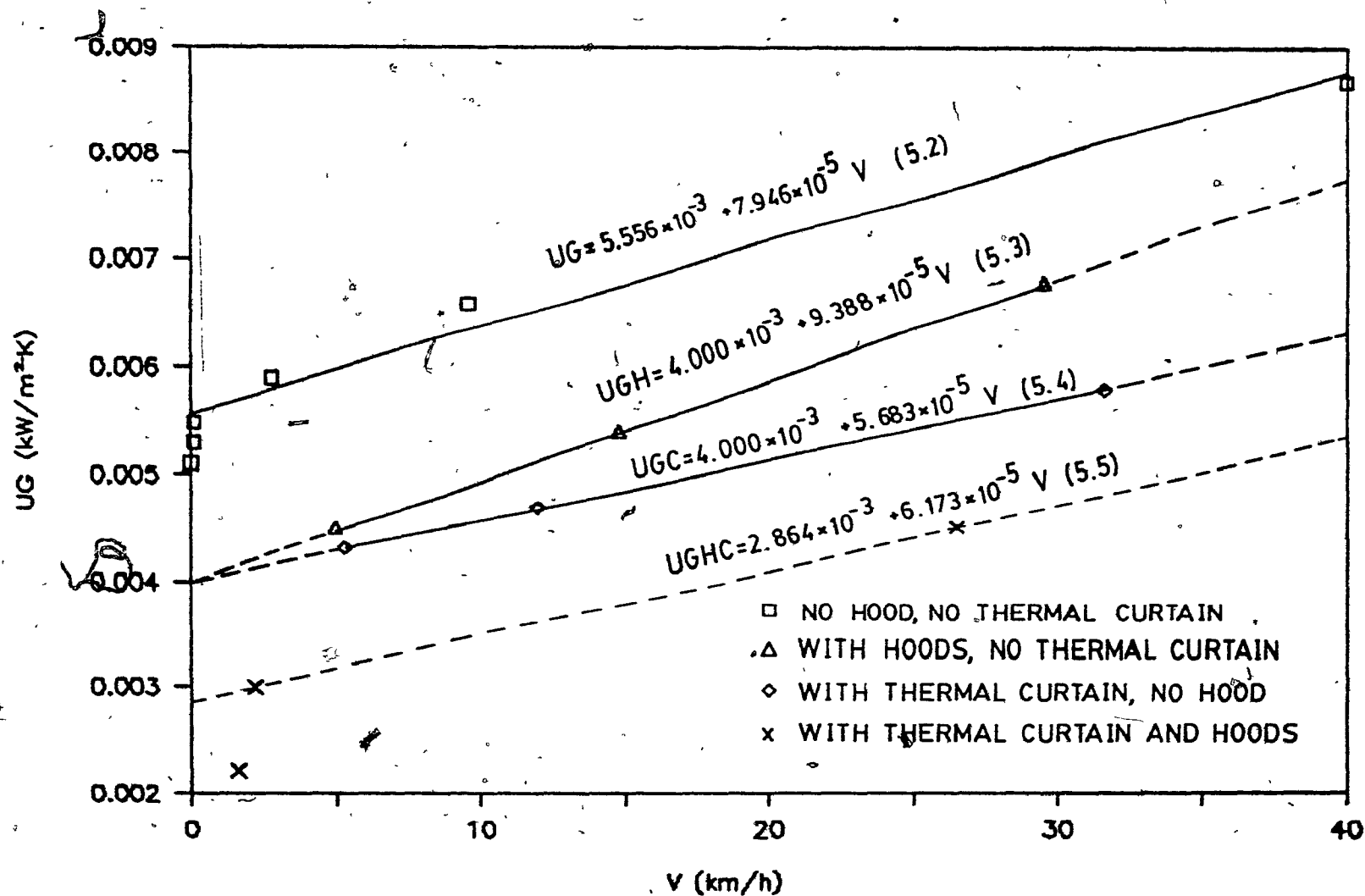


Figure 5.1 Wind effect on the overall heat loss coefficient.

infiltration heat losses are relatively important, since the greenhouse is exposed to wind conditions; because of that, a UG value of 0.0051 was used in order to prevent the over estimation of the greenhouse heat losses, which would contribute to the over estimation of the solar contribution to the greenhouse heat load.

5.3.2 Convective Heat Transfer Coefficient

The convective heat transfer coefficient in the heat exchanger has been estimated from the empirical equation given by Sibley and Raghavan (1984), for the temperature conditions encountered on the 86-12-01, the value obtained is $0.015 \text{ kW/m}^2 \cdot \text{K}$.

For the same conditions, the value computed from the experimental data using equation 3.30 was $0.013 \text{ kW/m}^2 \cdot \text{K}$, based on sensible heat exchange. The difference might be explained by the fact that the empirical equation of Sibley and Raghavan was derived from the data obtained with a laboratory setup using relatively dryer air; in this case, the air circulating in the heat exchanger storage system was at 80 % (average) air relative humidity. Therefore, the physical properties of air and the heat transfer mechanisms involved are not likely to be the same. An average value of $0.012 \text{ kW/m}^2 \cdot \text{K}$ was obtained for the spring and fall of 1986 test periods (Appendix D).

5.4 Electrical Energy Used

Measurements taken on the electrical equipments are presented in Table 5.8. However, these values do not account for variations in the electricity supply.

Table 5.8 Voltage, current and electrical power

Equipment	Voltage (V)	Current (A)	Power (W)
Blower	240	4.9	1.76
Ventilator Low Speed	122	3.2	390
Ventilator High Speed	121	6.0	726
Ventilator Humidity	123	5.5	677
Burner	123	5.9	725
High Pressure Sodium Lamps	122	3.9 ¹	475

¹ estimated

5.5 System performance

All the data collected were processed on a daily basis, using the "Lotus 1-2-3" software. Pretreated and compiled data account for 800 pages of computer output, and therefore, these pages are not included in the thesis. However, these data are available for consultation on the I.T.A. La Pocatière premise. For illustration, raw, pretreated and compiled data for a given day are presented in Appendix E.

The computer program used to process the data is listed in Appendix F.

5.5.1 Heat Storage and Recovery

During the test period, the thermal mass had seasonal temperature fluctuations of approximately 10°C , as shown in Figure 5.2. During the fall period, the thermal mass temperature decreased gradually down to a temperature slightly above the minimum air temperature maintained (13 to 15°C). In the spring, during the storage phase, the higher rates of temperature increase were encountered in April and May. In summer, since the ventilators work more often and the thermal mass temperature is high, the efficiency of the exchanger decreases and the rate of temperature increase decreased.

In early summer, the average soil temperature went up to 24°C . This increase in temperature as compared with a conventional greenhouse, could have a positive impact on plant yield, provided that the root zone has a good thermal contact with the thermal mass.

For a typical spring day, the temperature of the soil layers has a sine wave type variation. The curves corresponding to the top and bottom layers of soil, show that the system has a significant effect on soil temperature (Figure 5.3). The system was operated during the day for heat storage and at night, for heat recovery.

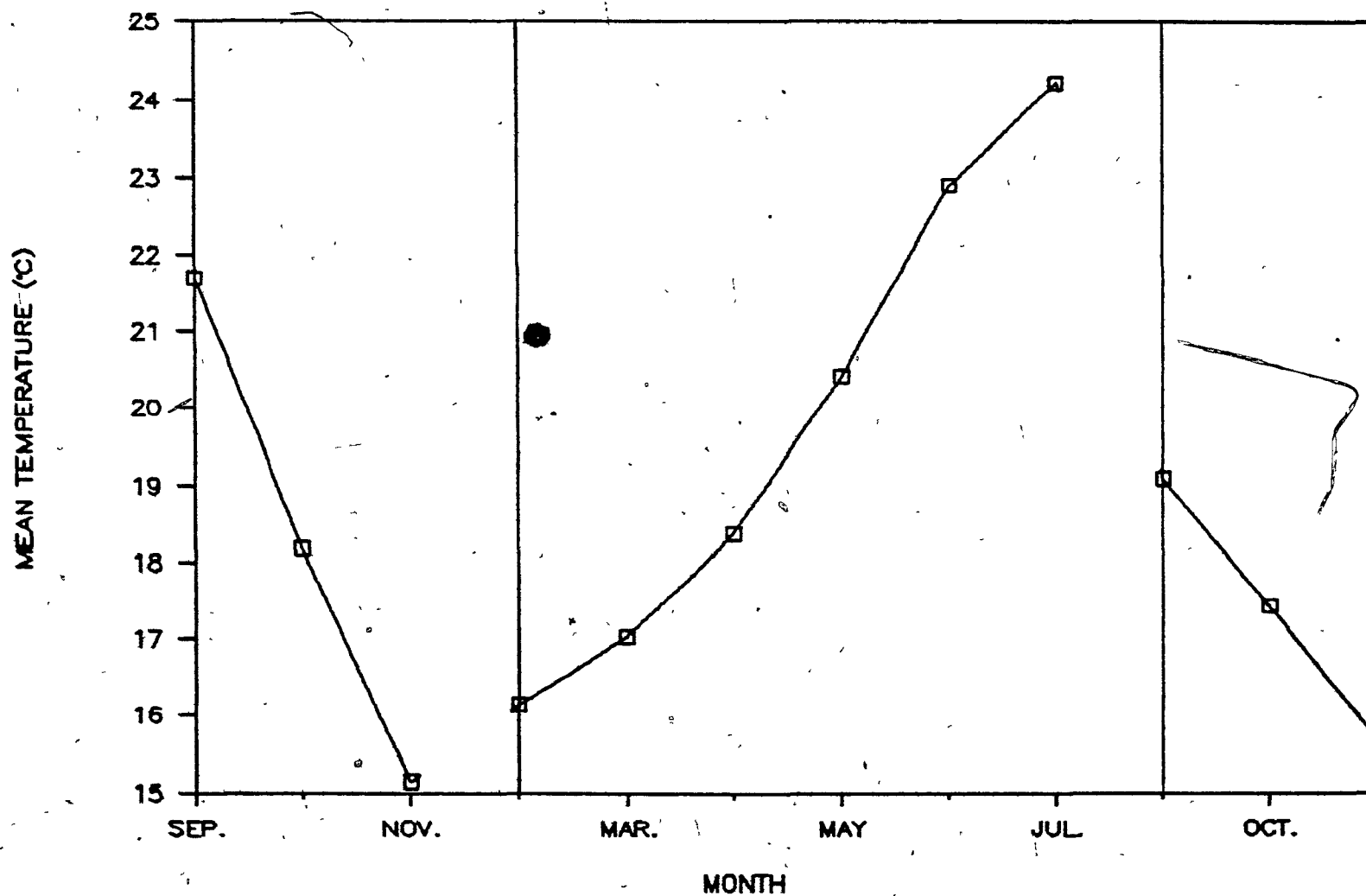


Figure 5.2 Average thermal mass mean temperature fluctuation for the periods of September through November 1985 and 1986, and February through July 1986.

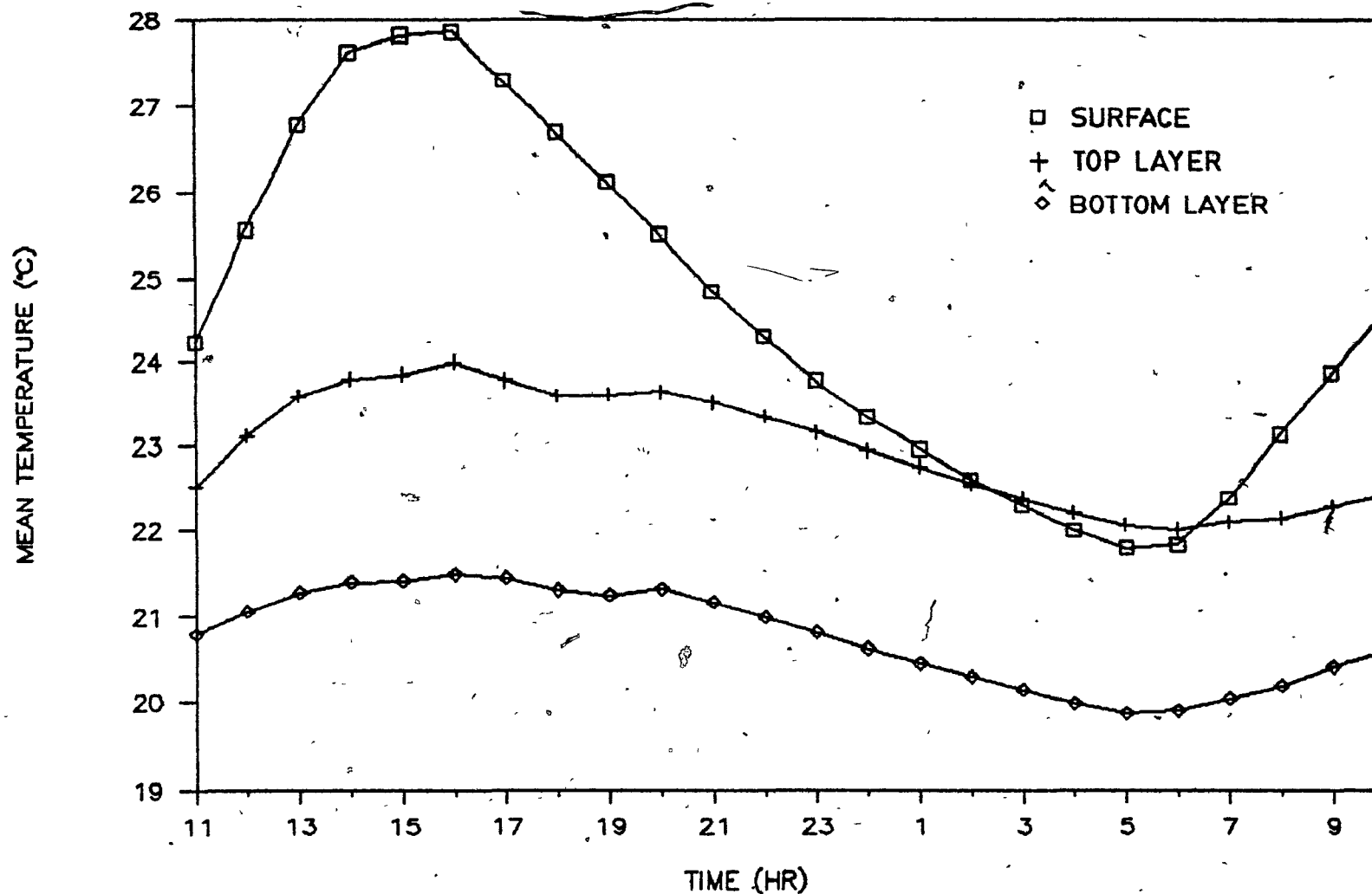


Figure 5.3 Soil daily temperature variation during the storage phase in the spring of 1986.

The soil temperature variation at the surface has a higher amplitude than the one observed for the bottom layer, since top soil also absorbs directly part of the incident solar radiation. During the night, the temperature at the soil surface goes below the temperature of the top layer; therefore, some stored heat migrated to the soil surface and is released to the greenhouse.

The daily variation in soil temperature for a given day during the first fall test period is shown in Figure 5.4, the system was then operated during daytime only for heat storage. During nighttime, heat is gradually lost from the soil layers, since the soil temperature is higher than the air temperature inside the greenhouse, part of that heat is recovered passively inside the greenhouse.

During spring, the temperature is relatively uniform throughout the thermal mass. By mid-spring, the temperature differentials between the west end and the east end of the thermal mass are approximately 1.5 and 0.5°C for the bottom and top layer of soil respectively (Figure 5.5). By early summer, the temperature differentials become negligible for the top layer and approximately 1°C for the bottom one (Figure 5.6).

Soil surface temperatures measured in the center of the greenhouse are consistently higher than the ones measured close to the west end and east end for the greenhouse; this is likely to be due to the shading effect of the end walls on these area in early morning and late afternoon.

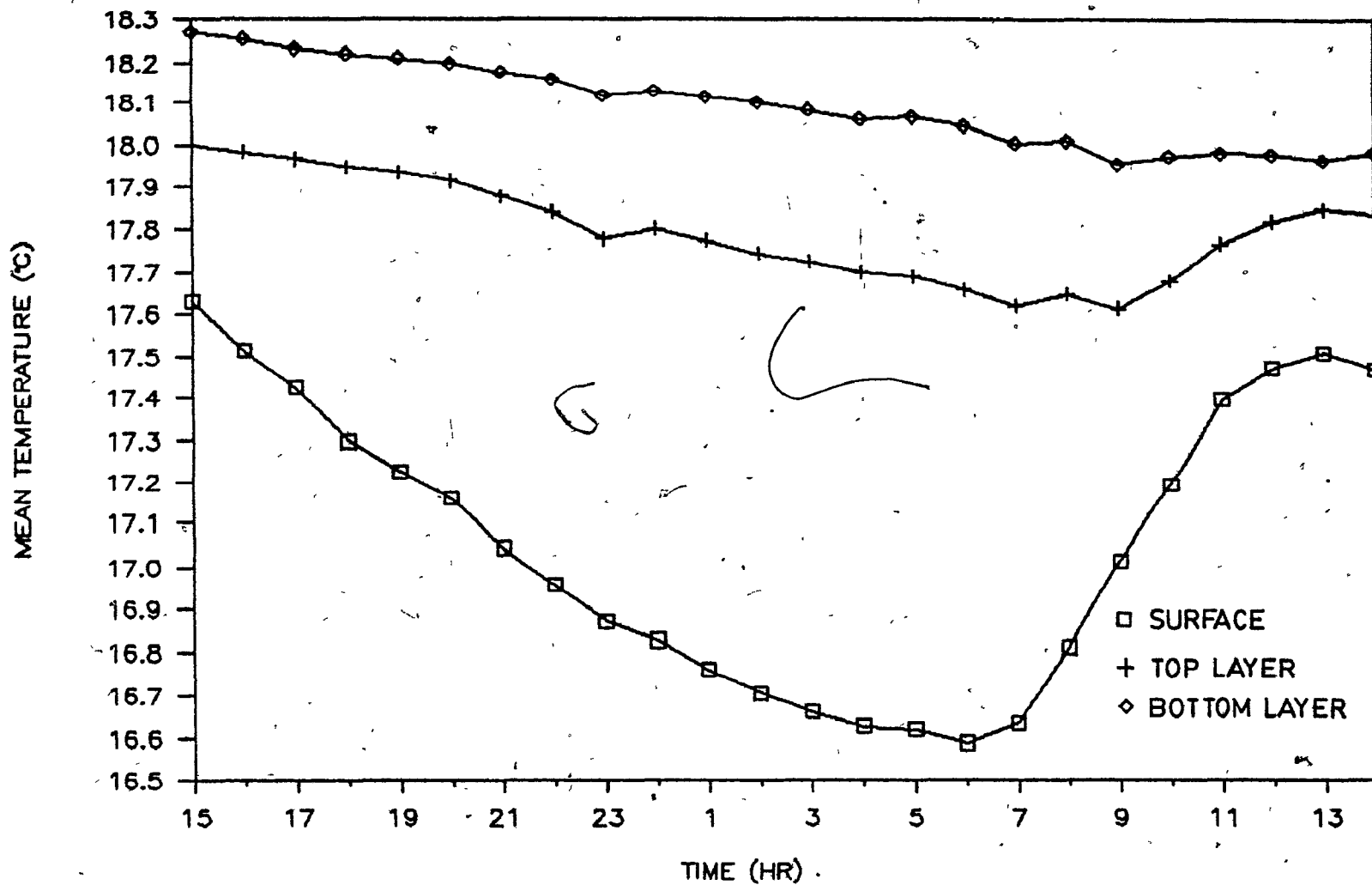


Figure 5.4 Soil daily temperature variation in the fall of 1985 with passive heat recovery.

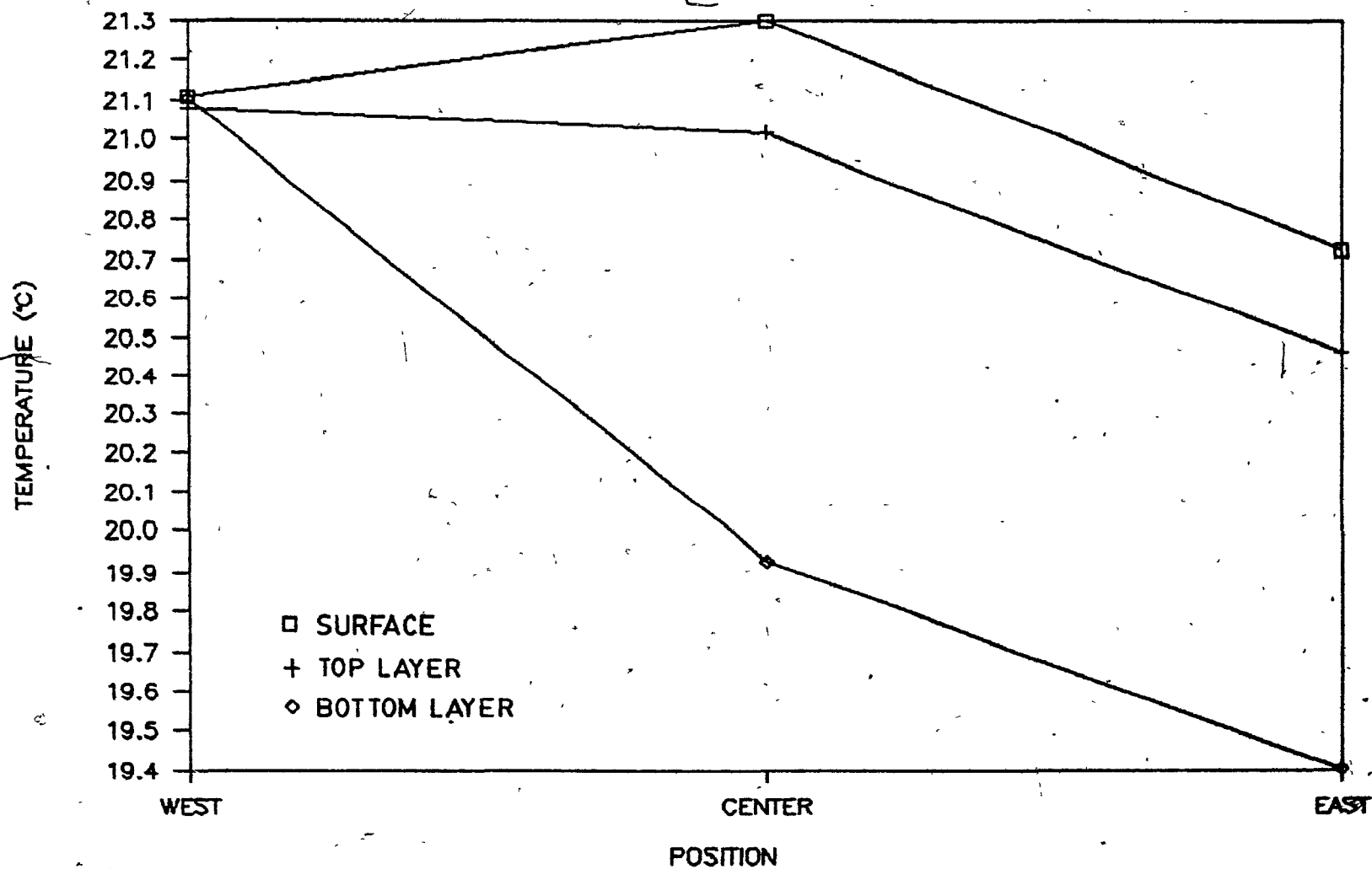


Figure 5.5 Soil longitudinal temperature profile for mid-spring.

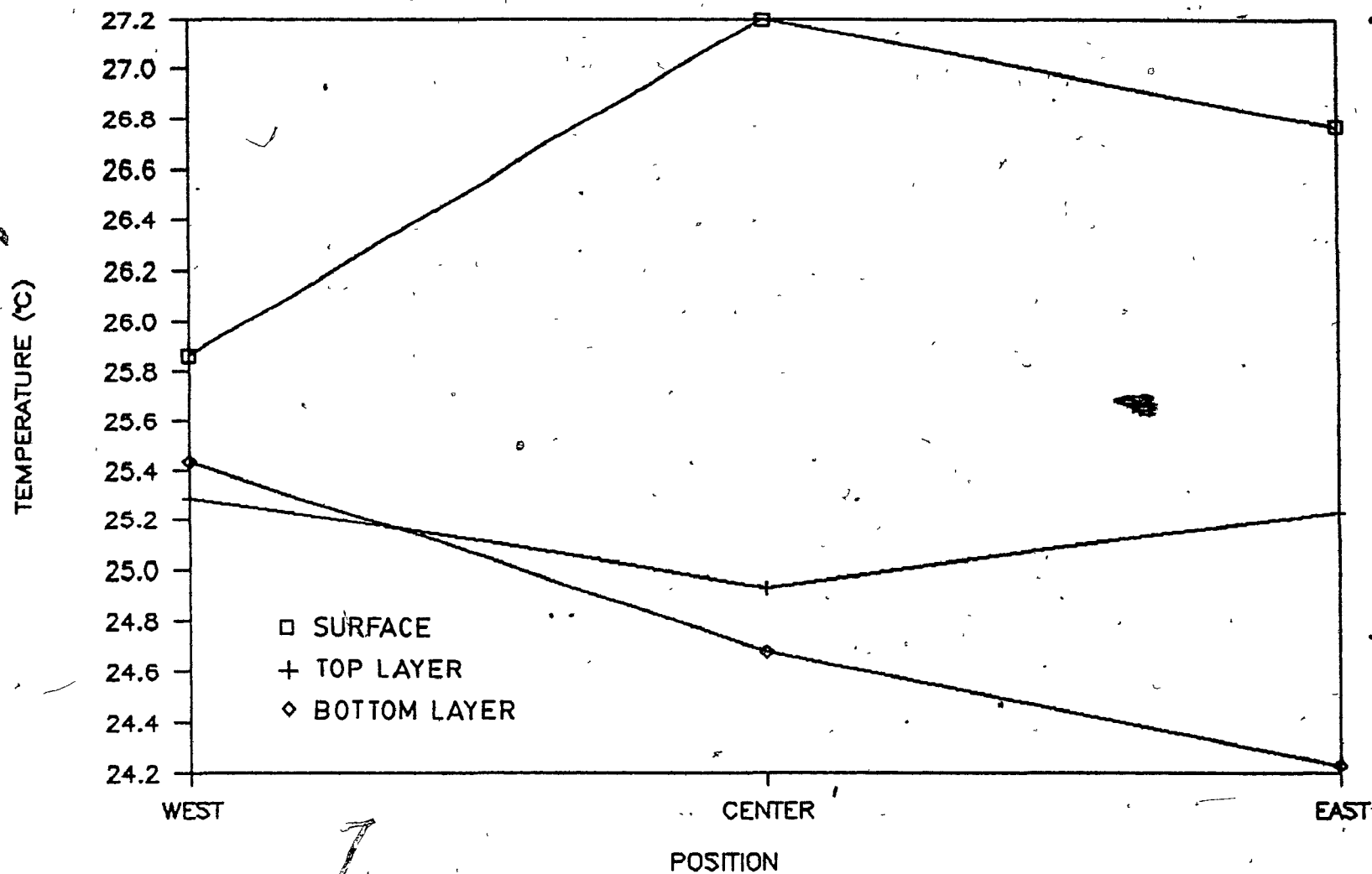


Figure 5.6 Soil longitudinal temperature profile at the beginning of summer.

In order to assess the effect of active and passive heat recovery on the heat exchanger-storage system performance, the system was operated day and night whenever excess heat was available, during the second fall test period. The soil longitudinal temperature profile had somewhat the same pattern (Figure 5.7). However, the temperature of the bottom layer was 1°C higher than that of the top layer. By mid-season, the temperature differentials between the west end and the east end of the thermal mass were in the order of 2°C . By early winter, the temperature differential increased up to 5°C . The soil surface temperature somewhat followed the same pattern as for the top layer of soil (Figure 5.8). On the overall, there was little horizontal temperature stratification in the thermal mass.

The air temperature differentials between the exchanger inlet and outlets seems to follow the same pattern as the air inside the greenhouse does (Figure 5.9).

For a typical sunny day in spring, the temperature differential during the day is higher for the bottom row of pipes as compared with the top row, since the soil surrounding the pipes is colder in the bottom layer; for the same reason, the temperature differential obtained at night is lower for that same row; nevertheless, this temperature differential is relatively constant. The capacity of the system to recover heat at night is limited by the low temperature differentials (between 3 and 5°C) encountered. The maximum and minimum temperature conditions imposed by the crop

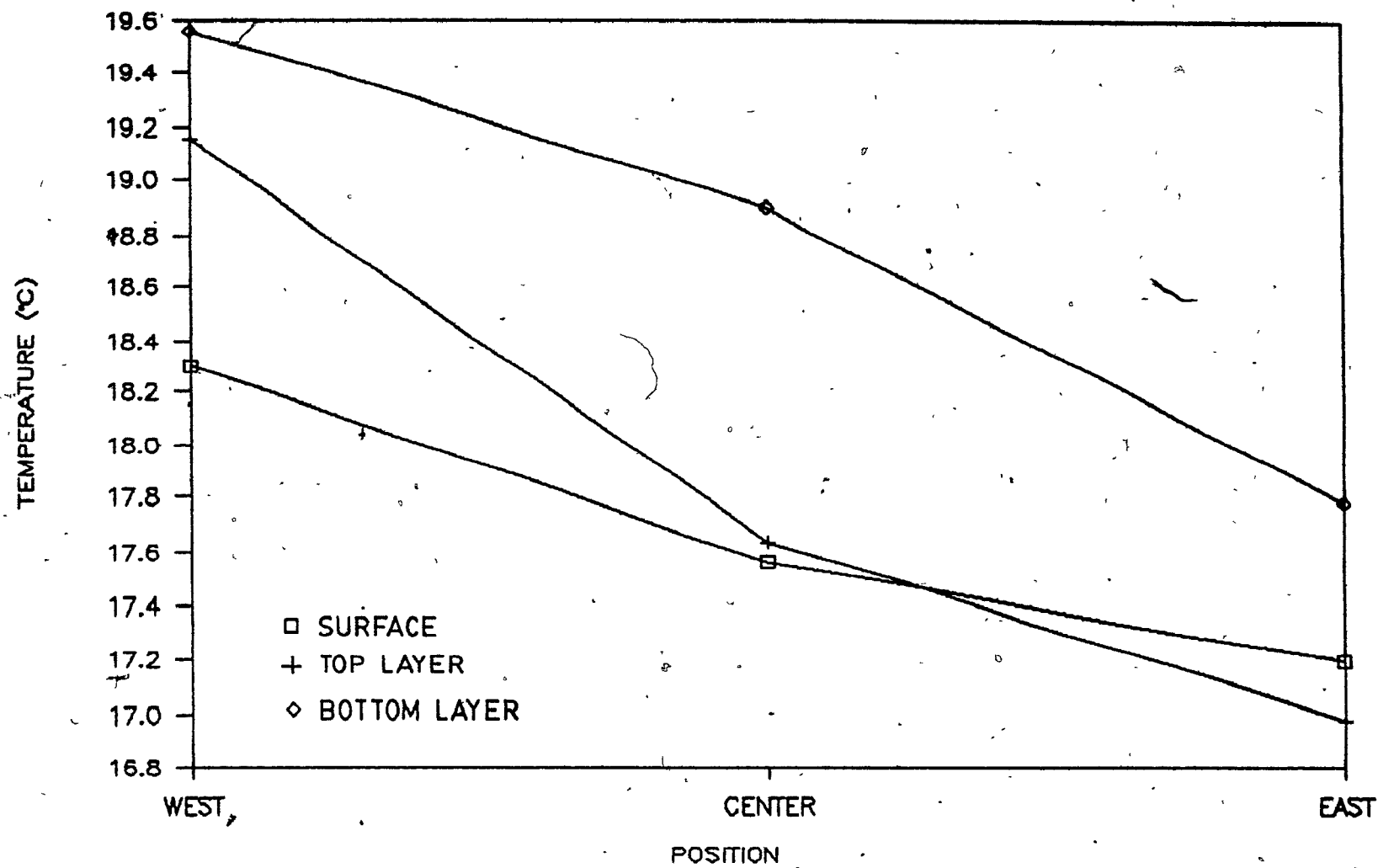


Figure 5.7 Soil longitudinal temperature profile during heat recovery in the fall of 1986.

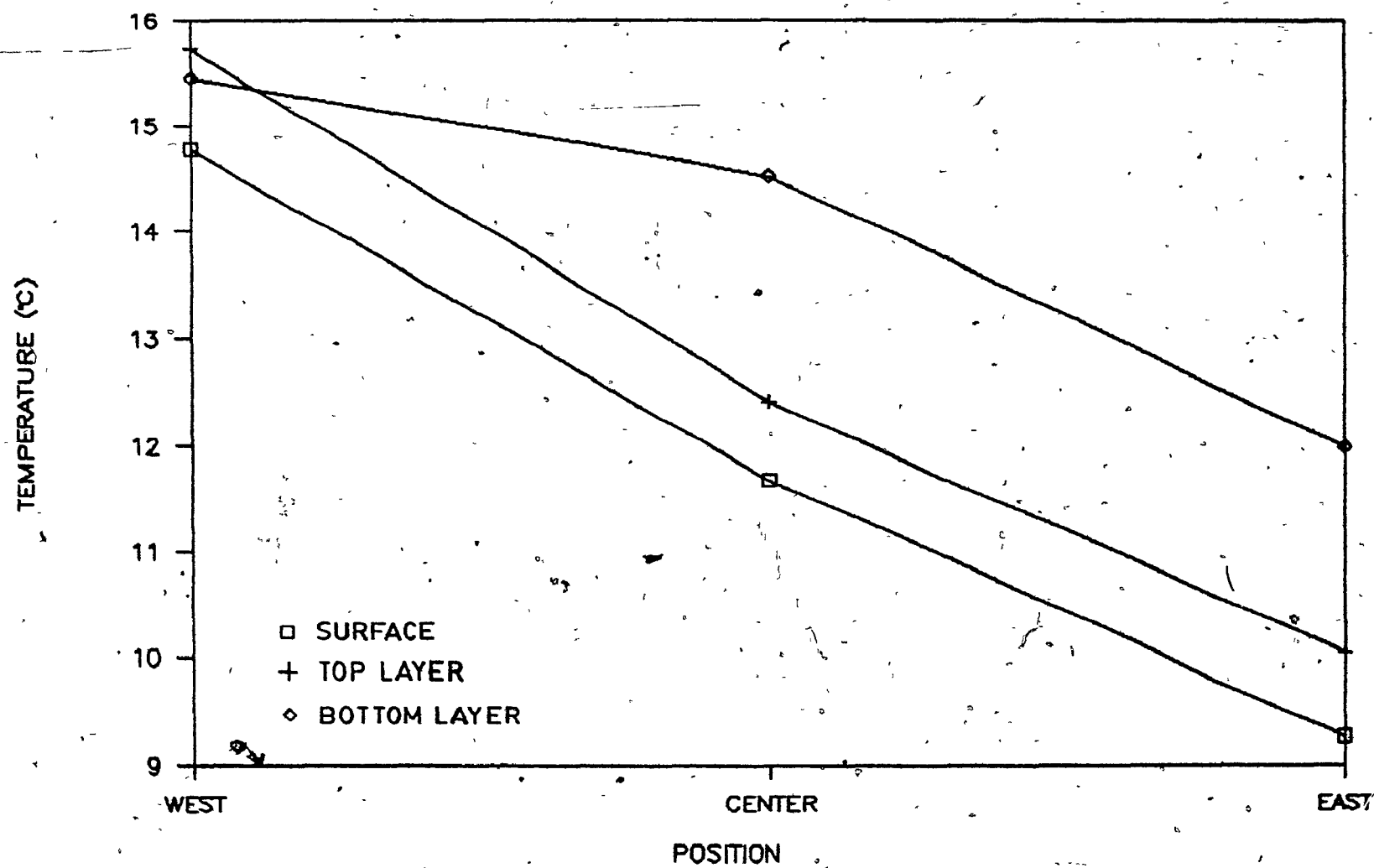


Figure 5.8 Soil longitudinal temperature profile at the end of fall.

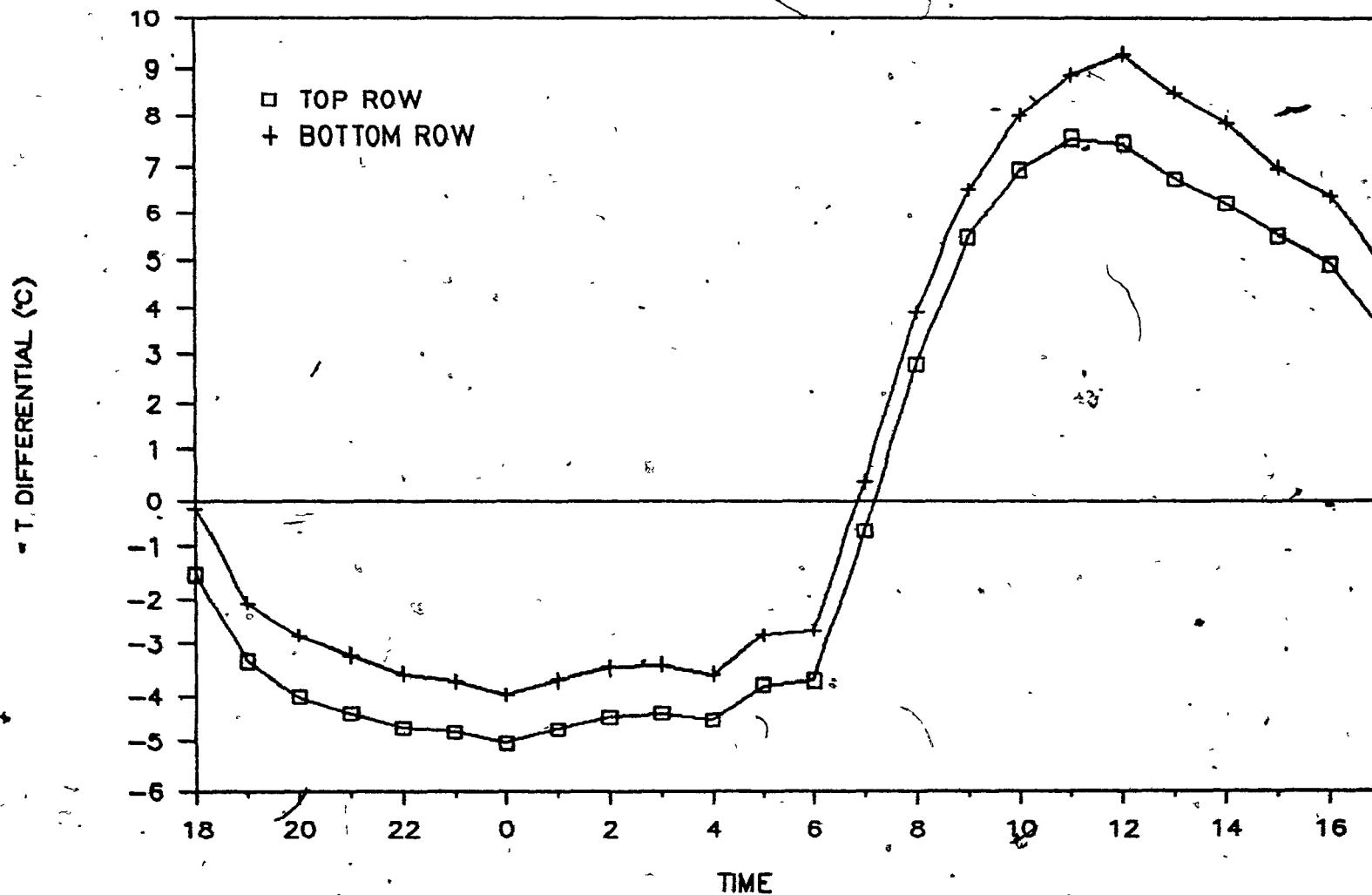


Figure 5.9 Daily variation of the temperature differential between the heat exchanger inlet and outlet.

prevent the operation of the system at higher temperature differentials.

During heat storage, after 4 hours of operation, the air temperature inside the pipes decreases rapidly between the pipe inlet, located at the west end of the greenhouse, and the pipe center, a temperature drop of 6°C was recorded. Between the pipe center and the pipe outlet, located at the east end of the greenhouse, the decrease was less steep, and the value was 2°C (Figure 5.10).

After 8 hours of operation, a similar profile is observed (Figure 5.11). As shown in Figures 5.10 and 5.11, it appears that the air circulated in the heat exchanger loses approximately the $2/3$ of its heat content in the first 5 meters of pipe. Since the top layer of soil is warmer than the bottom layer, the temperature differential between the air and soil rapidly approaches zero.

For heat recovery, the situation is reversed (Figure 5.12). After 8 hours of operation, the air-soil temperature differential being higher for the top row of pipes as compared with the bottom row, the heat exchange takes place on the full length of the top row pipes.

There is a good correlation (Figure 5.13) between the average soil temperature and the average air temperature inside the greenhouse. This further confirms that the system performance is limited by the maximum and minimum temperature conditions imposed by the crop. The soil temperature could get higher provided that a higher air temperature

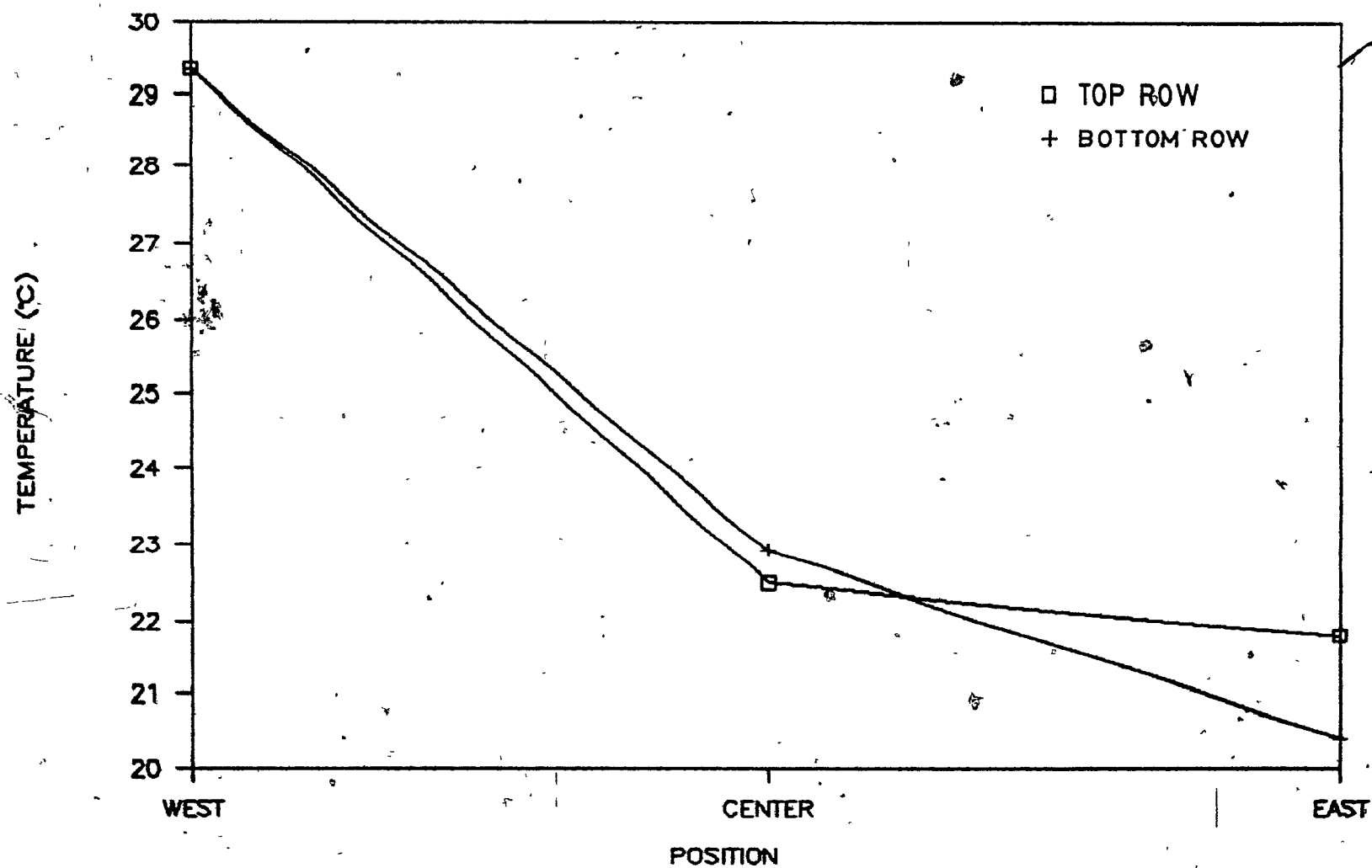


Figure 5.10 Air longitudinal temperature profile in the exchanger pipes, after 4 hours of heat storage.

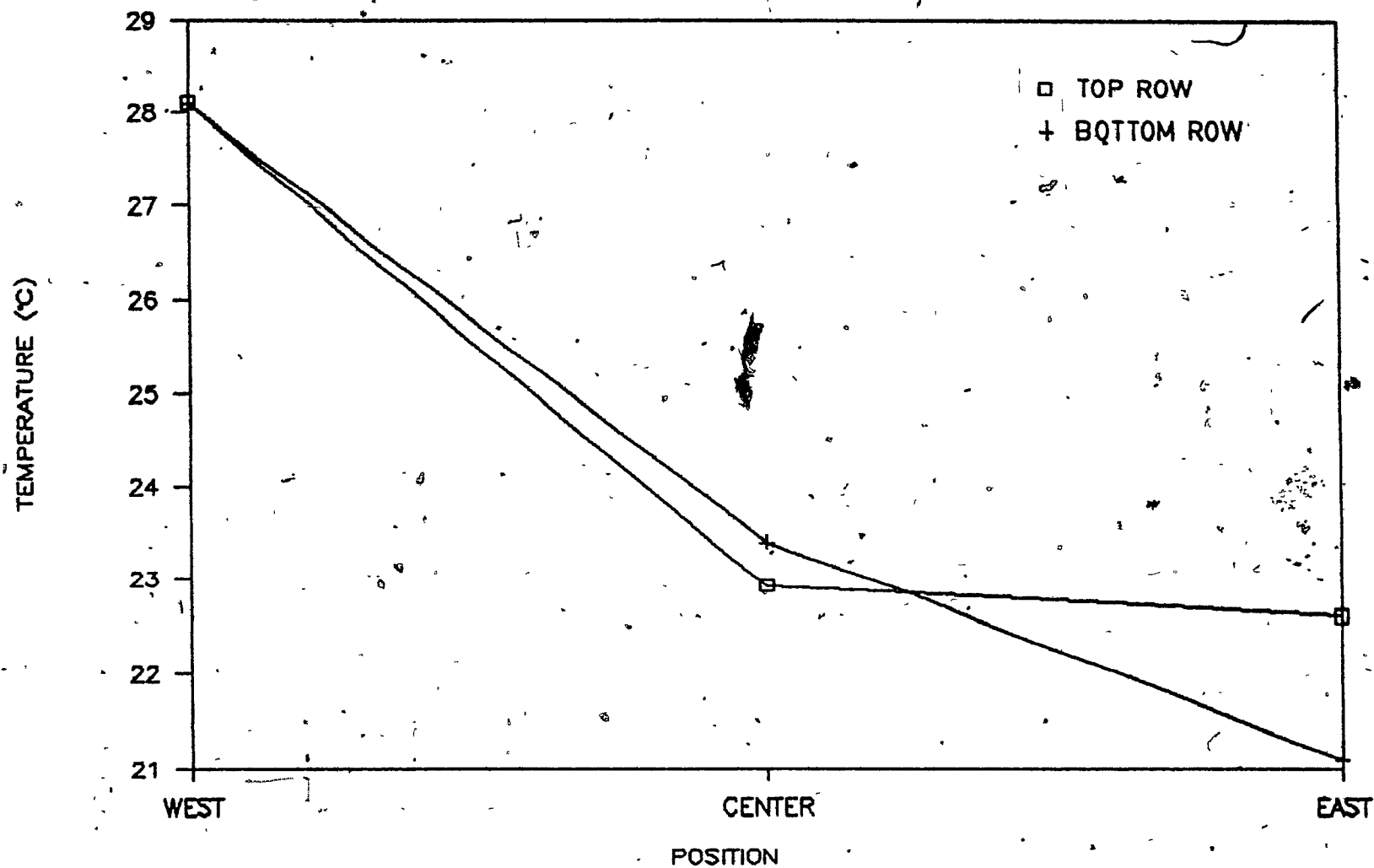


Figure 5.11 Air longitudinal temperature profile in the exchanger pipes, after 8 hours of heat storage.

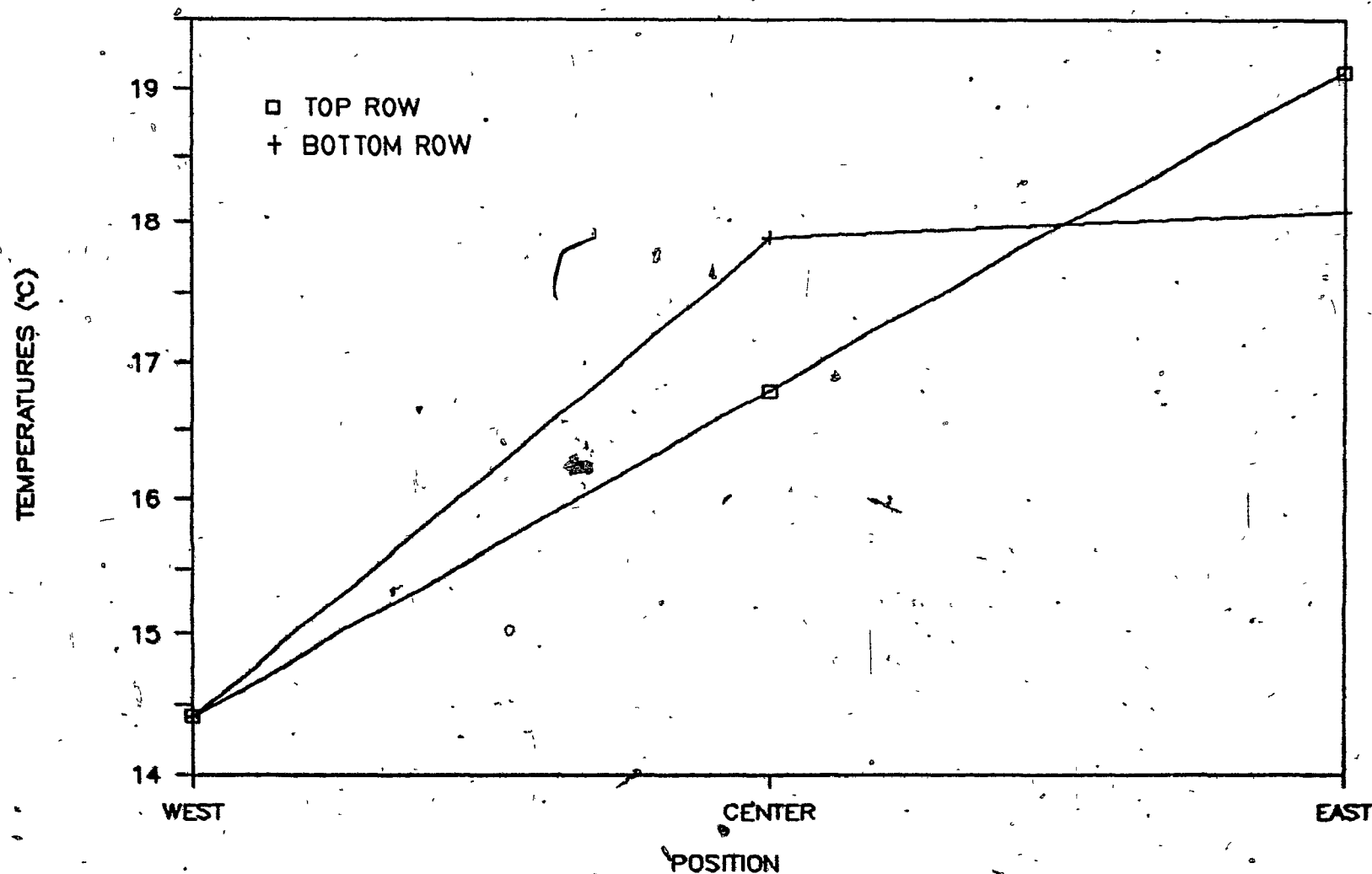


Figure 5.12 Air longitudinal temperature profile in the exchanger pipes after 8 hours of heat recovery.

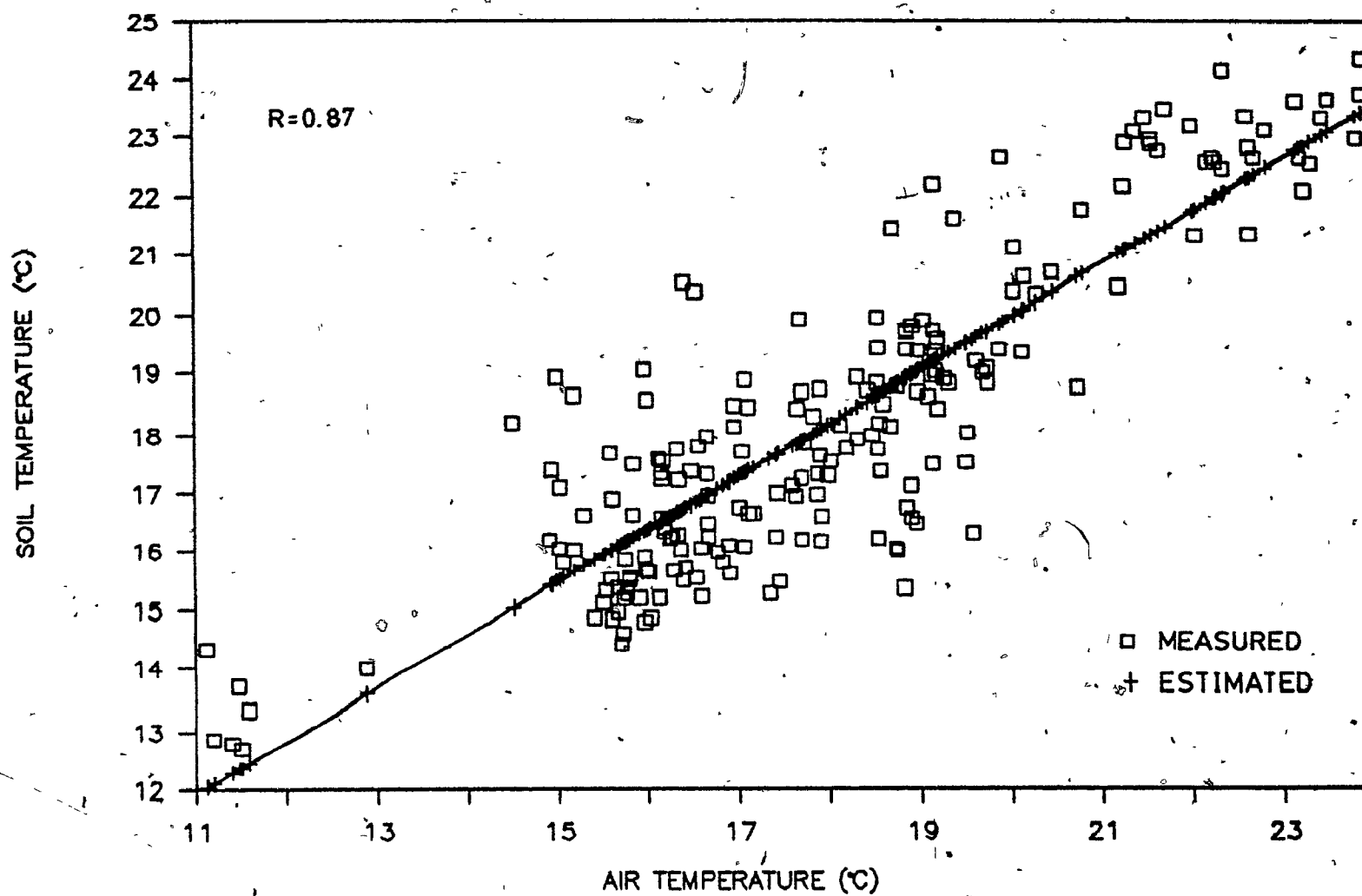


Figure 5.13 Soil-air temperature relationship.

could be allowed inside the greenhouse during the day. On the other hand, a lower air temperature at night would result in more heat recovered from the storage.

The previous results indicate that heat was stored and retrieved from the thermal mass, but they do not give any indication on the efficiency of these operations. To find out, heat storage (SE) and recovery (RE) efficiency parameters were computed from equations 3.25 and 3.28 respectively; however, these parameters did not turn out to be good indicator of the system performance. Results for the month of May, shown in Table 5.9, extracted from Appendix D, indicate that these two parameters seem to fluctuate randomly. There is no apparent relationship between these two parameters and other system performance indicators.

Two apparent reasons might explain this fact. Firstly, the soil water content has not remained constant throughout the test period. For example, prior to each crop planting the soil was washed through, to decrease the salt content. The irrigation of plants was not perfectly uniform, resulting in soil water content spacial variability, mostly in the sand layer (Table 5.4). The soil water content of the top layer is affected by the amount of solar energy intercepted at the surface, as more energy is received, more evaporation heat losses take place. Since the soil thermal properties are highly dependent on the soil water content, it does not seem to be appropriate to assume that the soil water content remained constant on a daily basis. Secondly,

Table 5.9 Fluctuation of the heat storage and recovery parameters (May 1986)

Date	Greenhouse Heat Loss (kJ)	Solar Energy (kJ/m ²)	Mean Soil Temp. (°C)	Storage Thermal Efficiency	Recovery Thermal Efficiency
01/05/86	840145	8794	19.0	0.58	0.50
02/05/86	1075260	14425	18.8	0.53	0.49
03/05/86	1117965	15772	19.4	0.48	0.37
04/05/86	705140	4346	18.9	0.55	0.79
05/05/86	766120	14606	18.5	0.54	0.66
07/05/86	758389	8411	17.3	0.72	0.83
08/05/86	800104	18570	18.2	0.48	0.37
09/05/86	467592	12669	19.4	0.57	0.35
10/05/86	467592	12669	19.4	0.57	0.35
11/05/86	1090588	17857	18.7	0.51	0.57
12/05/86	725217	19826	19.4	0.47	0.45
13/05/86	498945	19487	20.4	0.46	0.73
14/05/86	307479	19769	21.3	0.40	0.52
15/05/86	251637	14463	22.1	0.44	0.28
16/05/86	335847	11189	22.6	0.46	0.32
17/05/86	326645	13073	23.0	0.49	0.32
26/05/86	409772	22566	21.3	0.17	0.19
27/05/86	312650	17385	22.5	0.44	0.31
28/05/86	434188	18020	22.6	0.43	0.44
29/05/86	289021	13210	22.6	0.52	0.39
30/05/86	496851	15261	22.6	0.47	0.41
31/05/86	727238	19794	23.1	0.41	0.50

there is a time lag between the heat exchange from the pipes and the temperature variation within the thermal mass, due to the soil thermal diffusivity.

For example during heat recovery, the soil temperature in the bottom layer can continue to increase even if the soil temperature in the top layer has started to decrease.

A piezometer was installed in the greenhouse in order to monitor a potential water table inside the greenhouse thermal mass, that could affect the system performance. Regular measurements have indicated that the water table remained below the greenhouse footing and therefore, did not affect the thermal mass.

5.5.2 Dehumidification-humidification of the air

As shown in Figure 5.14, at night, the air relative humidity inside the greenhouse remains almost constant at approximately 80 %. In the morning, when the thermal curtain is opened, the relative humidity level increases temporarily, as the air temperature increases, the air relative humidity decreases gradually.

The air relative humidity at the exchanger inlet follows a pattern similar to the air relative humidity inside the greenhouse. However, the relative humidity at the inlet is lower than that of the air inside the greenhouse, since the air temperature at the exchanger inlet is higher than the one inside the greenhouse.

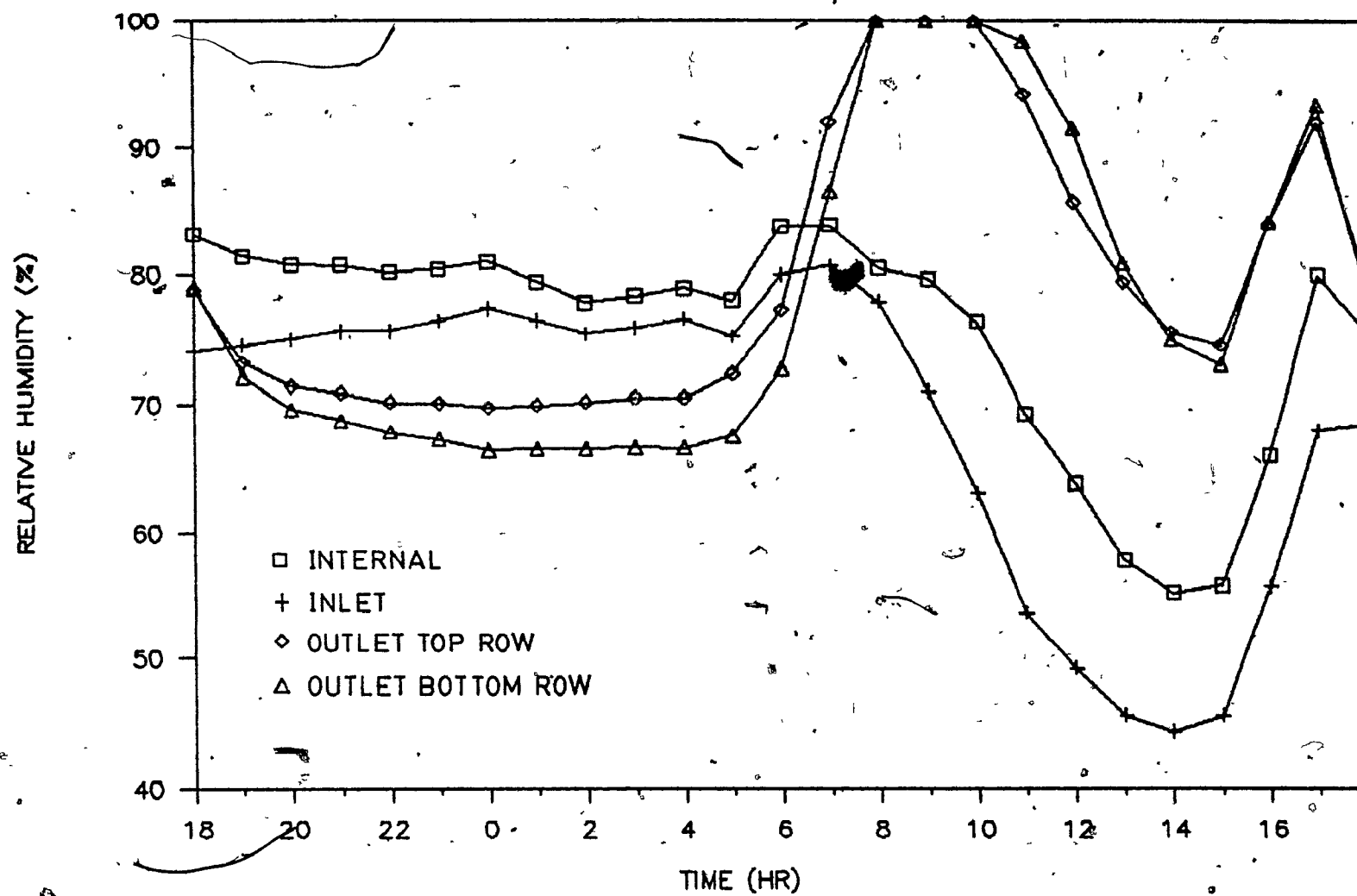


Figure 5.14 Air relative humidity daily variation.

At night, the air relative humidity at the exchanger outlet is lower than that of the inlet, since air is warmed as it gets through the exchanger; nevertheless, the relative humidity is almost constant. During the day, a pattern similar to the one obtained for the air inside the greenhouse is observed. The air relative humidity is higher at the outlet than at the inlet, since air is cooled as it gets through the exchanger. In the morning, saturation and condensation occur as long as the air relative humidity inside the greenhouse stays high. This is likely to happen when the thermal curtain opens; an important mass of saturated air is then introduced in the greenhouse and in the heat exchanger. As the air temperature increases in the greenhouse, the air relative humidity gradually decreases.

Visual inspections confirmed that the system dehumidifying effect is not negligible, mostly with the lower pipes located near the perimeter of the greenhouse. During spring and fall, important amounts of condensate were drained in the soil via the plenum floor. However, these amounts were not measured.

5.5.3 Heat Exchanger

A typical daily variation of the coefficient of performance (COP) computed from equations 3.34 and 3.35, is shown in Figure 5.15. At night, during heat recovery, the average COP is approximately 4.0; however, at sunset and sunrise, the COP decreases below 1.0, and the system becomes inefficient.

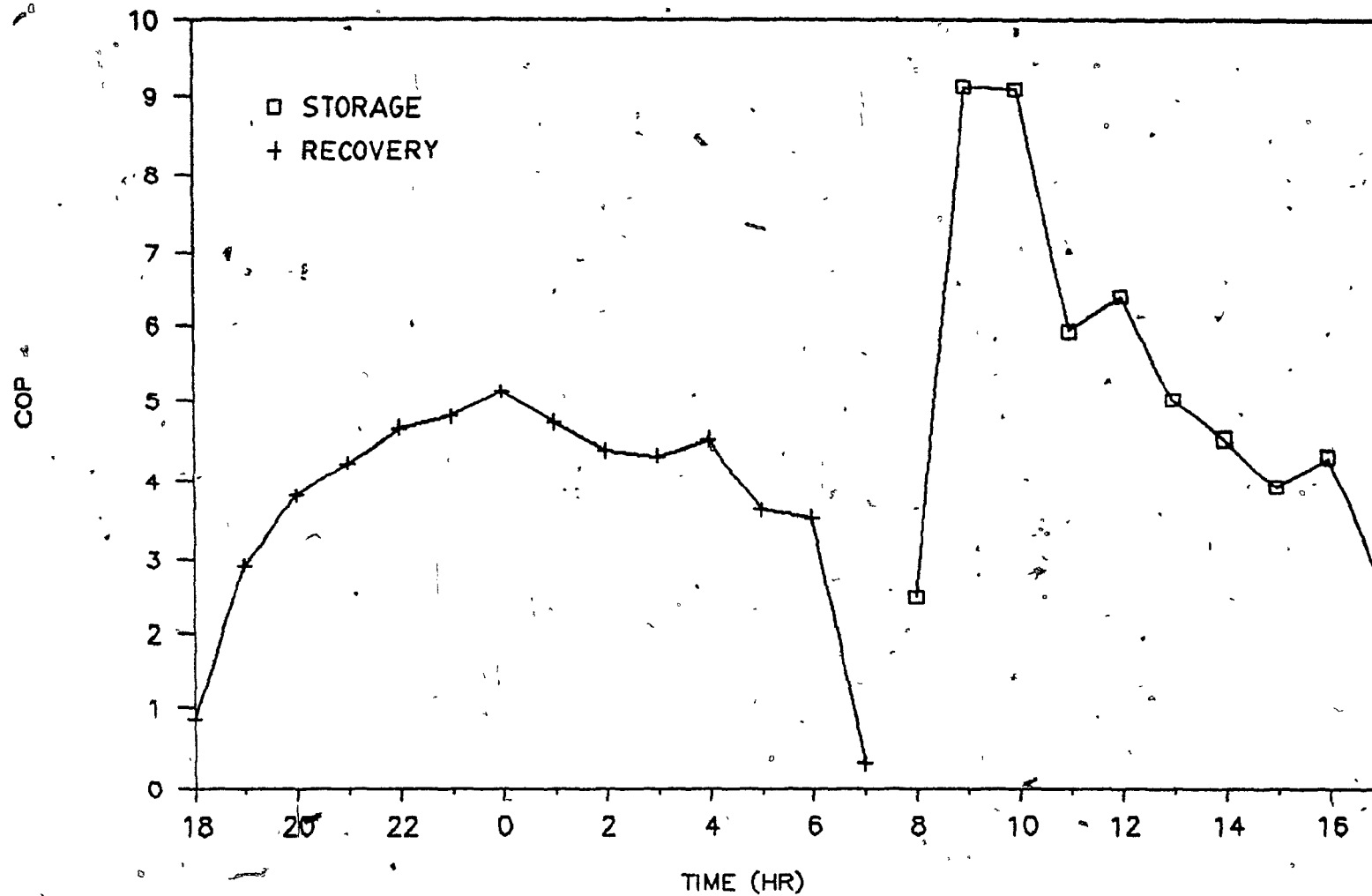


Figure 5.15 Daily variation of the coefficient of performance.

During daytime, the COP gradually decreases as more heat is stored in the thermal mass and somewhat follows a pattern of variation similar to the air temperature differential between the heat exchanger inlet and outlet.

The system operation was controlled by the temperature differential between the air in the upper part of the greenhouse near the heat exchanger inlet, and the soil. The higher the absolute differential, the higher the COP (Figure 5.16). A temperature differential above 6°C seems to be adequate for an efficient operation of the system.

The COP seems to be more influenced by the temperature differential between the air in the lower part of the greenhouse (below the thermal curtain position) and the soil (Figure 5.17), than by the temperature differential between the air in the upper part (above the thermal curtain position) and the soil, the coefficients of correlation being higher for the first case (Table 5.10).

When the thermal curtain is in operation, the system has its air intake below the curtain. This can explain the fact that a better correlation is obtained between the COP and the temperature differential between the air in the lower part of the greenhouse and the soil, for heat recovery. During heat storage, it seems that air from both the upper and lower part of the greenhouse west end, are mixed and circulated through the exchanger.

It is likely that a higher COP would be obtained if the air intake would be spread along the horizontal axis of the

Table 5.10 Effect of the temperature differential on the COP and on the heat exchanger efficiency

Parameter	Mode	<u>Coefficient of correlation</u> <u>Temperature differentials</u>	
		Upper Part ¹	Lower Part ²
COP	Storage	0.60	0.86
COP	Recovery	0.92	0.95
Efficiency	Storage	0.97	0.64
Efficiency	Recovery	0.95	0.99

¹ between the air above the thermal curtain and the soil

² between the air under the thermal curtain and the soil

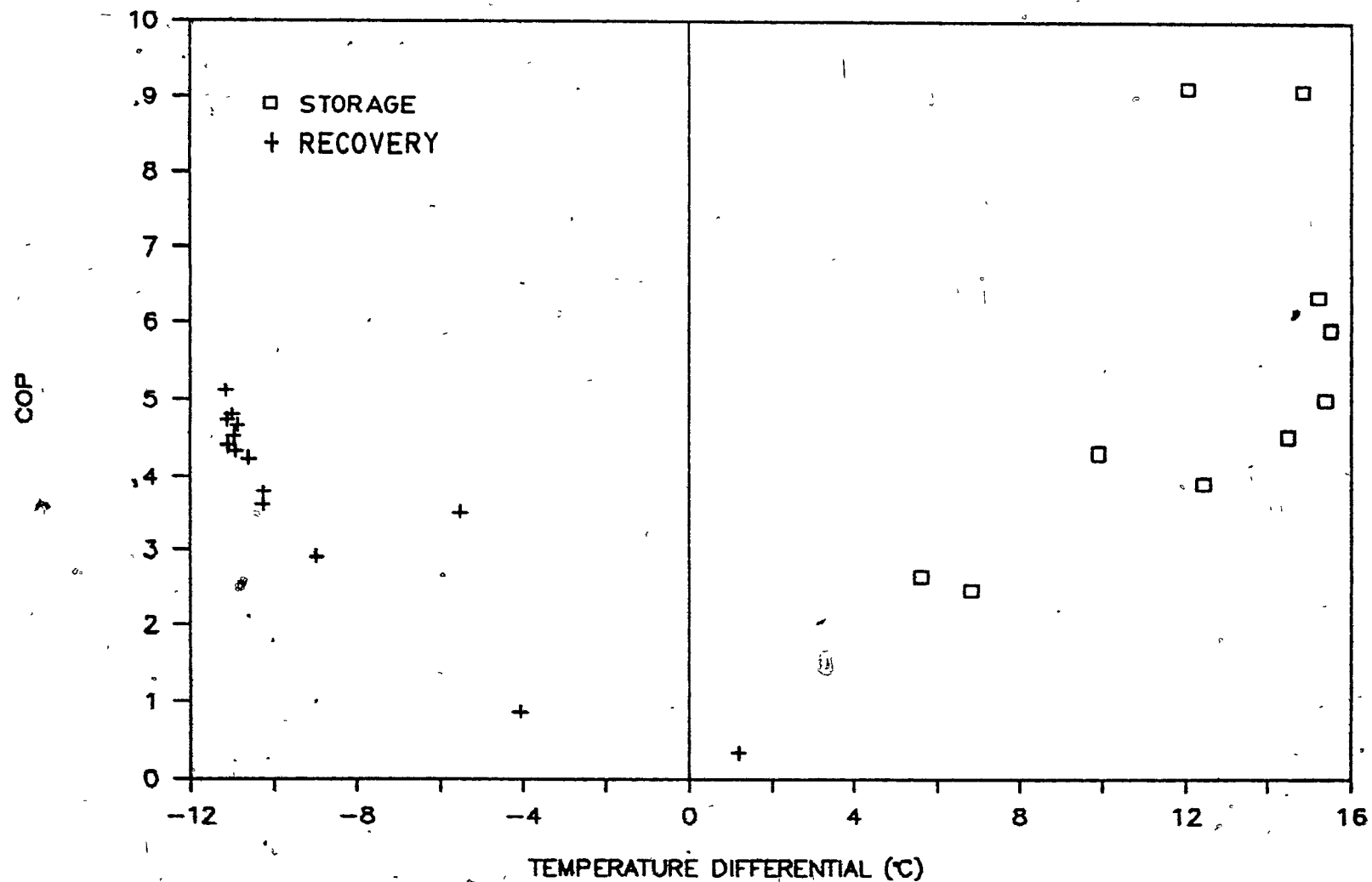


Figure 5.16 Effect of the temperature differential between the air at the blower inlet and the soil, on the COP.

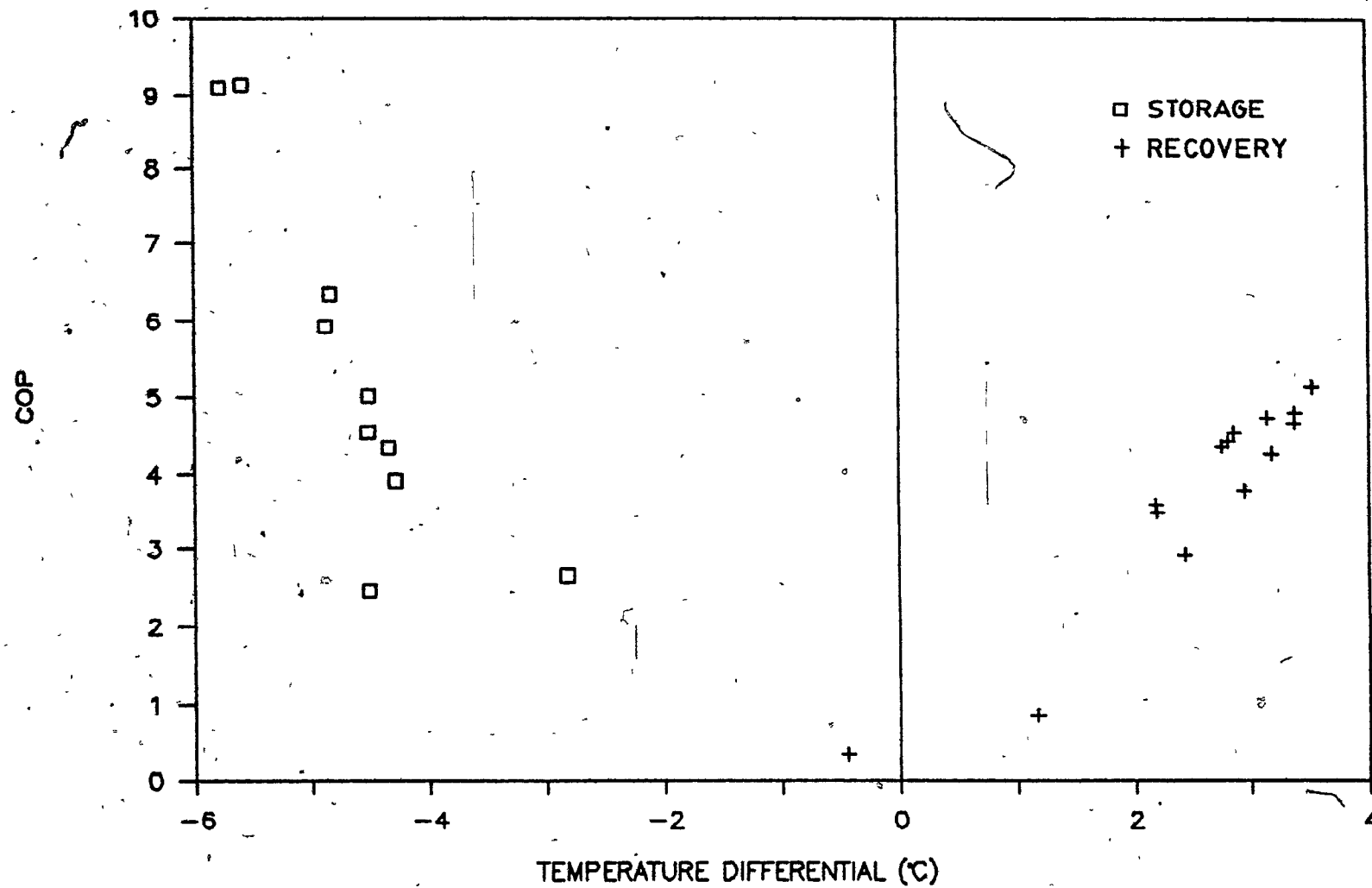


Figure 5.17 Effect of the temperature differential between the soil and the greenhouse air, on the COP.

greenhouse near the roof ridge. This would also result in a better correlation between the COP and the temperature differential between the air above the curtain and the soil.

Observations similar to those stated for COP, also apply to the heat exchanger efficiency computed from equations 3.32 and 3.33 (Figures 5.18 and 5.19). However, for heat storage, the correlation between the efficiency and temperature differential between the air in the upper part of the greenhouse and the soil, is higher than the one obtained between the efficiency and the temperature differential between the air in the lower part and the soil. The higher the temperature in the upper part, the higher is the temperature at the exchanger inlet; since the exchanger efficiency is computed (equation 3.33) from parameters such as the inlet temperature, the correlation is likely to be high between the efficiency and the temperature differential between the air from the upper part and the soil. For the same reason, during heat recovery, the correlation between the efficiency and the temperature differential between the air from the lower part and the soil, is likely to be high.

Figure 5.20 shows the ratio of the amount of energy recovered at the soil surface, over the total amount of energy recovered from both the soil surface and the heat exchanger, during the spring and fall of 1986 test periods.

From February to mid-April, the exchanger was operated during the day only, in order to rapidly increase the soil temperature for crop benefit. From mid-April to mid-June,

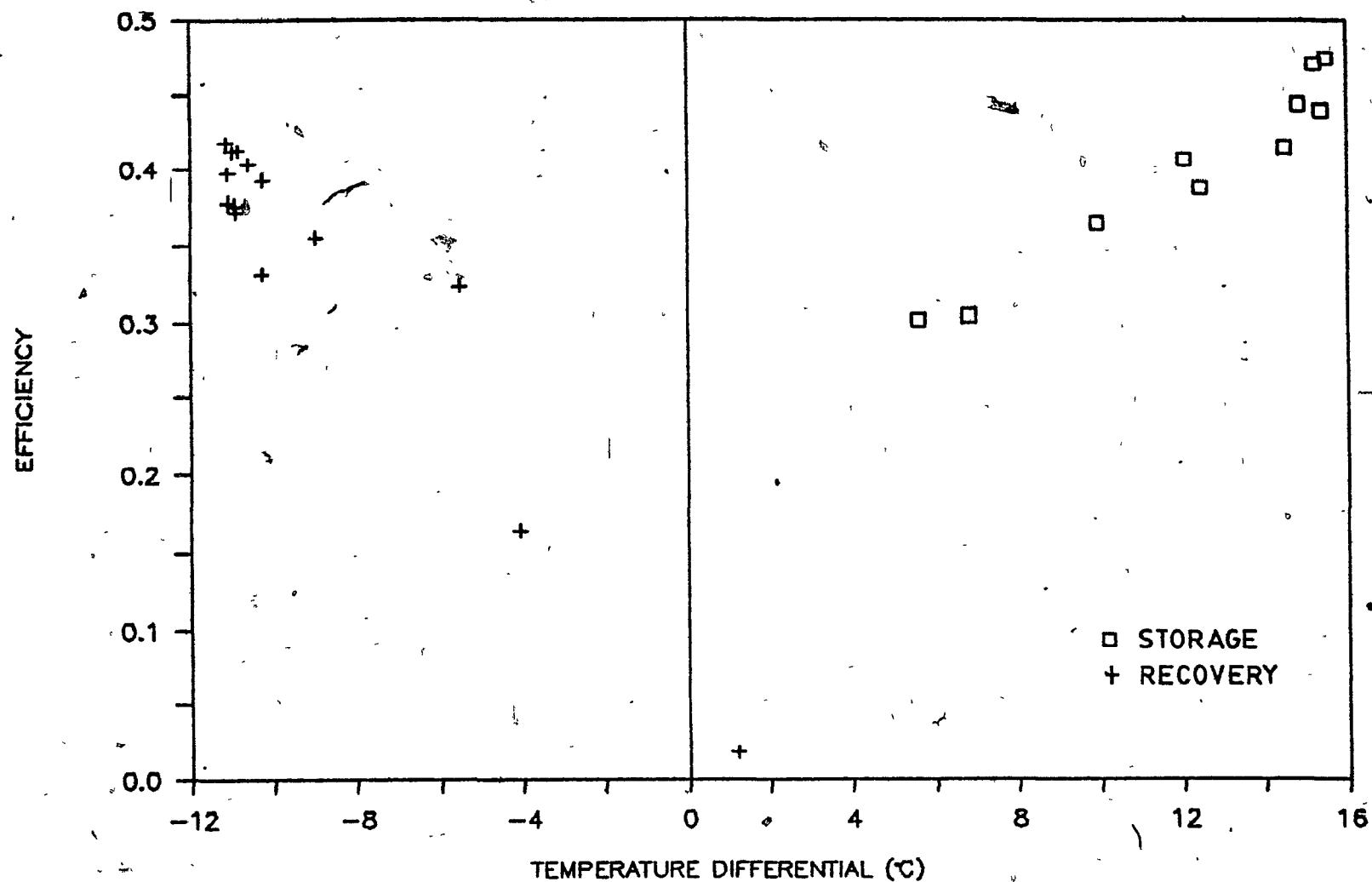


Figure 5.18 Effect of the temperature differential between the air at the blower inlet and the soil, on the efficiency.

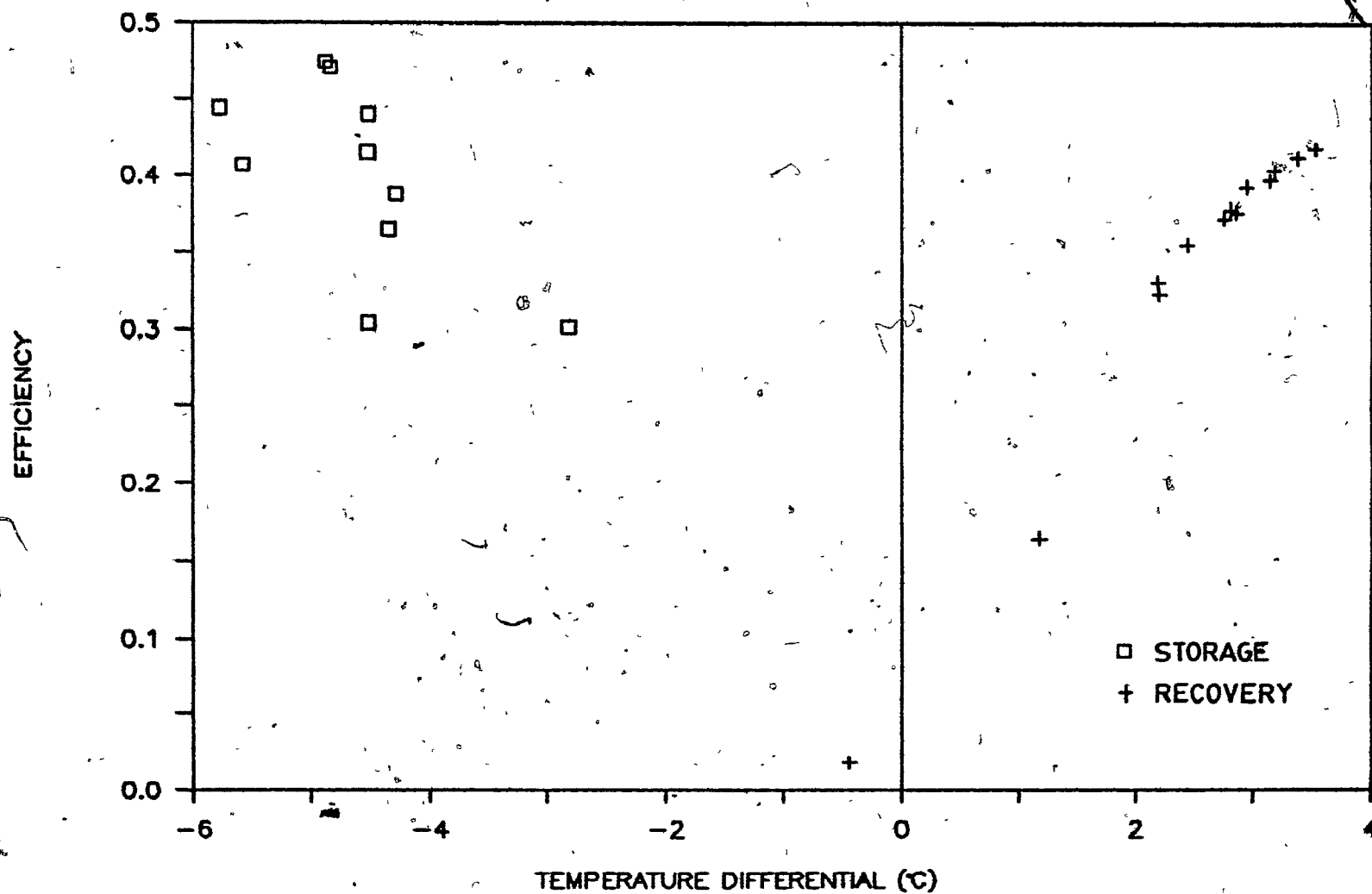


Figure 5.19 Effect of the temperature differential between the soil and the greenhouse air on the efficiency.

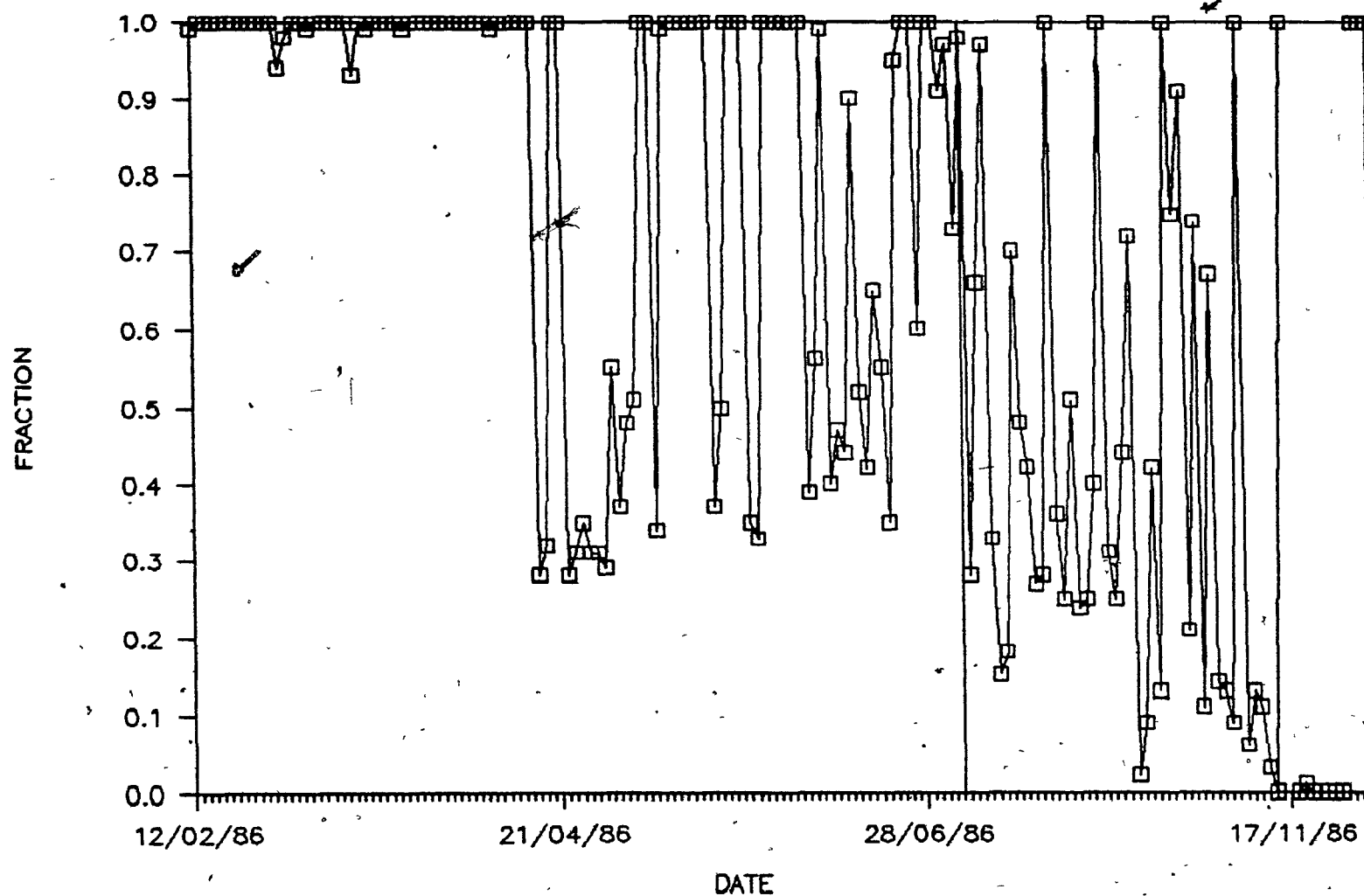


Figure 5.20 Ratio of the heat exchanger efficiency for the periods of February through July 1986 and September through November 1986.

the exchanger was operated day and night whenever the temperature differential between the air and the soil was sufficiently high. For that period, approximately 40 % of the heat was recovered from the soil surface. This contribution went from approximately 30 % in April to 50 % in June. During the Fall, the soil being colder, the contribution was estimated at 20 %, going from 50 % in September to 10 % in November.

Therefore, passive recovery at the soil surface seems to be substantial, provided that the convective heat transfer coefficient at the soil surface used, is a good estimate.

The heat exchanger efficiency is maximum in winter and in late fall for heat storage, since the thermal mass is relatively cool. As the temperature in the thermal mass increase, the efficiency decreases. On the contrary, for heat recovery, the efficiency becomes maximum in summer, since the soil is warmer (Figure 5.21).

5.5.4 Greenhouse Energy Consumption

The solar energy contribution to the greenhouse heat load computed from equation 3.1, does not seem to be as much influenced by solar radiation (Figure 5.22) as initially expected. However, as solar radiation increases, the exchanger cannot extract all of the surplus heat that is available and therefore, the ventilation is activated to evacuate some of the heat, maintaining that way an acceptable temperature for the plants.

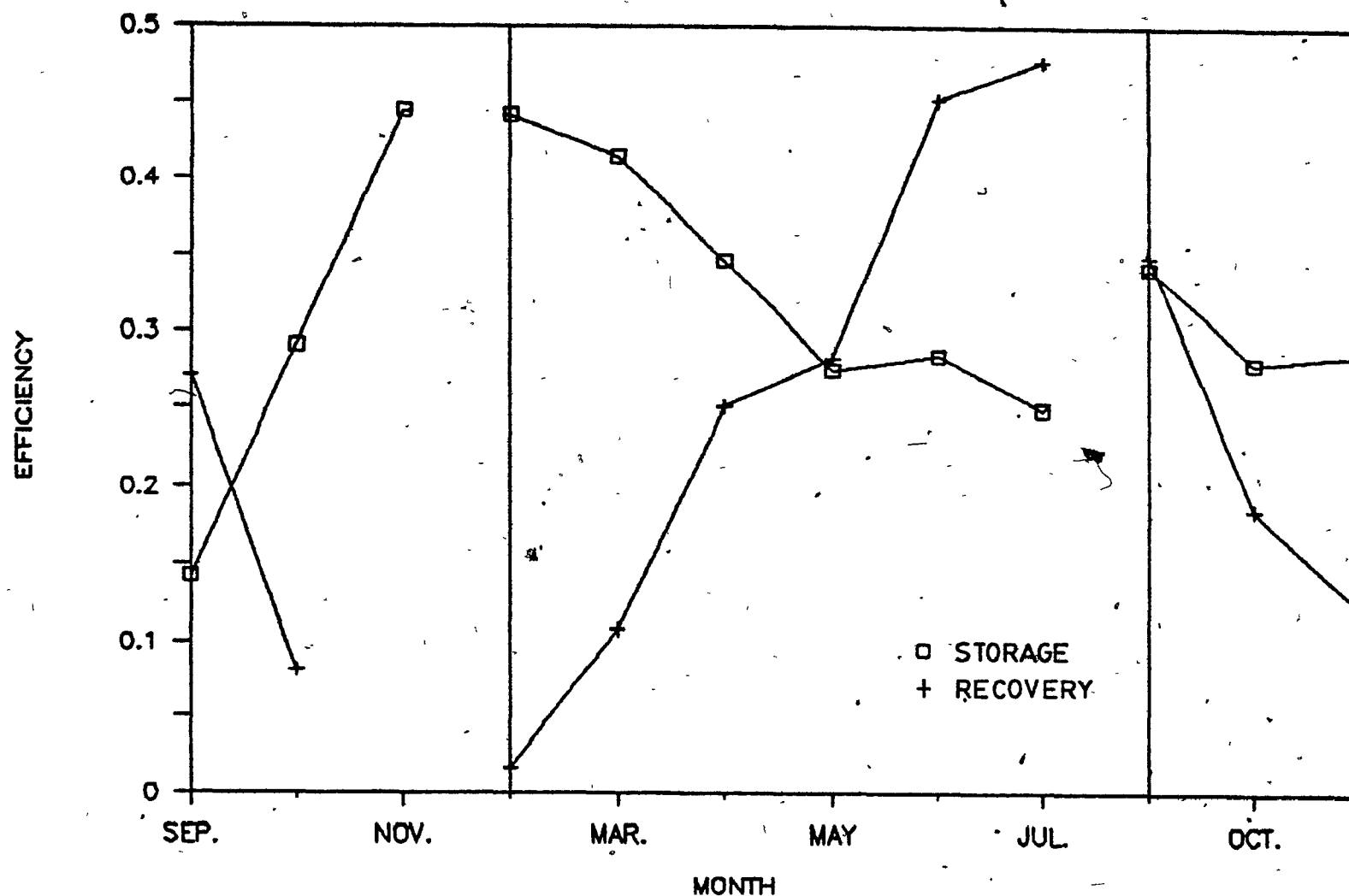


Figure 5.21 Variation of the heat exchanger efficiency for the periods of September through November 1985 and 1986, and February through July 1986.

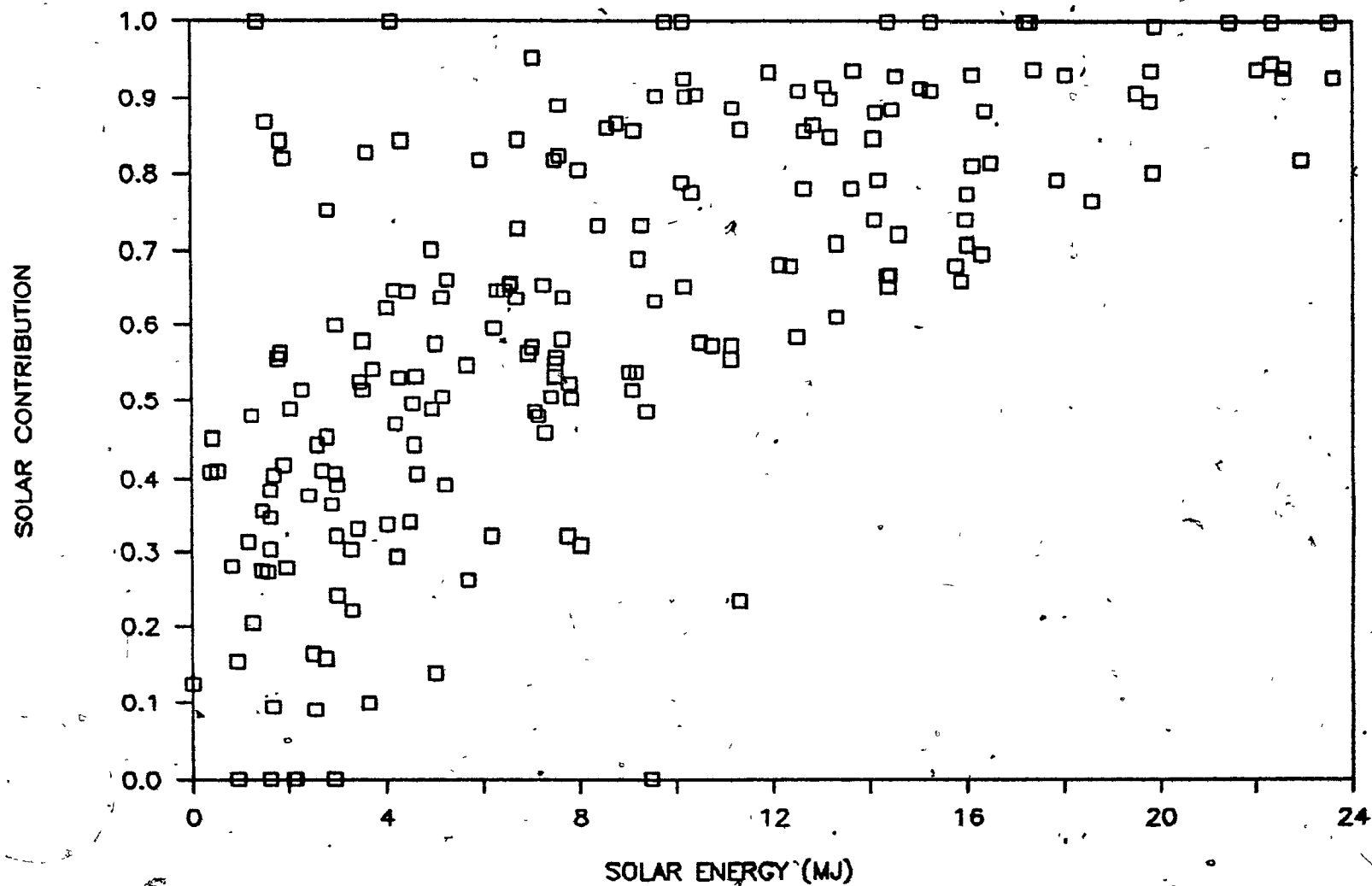


Figure 5.22 Effect of solar radiation on the solar contribution to the greenhouse heat load.

The same comment also applies for the air temperature in the lower part of the greenhouse, which follows a pattern of variation similar to solar radiation up to 21°C to 23°C, corresponding to the activation of the first ventilation stage (Figure 5.23).

There is a closer relationship between the average soil temperature and the solar contribution to the greenhouse heat load. Since the soil acts as a thermal mass, the warmer the soil, the more heat that can be recovered at night, resulting into more energy conservation on heating (Figure 5.24).

It appears that almost a 100 % of the greenhouse heat load can be provided by solar energy when the temperature of the thermal mass gets above 21°C. However, above 21°C, the stored heat is not used quite efficiently. For example, the temperature of the air inside the greenhouse at night has a tendency to remain higher than the minimum level recommended for a given crop.

Figure 5.25 shows that from March to July. The solar contribution went from 50 % to 100 %. One should note that the conventional greenhouses located in the La Pocatière region require auxiliary heat at night, even during summer. During the following fall season, the contribution decreased from 90 % to 30 % by the end of November 86.

From February to June and from September to November 1986, the overall solar contribution was estimated at 58 %; on the other hand, the contribution of the artificial

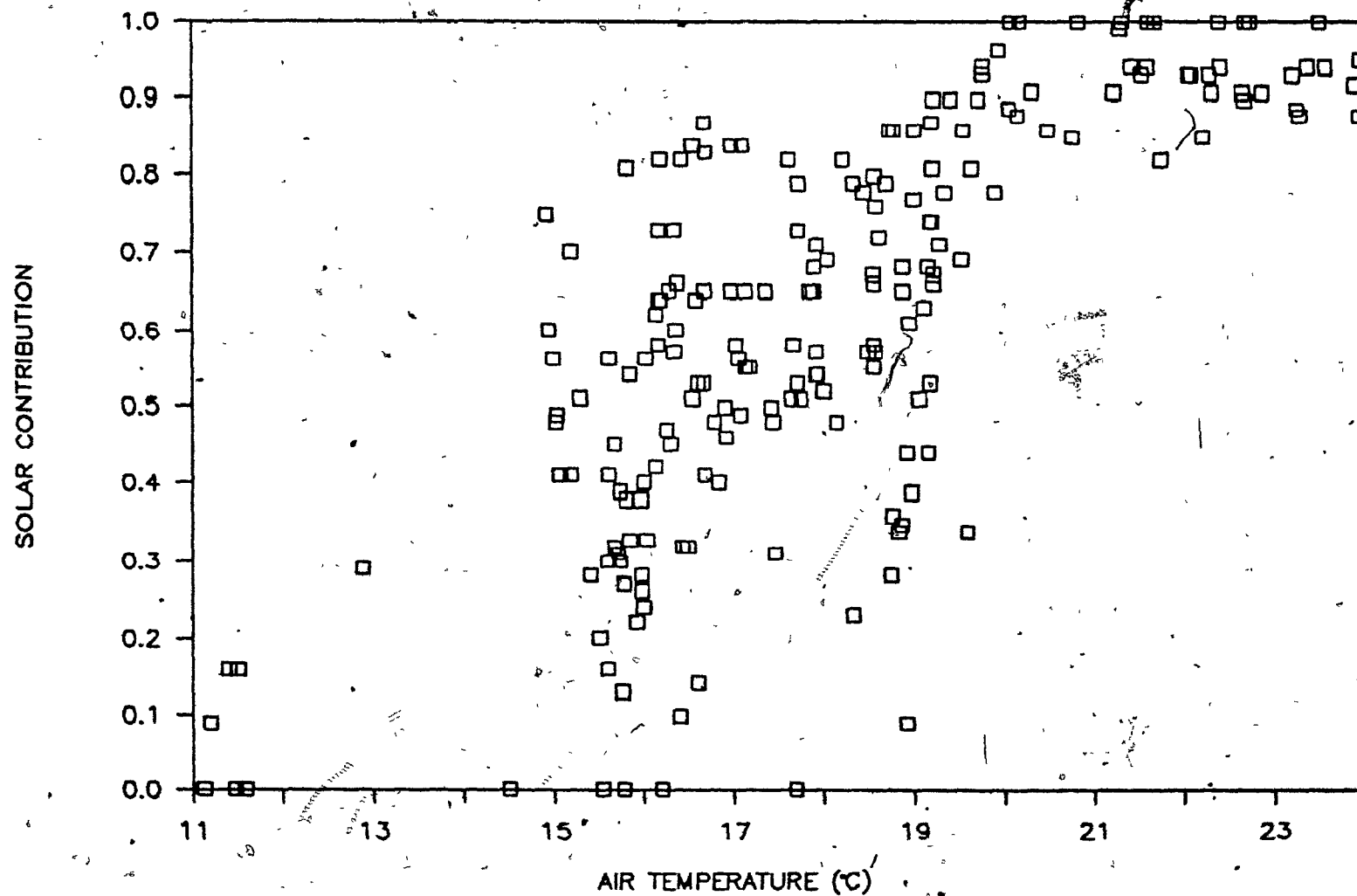


Figure 5.23 Effect of the greenhouse air temperature on the solar contribution to the greenhouse heat load.

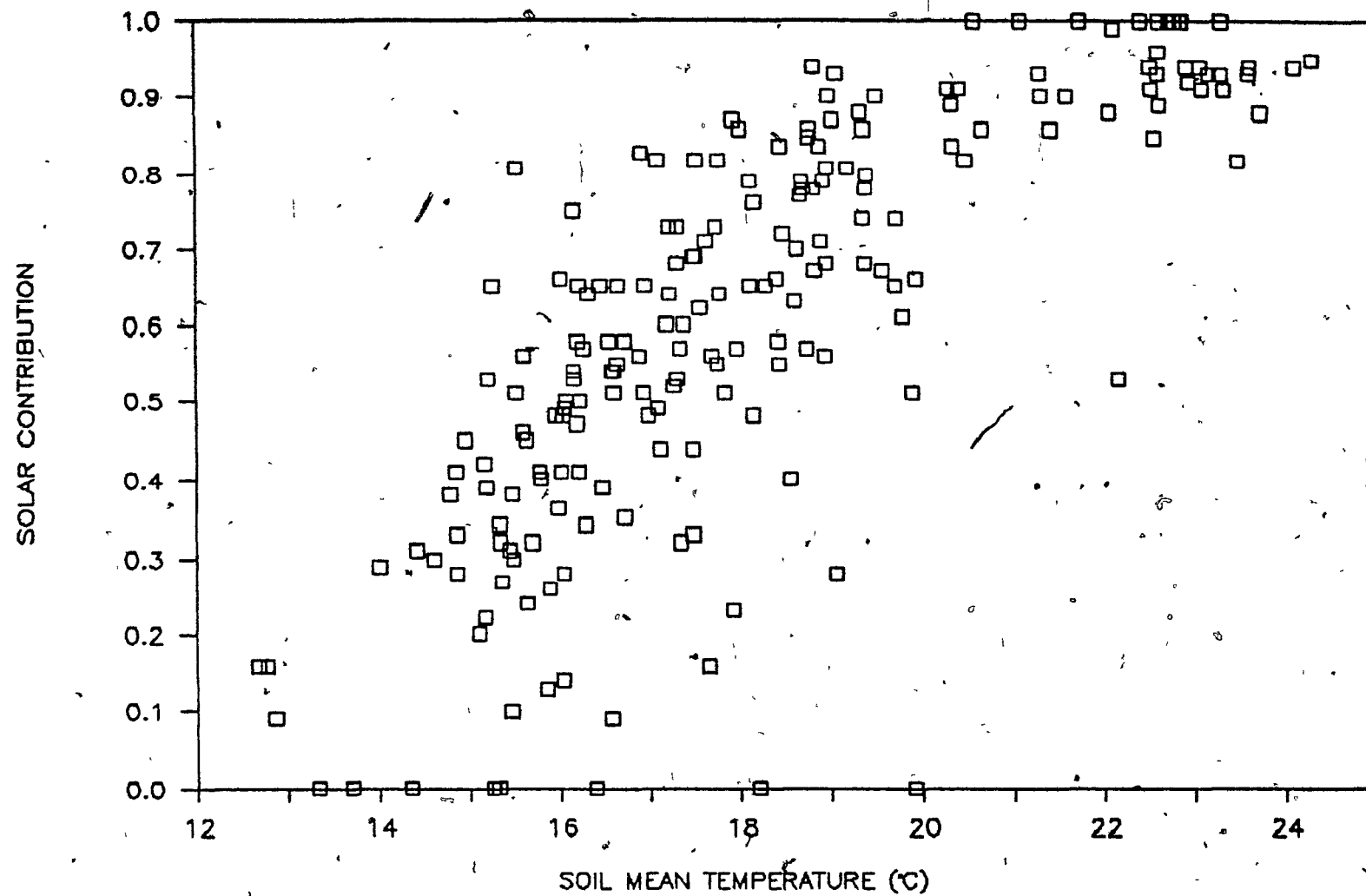


Figure 5.24 Effect of the soil temperature on the solar contribution to the greenhouse heat load.

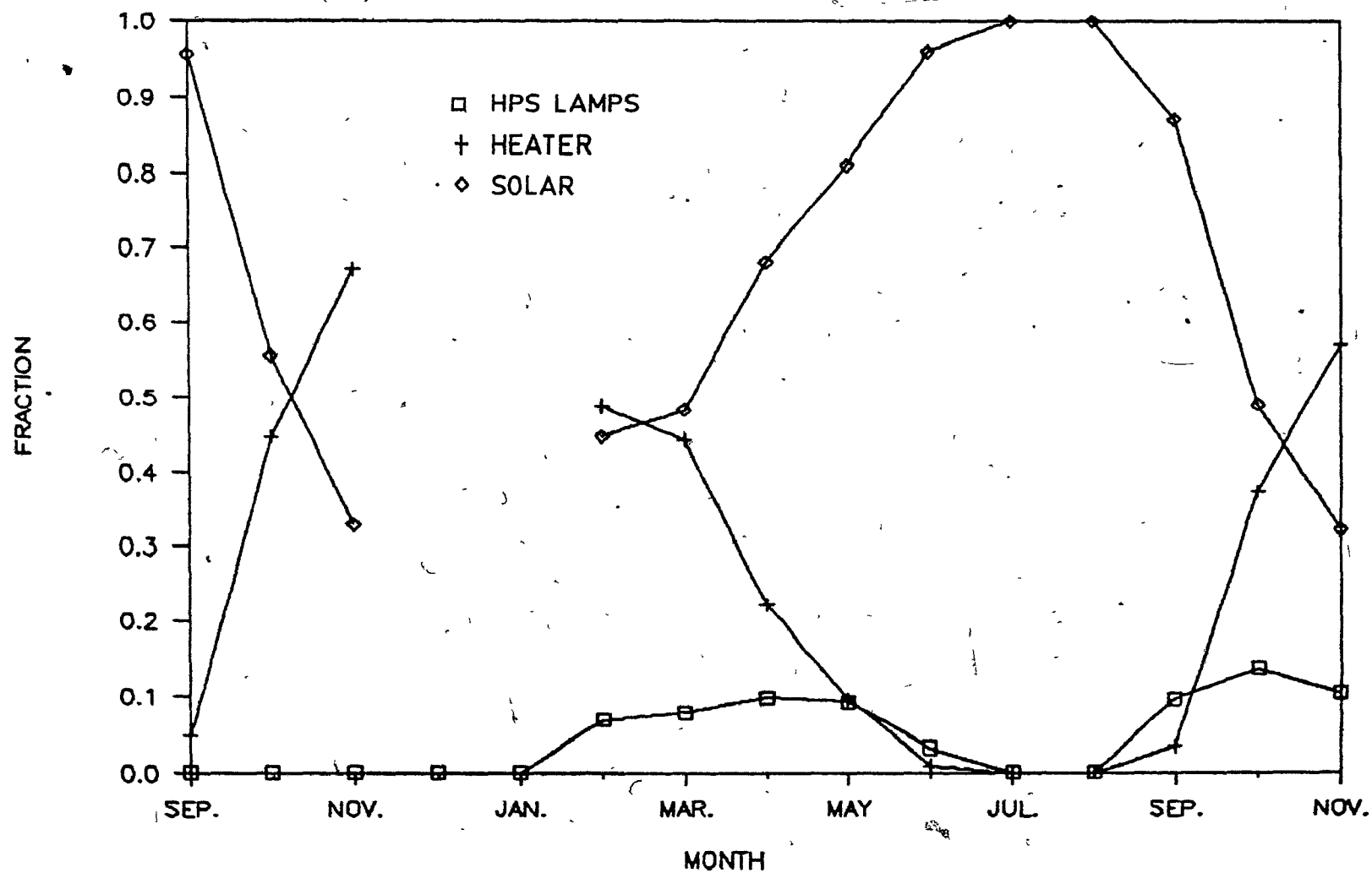


Figure 5.25 Variation in the contributions to the greenhouse heat load.

lighting system (equation 3.9) and the contribution of the heater (equation 3.5) were 7 % and 35 % respectively. Knowing that a conventional greenhouse gets 20 % to 25 % of its heating requirement from solar energy, the system might have resulted in an energy conservation of over 33 %.

One should note that the resulting performance of the system might not be representative of the full potential of the system, since it was not operated based on optimum conditions, as they are not known.

5.6 Tomato Crop

The tomato crops (Figure 5.26) were grown during the test period. The average yields were 7.4 kg/m^2 during the fall of 85 and 86, and 17.9 kg/m^2 during the spring of 1986, resulting in an overall yield of 25.3 kg/m^2 . The crops were subjected to the conditions recommended by the C.P.V.Q. (1984), except for the spring nighttime temperature which was maintained 2°C lower than the minimum temperature recommended.

In Quebec, the average yields for a greenhouse grown tomato crop are approximately 5 and 10 kg/m^2 for the fall and the spring crop respectively, which accounts for a total of 15 kg/m^2 for both season.

However, even if standard conditions were imposed to the crop during the experiment, the higher yields obtained in the experimental greenhouse as compared with the provincial average, are not necessarily due to the soil heat



Figure 5.26 Tomato crop grown in the experimental greenhouse.

exchanger-storage system. Favorable climatic conditions, the use of a high light transmission glazing, the use of a microcomputer to control the greenhouse climate and the use of four HPS lamps might also have contributed to the higher yields. Therefore, the effect of the system on the crop yields cannot be assessed.

However, the general appearance of the plants and the stem and leaf dimensions, seem to indicate that the soil stored heat had a positive impact on the crops, but again, this observation has not been verified experimentally.

5.7 Payback period

The payback period estimates were computed according to different scenarios based on a commercial type greenhouse operation, as described in section 4.2.8. These estimates are presented in Table 5.11.

It can be seen that the payback period is strongly influenced by the capital cost involved and the system impact on plant productivity. The system becomes less cost effective when the capital cost involves the installation of the system and when the system does not contribute to enhance plant productivity, as it would be for plants grown on benches for example. On the other hand, a greenhouse owner, installing himself the system could get a payback period of approximately two years, based on energy conservation alone; further, the payback period could get below twelve months if

the system enhances plant yields, as it appears to do for a soil grown tomato crop.

Table 5.11 Payback period estimation

Scenerio #	Auxiliary Heat Source	Initial Capital Cost (\$)	Operation Cost (\$)	Higher Plant Productivity Considered	Payback Period (years)
1	Heating oil	1593	0	No	2.2
2	Heating oil	3433	0	No	4.6
3	Heating oil	1593	222	No	2.6
4	Heating oil	3433	222	No	5.5
5	Heating oil	3433	222	Yes	0.9
6	Natural gas	3433	222	Yes	0.9
7	Wood chips	3433	222	Yes	1.0

VI SUMMARY AND CONCLUSIONS

The use of soil as a thermal mass inside a greenhouse has been studied in many countries, and many Japanese greenhouse owners have installed soil heat exchanger storage systems in their greenhouses to lower their production costs. The technique seems to be cost effective in countries having a mild climate, but would the technique be interesting if implemented in colder climate?

The following conclusions can be drawn from the experiments conducted on such a system installed in a commercial type greenhouse located in La Pocatière, Québec.

During the test period, a seasonal temperature fluctuation of the thermal mass of approximately 10°C was recorded. By early summer, the average temperature was 24°C , and by late fall, it decreased down to approximately 15°C .

The temperature is relatively uniform throughout the thermal mass. Over a 10 m length, the average temperature differential between the west and the east end of the storage was 2°C .

An important fraction of the heat exchange takes place in the first five meters of the heat exchanger.

The average temperature of thermal mass correlates well to the average temperature of the air inside the greenhouse. Therefore, the system performance is limited by the maximum and minimum temperature conditions imposed by the crop. The effect of temperature being cumulative for a plant, the

amount of heat stored and recovered might be improved if higher and lower temperatures were allowed for daytime and nighttime respectively, as compared with those generally recommended; however, that approach would require more sophisticated means of temperature control (based on temperature integration), which are not readily available right now.

On a daily basis, the system efficiency parameters in terms of heat exchange, and temperature fluctuation in the thermal mass, did not turn out to be good indicators of the system performance.

From February to December 1986, the coefficient of performance (COP) of the system was 3.6. Therefore, the system used efficiently the electrical energy consumed for its own operation; however, the system becomes not efficient around sunset and sunrise and should not be operated for temperature differential between the air near the roof ridge and the soil, lower than 6°C . The system performance is better during heat storage than during heat recovery. The COP might probably increase provided that heat is collected along the full length of the greenhouse.

When the system is used for heat recovery, approximately 30 % of the heat is recovered passively at the soil surface. The passive recovery of heat seems to be the appropriate mode of heat recovery during fall operation. For heat recovery, the maximum efficiency was encountered during summer. The average convective heat transfer coefficient in the heat exchanger is $0.012 \text{ kW/m}^2\cdot\text{K}$, this value is similar to those

listed in the literature, but for lower air velocities than the one used.

The use of a plastic mulch over the soil surface would likely increase the system efficiency, by reducing the evaporation heat loss from the storage.

The heat exchanger has a dehumidifying effect on the air circulated in the pipes. Visual observations as well as results confirmed this conclusion. In order to efficiently dehumidify the air, a slope should be provided to the pipes, so that the condensate would be drained in the soil while flowing in the plenum. However, more reliable results have to be obtained to further assess this aspect.

The solar energy contribution to the greenhouse heat load seems to be more influenced by the average soil temperature than by the average temperature of the air inside the greenhouse or by the solar radiation entering the greenhouse. This contribution has been estimated to be approximately 100 % for a thermal mass temperature above 21°C.

For a typical operation (from February to June and from September to December 1986), the solar energy contribution to the greenhouse heat load was estimated at 58 % (excluding summer operation). For the same period, the contribution of the artificial lighting system and of the heater were 7 % and 35 % respectively. Knowing that solar energy normally contributes from 20 % to 25 % of a greenhouse total heat load, the system provided a 33 % energy conservation.

More accurate estimates would require the comparison of energy consumptions between a control greenhouse and a similar greenhouse equipped with the soil heat exchanger-storage system, over a few years of operation.

From an agronomic point of view, the overall yield for the soil grown tomato crops has been 25.3 kg/m^2 . According to the C.P.V.Q. (1984), the average yield in Québec for greenhouse produced tomatoes is 15 kg/m^2 . However, the difference in yields, mostly observed on the spring crop, could be attributable in part to causes other than the high soil temperature related to heat storage. The effect of the soil heat exchanger storage system on the crop could not be measured, since no control greenhouse was available. However, qualitative observations made on the crop seems to indicate that the soil stored heat was beneficial to the plants.

A rough economic calculation indicates that for a commercial greenhouse heated with fuel oil, in which tomatoes are produced, the payback period for such a system would be two years, if the system is built by the greenhouse owner. Furthermore, if some increase in the yield due to the system is accounted for, the payback period decreases down to approximately one year. However, a complete economic analysis should be done in order to verify that aspect.

It seems that this new energy conservation technique could be as easily implemented in existing greenhouses as in newly constructed ones. In addition to the energy savings obtained by reducing the nighttime temperature and by recove-

ring some of the stored heat, savings are also obtained on the electrical energy used for ventilation, since the air evacuation needs are reduced. It should also be noted that the experimental greenhouse was not equipped with a fan jet, since the heat exchanger blower was anticipated to play a similar role.

Such a system, might be effectively used in combination with an artificial lighting system. The heat released by it would delay the need to recover the stored heat, and extend the usefulness of the thermal mass. Furthermore, some of the excess heat released by the lighting system during the day, could also be stored. Such a storage system might also be used effectively in combination with a CO₂ enrichment system, since air evacuation needs are reduced. Burning gas enrichment systems also released a significant quantity of excess heat that could be stored for later use.

The soil heat exchanger-storage system seems to be appropriate for greenhouses located in cold climate, and could be adapted for different greenhouse production schemes, in order that soilless crop root zone could also benefit from the stored heat.

RECOMMENDATIONS

Because of the limitations of the present study, further investigations are required in order to assess all the impacts of this new technique. For example:

a study on the dehumidification effect of the system could be performed on a laboratory setup, in order to quantify the sensible heat transfer and the amount of condensate generated for different air temperature and humidity conditions;

a comparative study between a control greenhouse and a greenhouse equipped with the system could be done, in order to evaluate the overall impacts on energy consumption and on crop yield, for different production schemes;

an economical study could be conducted, in order to determine the conditions that would guarantee the cost effectiveness of the system;

a mathematical model could be developed in order to simulate the thermal behavior of the system for different operating conditions;

comparison tests could be performed with and without a plastic mulch on the soil surface, in order to find out which approach is best;

a study could be done in order to evaluate the impacts of a soil heat exchanger storage system in combination with an artificial lighting system and/or with a CO₂ enrichment system.

REFERENCES

- Agriculture Canada, 1979. Réduction des dépenses d'énergie pour la serriculture. Communiqué de presse daté du 27 avril.
- Arcus Consulting Ltd. 1985. An economic analysis of the new greenhouse technologies, Report presented to Agriculture Canada, New-Westminster, B.C. 101 p.
- A.S.H.R.A.E. 1968. Applications. American Society of Heating Refrigerating and Air-conditioning engineers Inc. New York, N.Y.
- Audet, R. et J. Paris. 1985. Evaluation d'un nouveau système de gestion de la chaleur dans une serre. Rapport soumis au Ministère de l'Agriculture, des Pêcheries et de l'Alimentation du Québec. 30 p.
- Baird, C.D. and W.E. Waters, Solar energy and greenhouse heating. 1979. Hort Science, vol. 14 (2): 147-151.
- Baver, C.D. 1940. Soil physics. John Wiley and Sons Inc. London. 432 p.
- Caffel A., and K.T. Mackay. 1981. Mud storage: A new concept in greenhouse heat storage. 1981. Energy conservation and use of renewable energies in the bio-industries. Pergamon Press, Oxford. pp. 75-83.
- Candura, A. et A. Gusman. 1977. Studio per una migliore utilizzazione dell' energia solare nelle serre in vetro. Culture Protette. 6 (10): 23-40.
- Chapman, A.J. 1976. Heat transfer. Macmillan Company, London. 617 p.
- Cormany, Y. et C. Nicolas. 1985. La thermique des serres. Editions Eyrolles, Paris. 357 p.
- C.P.V.Q. 1984. Légumes de serres. Ministère de l'Agriculture, des Pêcheries et de l'Alimentation. Québec. 156 p.
- Dale, A.C. and P.A. Hammer, 1983. Performance of an energy conserving greenhouse with solar heated plant-root-soil. ASAE paper no. 83-4021. ASAE, St. Joseph, MI.
- Dwyer Instruments Inc. 1984. Air velocities with the Dwyer Pitot Tube. Bulletin no. H-11. Michigan City, Indiana. 1984.
- Eckert, E.R. 1976. The ground used as energy source, energy sink or for energy storage. Solar energy digest, vol. I, p. 315-323.

Eckoff, S.R. and M.R. Okos. 1980. Thermal storage comparison and design for rock, saturated soil and sodium sulfate decahydrate for agricultural applications. Transactions of ASAE. 23(3):722-728.

French, J. 1987. Communication personnelle. Agriculture Canada, Vancouver, B.C.

Givoni, B. 1977. Underground longterm storage of solar energy - an overview. Solar Energy, vol. 19, pp. 617-623.

Gosselin, A. et J.M. Turdel. 1983. Utilisation du chauffage du sol dans les cultures de légumes de serres. Québec Vert, avril p. 12-17.

Ingratta, F. 1978. Major research on solar-heated greenhouses launched at HRIO. News release. Ontario Ministry of Agriculture and Food. October 9.

Ingratta, F. 1979. Solar research brightens greenhouse future. News release. Ontario Ministry of Agriculture and Food. June 4.

Kreith, R. and J.F. Kreider. 1978. Principles of solar engineering. McGraw-Hill Book Company. 778 p.

Lawand, T.A., R. Alward, M. Papadopolis and W. Coffin. 1983. Guide to Soil as an Underground Heat Storage Medium. Phase I. Final Report. Engineering and Statistical Research Institute. Agriculture Canada, Ottawa, Ont., 145 p.

Lawand, T.A., R. Alward, E. Saulnier, E. Brunet et J. L. Mussier. 1974. Le développement d'une serre spécialement adaptée à l'environnement des régions froides. Communication présentée à la rencontre annuelle de la Société Canadienne de Génie rural. Université Laval. 4-8 août. Ste-Foy, Québec.

Lawand, T.A., W. Coffin, R. Alward, R. Chagnon. 1985. Design guide for underground heat storage. Technical Bulletin no. 21. Agriculture Canada, St-Jean-sur-Richelieu, Québec, 75 p.

Maes, R.E. 1978. A Northern climate solar garden. American Vegetable Grower, 26 (6): 18.

Maghsood, J. 1976. A study of solar energy parameters in plastic-covered greenhouses. J. Agric. Eng Res. 21: 305-312.

Midwest Plan Service. 1984. Earth Tube heat exchange systems. Agricultural Engineers' Digest AED-25. Iowa State University, Ames, Iowa. 8 p.

Milburn, W.F. and R.A. Aldrich. 1979. Optimization of excess internal heat collection and thermal storage in a greenhouse. ASAE paper no. 79-4026. ASAE, St. Joseph, MI.

Monk, G.J., L.M. Staley and J.M. Molnar. 1983. Design, Construction and Performance of two Earth Thermal Storage Solar Greenhouses. CSAE Paper no. 83-405. CSAE, Ottawa, Ont.

Mori, T. 1978. Utilization of solar energy for winter cropping greenhouse tomato. *Acta Horticulturae*. 87: 321-327.

Munn, R.E. 1966. Descriptive Micrometeorology. Academic Press, New York. 245 p.

Mustacchi, C., V. Cena and M. Rocchi. 1981. Buried pipe seasonal heat storage. *Solar World Forum*, vol. 1, pp. 786-791. Brighton, England.

Paris, J. et G. Jacqmain, 1983. Développement et réalisation d'une serre solaire pour la production agricole commerciale dans les conditions climatiques canadiennes. Rapport d'étape préparé pour Agriculture Canada, Montréal, Québec. 88 p.

Perry, D.B. and J.L. Robertson. 1980. An economic evaluation of energy conservation investments for greenhouses. Research bulletin 1114. Ohio Agricultural Research and Development Center. Wooster, OH. 21 p.

Phillips, D.W. and Aston, 1980. Canadian solar radiation data. Environment Canada. Downsview. 108 p.

Portales, B., M. Le Ray, and R. Torguet. 1982. Experimentation of the heating of greenhouses with solar energy stored in subsoil in the north of France. *Proceedings*, Vol. 1. Energex' 82. Solar Energy Society of Canada, August 23-29, Regina, Sas. p. 381-384.

Portugais, J. et J. Paris. 1983. Simulation des transferts de chaleur dans le sol d'une serre solaire. Communication présentée à la 9 ième conférence annuelle de la Solar Energy Society of Canada Inc. Windsor, Ontario, août 1-5. pp. GR. 3.1 - GR. 3.6.

Puri, V.M. 1986. Feasibility and performance curves for intermittent earth tube heat exchangers. *Transactions of the ASAE*, 29 (2): 526-532.

Rabbinov, R.T., G.Y. Umarov and R.A. Zakhidov. 1971. Storage of solar energy in a sandy-gravel ground. *Geliotekhnica*, 7 (5): 57-64.

Richard, P., K. J. Sibley and G.S.V. Raghavan. 1981. Soil as a heat source/sink for energy conservation in a greenhouse. ASAE paper no. NAR 81-231. ASAE, St. Joseph, MI.

Roberts, W.J. and D.R. Mears 1977. Utilisation of solar energy for greenhouse heating. *Plasticulture* # 33. p. 29-36.

Rosenberg, N.J., B.L. Blod and S.B. Verma. 1983. *Microclimate the biological environment*. 2nd edition. John Wiley and Sons, New York. 495 p.

Sasaki, K. and T. Itagi. 1979. Studies on solar greenhouse heated by mean of an earth storage exchange system (I). *Bulletin of the Kanagawa Horticultural Experimental Station*, No. 26.

Sasaki, K., T. Itagi and M. Takahasi. 1980. Studies on the solar greenhouse heated by means of an earth storage heat exchange system (II). *Bulletin of the Kanagawa Horticultural Experiment Station*, No. 27.

Sasaki, K., T. Itagi and M. Takahasi. 1981. Studies on the solar greenhouse heated by means of an earth storage heat exchange system (III). *Bulletin in of the Kanagawa Horticultural Experiment Station*, no. 28.

Scott, N.R., R.A. Parsons and T.A. Koehler. 1965. Analysis and Performance of an earth-air heat exchanger. ASAE paper no. 65-840. ASAE, St. Joseph, MI.

Services de l'environnement atmosphérique. 1982. Normales climatiques au Canada, Insolation effective 1951-1980. Vol. 7. Environnement Canada, Ottawa.

Services de l'environnement atmosphérique. 1982. Normales climatiques au Canada, Températures 1951-1980. vol. 2. Environnement Canada, Ottawa, 306 p.

Sheard, G.F. 1978. Shelter and the effect of wind on the heat loss from greenhouses. *Acta Horticulturae*, 76: 357-360.

Shelton, J. 1975. Underground storage of heat in solar heating systems. *Solar Energy*, 17: 137-143.

Short, T.H., P.C. Badger, and W.L. Roller. 1976. OARDC's Solar-heated greenhouse. *Agricultural Engineering*. July. p. 30-32.

Sibley, K.J. and G.S.V. Raghavan. 1984. Heat transfer coefficient for air flow in plastic corrugated drainage tubes. *Canadian Agricultural Engineering*, 26 (2): 177-180.

Sinha, R.R., D.Y. Goswami and D.E. Klett. 1981. Theoretical and experimental analysis of cooling technique using underground pipe. *Solar World Forum*. Brighton, England. pp. 2105-2114.

Staley, L.M., J.M. Molnar and G.J. Monk 1982. First year performance of a shed-type solar greenhouse. Proceedings, Vol. 1. Energex 82. Solar Energy Society of Canada, August 23-29, University of Regina, Sas. p. 375-380.

Staley, L.M., G.J. Monk, D.A. Thomas and J.M. Molnar. 1983. Earth thermal heat exchange systems for solar greenhouses. ASAE paper # 83-4525. ASAE, St. Joseph, MI.

Takakura, T., H. Nishin and K. Kurata. 1982. A simulation analysis of solar greenhouses with underground heat storage units. Report from Laboratory of Environmental Engineering. Dept. of Agricultural Engineering, University of Tokyo, Japan.

Takakura, T. and K. Yamakawa. 1982. Performance and evaluation of a greenhouse solar collection with underground heat storage system. J. Agr. Met. 37 (3): 187-196.

Taylor, S.A. and R.D. Jackson, 1965. Heat capacity and specific heat. In Methods of soil analysis, part 1. C.A. Black (ed.). Agronomy 9: 345-348.

Umarov, G.Y., B. Khairitdinov, A.B. Vardiashvili, R. Avezov, A. Gafurov and S.A. Babayants. 1981. The results of tests of a greenhouse with solar energy storage combined with a fuel boiler plant. Geliotekhnika, 17(4):67-71.

Van Wijk, W.R. 1963. Physics of plant environment. John Wiley and Sons Inc. New York, N.Y. 536 p.

Vardiashvili A.B. et V.D. Kim. 1980. Hydraulic and thermal design of subsoil storage system for solar hothouses. Geliotekhnika, 16 (6): 48-53.

Walker, J.N. and J. W. Buxton. 1977. Can circulating air through a buried pipe be used to heat and cool greenhouses. J. Amer. Soc. Hort. Sci. 102 (2): 626-629.

Wilson, G.E., D.R. Price and R. W. Langhans. 1977. Increasing the effectiveness of greenhouse as a solar collector. ASAE paper # 77-4527. ASAE, St. Joseph, MI.

Yamamoto, Y. 1973. Temperature conditions and thermal characteristics in the earth-air heat exchange greenhouse. J. Agr. Met. 46 (2): 96-105.

APPENDICES

APPENDIX A

Linear regression for the estimation
of the average air temperature
at the heater outlets

The average temperature of the air from the heater outlets (TOC) can be predicted from the following empirical equation based on the center outlet temperature (TCE) measurement:

$$TOHC = 1.0101 + 0.9527 \cdot TCE \quad (5.1)$$

The Tables A.1, A.2 and A.3 show the data obtained from two test runs, the SAS program for computing the regression parameters and the summarized SAS regression analysis output.

Table A.1 Air temperature at the heater outlets

TNO ¹ (°C)	TCE ² (°C)	TSU ³ (°C)	TNO ¹ (°C)	TCE ² (°C)	TSU ³ (°C)
46.0	50.5	50.5	51.5	58.2	60.0
28.5	31.8	33.0	42.0	47.5	49.5
30.5	34.5	35.0	28.0	31.5	33.5
49.5	48.0	45.0	56.0	62.0	63.0
31.0	34.0	34.5	52.0	58.0	59.0
27.0	31.0	33.0	48.5	53.0	53.0
54.0	60.7	62.0	57.5	64.0	65.0
56.0	64.2	66.0	48.5	57.2	59.5
42.5	50.5	53.0	37.5	44.2	46.0
36.0	41.0	42.5	38.0	42.5	44.0
43.5	47.0	48.0	47.5	51.5	52.2
51.0	55.8	56.5	53.0	58.5	59.5
55.0	61.0	62.0	56.5	63.0	64.0
58.0	65.0	66.0	59.5	66.5	68.5
58.0	66.0	68.0	51.0	60.0	62.5
45.0	54.0	56.0	42.0	47.5	48.0
35.5	42.5	42.0	37.0	40.5	42.5
39.0	44.0	46.0	44.0	49.0	51.0
48.0	53.5	54.0	50.5	56.5	58.0
54.0	59.0	60.0	55.5	62.0	63.5
57.5	64.0	65.0	58.5	66.0	67.0
58.0	66.0	67.0	54.0	60.7	61.0
45.0	53.0	55.5	41.5	47.0	47.5
35.0	41.5	43.5	35.5	40.5	42.0
40.0	43.5	44.5	43.0	48.0	50.0
47.5	52.0	53.0	50.0	56.0	57.5
53.0	59.0	59.5	56.5	64.0	65.5
58.0	65.5	66.5	59.5	67.0	68.0
59.0	67.0	68.5	23.0	23.0	23.0
23.0	23.0	23.0	23.0	22.8	22.8
22.8	22.8	22.8	22.5	22.5	22.5
23.0	23.5	23.5	23.0	23.5	23.5
23.0	23.2	23.2	23.0	23.2	23.2
23.0	23.2	28.2			

1 outlet located along the north wall

2 outlet located on the greenhouse center line

3 outlet located along the south wall

Table A.2 SAS linear regression program for the estimation of the heater outlet mean air temperature

```

DATA TEMP;
INPUT TNO 9-12 TCE 18-21 TSU 27-30;
TOC=(TNO+TCE+TSU)/3;
CARDS;
PROC GLM DATA=TEMP;
MODEL TOC=TCE;
TITLE "LINEAR REGRESSION ON OUTPUT TEMPERATURE";
RUN;
QUIT;

```

Table A.3 Summarized SAS output for the estimation of the heater outlet mean air temperature

Parameter	Estimate	Probability	R ²
Model	*	(Pr>F)=0.0001	0.999165
Intercept	1.010101937	(Pr> T)=0.0001	
Slope	0.952654034	(Pr> T)=0.0001	

APPENDIX B

Data and results obtained from a heat loss test

Variables numbered from 1 to 73 and control actions numbered 1 and 2 are identified in Appendix E.

HEURES:	17	17	18	18	18	18	18	18	18	18	18	18	18	18	19	19	19	19	19	19	19	19	19	19
MINUTES:	50	55	0	5	10	15	20	25	30	35	40	45	50	55	0	5	10	15	20	25	30	35	40	45
BILAN :	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24
TOT.ECHANT	12	9	8	9	8	9	9	8	9	8	9	8	9	9	8	9	8	9	9	8	9	8	9	8
DUREE:	284.9	311.2	285.9	310.3	285.0	311.2	310.6	285.3	310.3	285.7	310.3	285.2	310.0	311.2	285.1	310.1	284.9	311.1	310.5	285.0	311.0	285.0	310.3	284.9
DATE: 16-Nov-86																								
VARIABLE:	HPE01	HPE12	HPE13	HPE14	HPE15	HPE16	HPE17	HPE18	HPE19	HPE10	HPE11	HPE12	HPE13	HPE14	HPE15	HPE16	HPE17	HPE18	HPE19	HPE20	HPE21	HPE22	HPE23	HPE24
51	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9
52	12.5	15.8	15.9	16.0	15.9	15.8	15.8	15.8	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.9	15.8	15.9	15.9	15.9	15.9	15.9	15.9
53	29.1	13.3	13.7	13.2	13.3	14.2	13.9	13.2	13.5	14.2	13.8	13.3	13.8	14.0	13.3	13.5	14.2	13.8	13.3	14.0	14.2	13.5	13.4	14.1
54	13.6	34.5	22.5	21.8	41.2	28.9	19.9	19.8	38.7	27.4	19.5	28.1	34.0	22.0	20.5	39.0	27.4	19.4	31.4	34.6	22.7	19.8	37.4	26.8
55	15.4	15.8	16.0	15.3	16.0	17.1	16.5	15.4	16.0	16.8	16.1	15.4	16.6	16.6	15.5	15.9	16.6	15.9	15.3	16.6	16.4	15.4	15.8	15.6
56	33.0	17.7	16.3	15.1	18.1	18.2	16.0	14.8	17.7	17.5	15.7	15.6	17.9	16.3	14.9	17.7	17.5	15.6	16.1	18.3	16.6	15.1	17.5	17.3
57	24.4	37.0	24.0	24.4	45.7	30.6	21.4	22.3	43.2	29.1	21.0	32.3	36.7	23.6	23.0	43.4	29.1	20.9	36.0	36.9	24.2	22.0	41.6	20.4
58	28.0	24.7	24.8	24.7	24.7	24.8	25.0	24.7	24.8	24.6	25.0	24.5	24.5	24.5	24.5	24.5	24.5	24.6	24.4	24.7	24.6	24.5	24.5	24.5
59	2.5	31.8	21.2	21.2	38.2	26.9	19.0	19.4	36.1	25.6	18.6	27.3	31.5	20.9	20.1	36.3	25.6	18.6	30.3	31.9	21.5	19.3	34.9	25.1
60	1.9	1.4	1.1	1.3	1.3	0.7	-0.1	1.1	1.2	1.6	0.1	2.2	2.0	2.1	2.0	2.1	2.1	1.8	2.3	1.4	1.4	2.1	2.1	2.2
61	74.2	1.0	0.7	0.9	1.0	0.4	-0.3	0.8	1.6	1.4	1.5	4.8	3.8	6.0	1.9	2.2	4.4	1.9	3.3	1.3	1.6	1.8	1.7	1.6
62	98.5	69.3	74.9	76.9	68.1	70.3	75.8	78.8	68.3	71.4	75.7	74.4	68.8	73.7	76.1	67.7	70.6	75.1	72.7	67.8	72.6	75.6	68.2	70.9
63	83.8	100.2	100.5	98.3	99.6	100.8	98.8	98.0	99.8	101.1	98.6	96.2	100.0	100.5	97.7	99.2	100.3	99.6	97.8	99.8	100.1	97.5	99.0	100.5
64	88.3	82.8	85.9	85.8	85.3	85.7	85.5	85.7	87.3	86.1	85.3	85.1	84.6	85.7	84.8	84.3	84.8	84.5	85.0	84.2	84.7	84.5	84.2	84.9
65	-0.4	88.5	87.2	88.8	89.0	86.7	87.1	88.6	88.6	86.8	86.9	88.4	87.8	86.0	87.1	88.1	86.2	85.6	87.6	87.1	85.2	86.6	87.8	86.1
66	17.6	1.6	0.4	-0.5	1.9	1.8	0.3	-0.8	1.7	1.4	.0	-0.2	1.7	0.4	-0.7	1.5	1.3	-0.1	0.1	1.8	0.6	-0.6	1.2	1.0
67	89.0	19.3	7.8	3.3	27.5	12.4	5.4	7.5	25.5	11.6	5.3	16.7	18.8	7.3	8.1	25.6	11.6	5.3	19.9	18.6	7.6	6.9	24.1	11.1
68	82.4	90.0	90.4	90.2	90.2	90.8	91.5	90.4	90.4	89.9	91.4	89.4	89.5	89.7	89.5	89.4	89.4	89.8	89.1	90.1	90.1	89.4	89.4	89.3
69	1.3	90.8	91.1	90.9	90.9	91.5	92.3	91.2	91.1	90.7	92.2	90.1	90.2	90.4	90.3	90.1	90.2	90.5	89.9	90.9	90.9	90.2	90.3	90.1
70	83.6	0.8	0.6	0.8	0.7	0.4	0.1	0.7	0.6	0.9	0.2	1.2	1.1	1.0	1.1	1.2	1.2	0.9	1.3	0.8	0.8	1.2	1.2	1.3
71	2.2	84.7	84.9	84.7	84.8	85.4	86.1	85.0	84.6	85.8	84.0	84.0	84.0	84.0	84.0	83.9	84.3	83.7	84.7	84.7	84.1	84.2	84.0	
72	17.6	1.5	1.4	1.6	1.4	1.1	1.1	1.6	1.5	1.2	1.2	1.8	1.3	1.2	1.6	1.5	1.2	1.2	1.6	1.2	1.1	1.5	1.5	1.2
73	2.7	19.7	18.5	17.7	20.1	20.0	18.4	17.4	20.0	19.7	18.3	18.1	20.0	18.8	17.7	19.9	19.7	18.4	18.6	20.4	19.2	18.0	19.8	19.6
		2.1	1.5	1.9	1.9	1.1	1.2	1.9	1.9	1.2	1.4	2.2	1.7	1.2	1.8	1.9	1.3	1.3	2.0	1.5	1.1	1.7	1.9	1.3
CORRANDES	DUREE:																							
1	253.8	0.0	0.0	202.6	107.7	0.0	0.0	177.4	107.8	0.0	0.0	259.9	0.0	0.0	177.3	107.9	0.0	25.3	259.9	0.0	0.0	151.8	107.8	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

APPENDIX C

Computer program for the calculation of
the overall heat loss coefficient

The program was written for the data management software "Lotus 1-2-3".

SRVTST

A93: 'FORMULAS FOR SRVTST
A95: ^V
B95: @IF(B67<0,0,B67)
A96: ^VM
B96: @AVG(B95..Y95)
A97: ^TB
B97: @SUM(C5..Y5)
A98: ^TF
B98: @SUM(B82..Y82)
A99: ^TCT
B99: @SUM(B83..Y83)
A106: ^TOHC
B106: 1.010102+0.95265*B63
A116: ^TFM
B116: (F1) (TOHC+B\$62)/2
A117: ^TFD
B117: +TOHC-B62
A118: ^ESAF
B118: 0.61078*@EXP((17.269*TFM)/(TFM+237.3))
A119: ^HUF
B119: (\$MW/\$MA*B68/100*ESAF)/(101.3171-(B68/100*ESAF))
A120: ^DAF
B120: +\$MA/\$R*(\$P-(1-0.622*DAH)*ESAF*B68/100)/(TFM+273.15)
A121: ^CAF
B121: +\$CPA+\$CPV*DAH
A122: ^PF
B122: @IF(+DAF*\$VAH*\$AI*CPF*TFD>=0,+DAF*\$VAH*\$AI*CPF*TFD,0)
A123: ^UG
B123: +PF/(\$AG*(B37-B39))
A124: ^UGM
B124: @AVG(C123..Y123)

APPENDIX D

Daily Results

RAPPORT DES CUMULATIFS

[illegible]

DATE	TFC	TCC	TPC	TVGC	TVBC	TVNC	TCIC	TLC	OLC	OLEC	OLUC	BFC	BTHLC	BSSTGC	BSSTLC	BESTC	BRTTC	BRSC	ISCC	BRFC	OGFC	BCC	BVC	BMC
15/10/06	1745	42448	0	0	0	1987	72963	63399	124257	0	124257	121941	554016	87651	239901	6634	201143	47401	1795	72347	2943	49918	16300	4636
16/10/06	3247	57740	489	0	0	952	55349	59153	112391	859	111532	192957	715626	190150	145062	77708	160211	133028	5030	50368	20347	67902	22172	4043
17/10/06	2015	46332	7606	0	0	3427	59784	61868	117550	14451	103099	130610	661050	288970	156791	367104	192879	170930	6474	54345	37496	54721	17868	2455
18/10/06	1500	33101	4429	0	0	654	58510	65443	124341	8415	115927	108913	635406	271511	163471	384434	45725	166903	6321	44269	43160	38927	12711	1205
23/10/06	7333	50682	0	0	0	10398	62301	65409	124278	0	124278	481861	892557	131255	74994	17857	225860	163217	6182	2306	27960	59601	19462	1954
24/10/06	7308	73966	1240	0	0	1660	55758	65428	124314	2372	121942	411867	1143044	265727	147183	82408	328000	198401	7514	12803	50313	86984	28403	2681
25/10/06	2013	20720	0	0	0	0	58310	65420	124298	0	124298	178037	855150	105976	225562	12954	81792	110414	4182	19011	20643	24366	7956	800
26/10/06	8417	41521	0	0	0	1389	85987	65433	124323	0	124323	557153	779375	9639	136412	0	242929	338	13	12829	6462	48829	15944	2017
27/10/06	608313	0	0	0	0	5910	676834	65418	124295	0	124295	3064252	4383394	20996	48351	0	0	41398	1568	138250	18095	0	0	2719
28/10/06	3158	1115	0	0	0	5940	70013	65413	124286	0	124286	219002	587935	39529	57092	0	5120	50556	1915	358	29525	1311	478	9
29/10/06	2779	7615	0	0	0	4305	65438	65404	124268	0	124268	183734	648986	109426	24855	0	14897	91801	3477	337	40183	8956	2924	7
30/10/06	6480	33254	1004	0	0	897	66609	58677	111486	1864	109622	377263	964654	104970	252406	55370	151188	120696	4571	11050	29062	29106	12769	1056
01/11/06	5095	10990	0	0	0	300	79325	65462	124378	0	124378	321578	723941	41190	101676	11045	14006	43084	1632	7741	17458	12933	4223	205
02/11/06	6839	61253	815	0	0	680	61054	65465	124383	1549	122834	384984	990196	206226	216223	63145	307142	131211	4969	12727	38917	72033	22521	2354
03/11/06	6807	5471	0	0	0	1362	67500	65425	124308	0	124308	391016	939064	32479	106506	1638	13810	73914	2799	4287	15756	6434	2101	126
04/11/06	13188	67766	0	0	0	245	60407	65457	124369	0	124369	961330	1261995	196786	163914	29895	375763	132673	5025	22328	28912	79693	26022	3057
05/11/06	11078	65053	0	0	0	0	72403	65404	124268	0	124268	566153	1161125	94116	200707	45189	357586	78563	2975	17033	22572	76502	24900	2670
07/11/06	5121	41450	0	0	0	217	64045	65421	124300	0	124300	319613	913239	110983	113600	2920	177547	93687	3548	4899	27728	48392	15801	1133
08/11/06	2126	0	0	0	0	0	79581	65439	124334	0	124334	191528	573241	8412	54564	0	0	11490	435	783	18823	0	0	15
09/11/06	9704	68537	0	0	0	491	61349	65440	124337	0	124337	533360	1105316	221174	214701	44462	354597	122742	4649	8551	31358	80600	26318	2761
10/11/06	11953	15258	0	0	0	709	68846	65450	124355	0	124355	1040484	968852	21302	31229	0	81024	55279	2094	3265	9757	17943	5859	482
11/11/06	10694	22994	0	0	0	163	61621	65423	124304	0	124304	879788	1114798	75612	73205	13	81942	96562	3657	2335	17701	27041	8870	431
12/11/06	70414	60255	0	0	0	0	64490	65424	124305	0	124305	980147	1626224	20010	82535	0	346592	79175	2999	3876	7224	70860	23138	2727
13/11/06	15955	37781	0	0	0	0	61870	65451	124358	0	124358	777872	1527644	5385	48634	0	169660	71460	2708	167	12942	44438	14508	834
14/11/06	10261	4196	0	0	0	109	66943	65441	124338	0	124338	579735	1127647	34033	53544	0	8356	63961	2422	28	27185	4934	1611	1
15/11/06	8267	0	0	0	0	0	70356	63621	120880	0	120880	488923	887544	21299	21873	0	0	31024	1175	0	29265	0	0	0
16/11/06	16579	71414	0	0	0	0	62575	62622	118981	0	118981	835782	1435296	128871	19125	6267	296052	106634	4039	0	28892	82983	27423	1728
17/11/06	20521	60087	0	0	0	0	71171	65454	124363	0	124363	1020530	1437066	7180	86024	0	299923	33216	1258	1361	6548	70662	23073	1864
18/11/06	21448	8770	0	0	0	0	76557	65419	124297	0	124297	982275	1684603	108852	19985	34087	13709	42864	1623	0	81981	10314	3368	40
19/11/06	26221	10701	0	0	0	0	69466	65436	124328	0	124328	1149220	1400842	72108	108160	30585	560	67289	2548	0	41466	12584	4109	0
20/11/06	14729	2449	0	0	0	0	71136	65456	124366	0	124366	719192	1165178	15458	68818	0	4988	38230	1448	0	45547	2879	940	4
21/11/06	13905	17414	0	0	0	109	64837	65431	124318	0	124318	653142	1219384	71806	17596	2151	13462	76636	2902	0	47606	13423	4383	9
22/11/06	12782	11810	0	0	0	0	62524	65455	124365	0	124365	588568	1170055	56565	111635	17699	8659	79377	3006	0	44526	13898	4538	3
23/11/06	11521	17881	3777	0	0	136	64025	65380	124223	6226	117997	560945	1026673	267411	27353	84802	0	118845	4501	0	68223	21028	6866	0
24/11/06	14735	18153	0	0	0	0	62539	65389	124239	0	124239	639150	1367678	23529	191402	5671	7757	69034	2615	130	27540	11940	3899	3
25/11/06	14010	6546	0	0	0	0	75320	65432	124321	0	124321	622494	1154089	75666	79175	6555	0	38769	1468	442	33027	7698	2513	9
26/11/06	18765	21830	0	0	0	0	60456	65320	124109	0	124109	784316	1627347	146508	116339	26313	7941	121732	4610	797	28686	24731	8075	16

SUPPORT DES CIVILAIRES

DATE	BEISIC	BEIRIC	INDC	WOC	TOSH	TSSIN	TSSON	TIH	TOM	MTN	SE	RE	COPSH	COPIN	FF	UF	SF	ESN	ESIN
20/09/85	137916	131168	1.63	328.19	17.7	22.3	22.8	1.8	22.7	20.9	0.0060	0.75	1.2	0.4	0.00	0.00	1.00	0.13	0.13
21/09/85	75931	2536	0.79	95.34	14.7	21.4	21.6	9.3	19.4	10.1	0.0060	0.45	1.5	0.9	0.10	0.00	0.95	0.15	0.30
27/09/85	45176	65	0.02	6.41	11.5	19.6	19.9	7.3	19.1	11.8	0.0070	0.45	2.7	0.2	0.49	0.00	1.00	0.70	0.30
05/10/85	71755	103	0.00	10.48	10.2	19.8	19.9	9.2	18.5	9.3	0.0083	0.66	3.1	0.7	0.49	0.00	1.00	0.21	0.00
06/10/85	0	0	0.00	0.00	10.5	17.9	18.9	2.2	15.0	12.8	MA	0.60	MA	MA	0.44	0.00	1.00	0.23	0.20
09/10/85	0	0	0.00	0.00	11.0	17.0	18.2	4.1	14.5	10.5	MA	0.91	MA	MA	1.00	0.00	1.00	MA	MA
11/10/85	137495	433	0.00	7.94	7.1	18.0	18.4	13.0	17.7	4.6	0.0113	0.60	7.1	0.6	0.42	0.00	0.58	0.38	0.13
12/10/85	6408	0	0.00	1.27	6.3	18.0	18.6	12.7	16.0	3.3	0.0065	0.22	1.0	MA	0.60	0.00	0.40	0.16	MA
14/10/85	0	0	0.00	0.00	8.2	17.1	17.7	8.5	15.6	7.1	MA	0.84	MA	MA	0.44	0.00	1.00	MA	MA
18/10/85	57074	263	0.00	4.63	8.2	17.7	17.7	3.9	17.1	13.1	0.0055	0.54	4.9	0.2	0.44	0.00	0.56	0.28	MA
19/10/85	71040	0	0.00	7.00	7.3	17.8	17.7	9.7	16.3	6.6	0.0087	0.64	3.9	MA	0.27	0.00	0.73	0.31	MA
21/10/85	131825	0	0.00	12.86	6.4	17.6	17.8	8.4	16.6	8.1	0.0089	0.60	4.0	MA	0.36	0.00	0.64	0.31	MA
22/10/85	131825	0	0.28	8.04	6.7	18.0	18.1	8.2	17.0	8.8	0.0089	0.61	4.7	MA	0.35	0.00	0.65	0.31	MA
23/10/85	100908	75	0.00	11.10	6.0	18.3	18.4	6.1	17.1	11.0	0.0084	0.64	3.1	0.4	0.45	0.00	0.55	0.27	0.03
28/10/85	108579	0	0.00	10.78	4.5	16.6	17.2	16.0	16.2	9.1	0.0100	0.77	4.3	MA	0.36	0.00	0.64	0.32	MA
29/10/85	0	0	0.00	0.00	4.1	16.8	17.4	13.4	14.9	1.5	MA	0.49	MA	MA	0.40	0.00	0.60	MA	MA
30/10/85	127237	9	0.65	5.01	3.8	16.8	17.2	14.4	16.4	1.9	0.0105	0.70	6.5	0.3	0.40	0.00	0.60	0.37	0.02
31/10/85	56452	0	0.26	3.25	3.2	17.3	17.6	14.4	16.1	1.8	0.0087	0.83	4.9	MA	0.38	0.00	0.62	0.36	MA
01/11/85	0	0	0.00	0.00	5.0	16.6	17.1	10.3	15.0	4.7	MA	0.59	MA	MA	0.51	0.00	0.49	1.00	MA
02/11/85	0	0	0.00	0.00	3.4	16.1	16.6	11.5	15.3	3.8	MA	0.77	MA	MA	0.49	0.00	0.51	1.00	MA
03/11/85	19020	0	0.00	2.71	4.6	16.2	16.6	16.4	16.2	-0.2	0.0075	1.00	2.7	MA	0.42	0.00	0.58	1.00	MA
04/11/85	0	0	0.00	0.00	5.3	15.7	16.0	11.7	15.2	3.5	MA	1.00	MA	MA	0.59	0.00	0.41	1.00	MA
05/11/85	75956	0	0.23	5.29	4.3	16.5	16.2	12.8	16.3	3.4	0.0100	0.86	4.8	MA	0.35	0.00	0.65	1.00	MA
08/11/85	72489	0	0.22	6.79	1.4	16.9	16.6	23.5	15.8	-7.7	0.0109	0.67	5.4	MA	0.46	0.00	0.54	1.00	MA
14/11/85	87854	0	0.34	12.88	1.9	16.6	16.0	19.0	16.4	-2.6	0.0056	1.00	5.9	MA	0.34	0.00	0.66	1.00	MA
15/11/85	127615	0	0.20	6.65	1.8	16.8	16.3	20.5	16.2	-4.4	0.0124	1.00	7.1	MA	0.36	0.00	0.64	1.00	MA
16/11/85	0	0	0.00	0.00	1.9	16.1	16.0	17.3	15.0	-2.2	MA	0.83	2.1	MA	0.54	0.00	0.48	1.00	MA
17/11/85	0	0	0.00	0.00	2.1	15.9	15.8	14.1	15.1	-0.9	MA	1.00	2.1	MA	0.58	0.00	0.41	1.00	MA
20/11/85	38228	0	0.00	2.91	3.7	15.7	15.3	10.8	15.5	4.7	0.0082	1.00	4.2	MA	1.00	0.00	1.00	0.39	MA
21/11/85	0	0	0.00	0.00	2.0	13.6	14.3	13.5	11.1	-2.4	MA	0.98	MA	MA	1.00	0.00	1.00	MA	MA
22/11/85	0	0	0.00	0.00	1.9	12.8	12.7	15.5	11.5	-4.0	MA	1.00	MA	MA	1.00	0.00	1.00	MA	MA
23/11/85	0	0	0.00	0.00	1.9	12.5	12.3	14.1	11.6	-2.5	MA	1.00	MA	MA	1.00	0.00	1.00	MA	MA
26/11/85	55142	1697	0.00	13.56	1.1	14.3	14.0	20.1	12.9	-7.2	0.0069	1.00	2.6	0.4	0.71	0.00	0.29	0.33	MA
27/11/85	308	0	0.00	0.04	1.1	12.2	12.9	16.2	11.2	-5.0	MA	1.00	2.0	MA	0.91	0.00	0.09	0.39	MA
28/11/85	9937	0	0.07	0.33	0.5	12.3	12.8	20.8	11.4	-9.4	0.0113	1.00	3.6	MA	0.84	0.00	0.16	1.00	MA
29/11/85	14946	0	0.10	0.22	0.1	12.5	12.7	19.8	11.5	-8.3	0.0105	1.00	4.5	MA	0.81	0.00	0.16	1.00	MA
30/11/85	53574	358	0.00	11.06	-3.8	17.1	15.6	26.2	16.0	-10.2	0.0099	0.65	2.1	0.6	0.39	0.03	0.56	0.64	0.03
01/12/86	11246	0	0.34	16.59	-3.7	17.6	16.0	23.9	16.8	-9.1	0.0103	0.88	4.7	MA	0.46	0.07	0.48	1.00	MA
02/12/86	8445	0	0.00	3.98	-3.6	16.5	15.2	22.9	15.7	-7.1	0.0044	0.51	0.9	MA	0.54	0.06	0.39	1.00	MA
03/12/86	164766	0	0.69	16.87	-3.5	18.1	16.2	25.5	17.4	-8.1	0.0111	0.85	6.6	MA	0.42	0.07	0.50	1.00	MA
04/12/86	137223	0	0.21	11.53	-3.6	18.0	16.4	26.7	16.2	-10.5	0.0102	0.87	8.1	MA	0.97	0.07	0.00	0.47	MA
07/12/86	178172	0	0.29	12.15	-4.1	18.8	16.5	26.5	17.6	-8.9	0.0129	0.80	8.2	MA	0.42	0.07	0.51	1.00	MA
08/12/86	0	0	0.00	0.00	-3.1	16.8	15.5	20.3	15.6	-5.1	MA	0.70	MA	MA	0.62	0.07	0.30	1.00	MA
09/12/86	179196	0	0.00	24.75	-3.2	18.3	16.6	24.0	17.9	-6.1	0.0118	1.00	5.9	MA	0.39	0.07	0.54	1.00	MA
10/12/86	194090	0	143.31	0.77	-3.2	19.1	17.3	18.9	18.0	-0.9	0.0155	0.36	31.0	MA	0.39	0.03	0.52	1.00	MA
11/12/86	0	0	0.00	0.00	-2.4	16.3	15.6	22.1	16.0	-6.1	MA	0.65	MA	MA	0.67	0.05	0.24	1.00	MA
14/12/86	127693	0	0.11	22.88	-2.3	18.6	17.0	24.2	17.4	-6.8	0.0101	0.76	4.0	MA	0.44	0.07	0.48	1.00	MA
15/12/86	19169	0	0.06	6.86	-2.1	17.2	16.2	23.4	16.2	-7.2	0.0084	0.27	1.4	MA	0.46	0.07	0.47	1.00	MA
16/12/86	147889	548	0.13	22.59	-3.1	16.6	15.3	24.6	17.3	-7.2	0.0124	0.86	6.1	0.8	0.31	0.04	0.63	0.53	0.00
17/12/86	178308	408	0.00	21.32	-2.9	17.7	16.2	22.6	17.9	-4.7	0.0113	0.71	6.7	0.7	0.39	0.07	0.54	0.47	0.09
18/12/86	40883	0	0.00	32.55	-2.9	18.7	17.3	23.0	18.6	-4.4	0.0128	0.74	6.4	0.1	0.36	0.07	0.57	1.00	0.01
01/03/87	0	0	0.09	7.40	-2.6	18.3	17.3	16.6	16.7	9.0	0.0132	0.71	4.6	MA	0.36	0.10	0.53	1.00	0.46
02/03/87	85044	161	0.00	16.67	-2.0	17.5	16.6	29.1	17.2	-12.0	0.0105	0.79	4.7	0.3	0.39	0.06	0.55	0.37	0.02

[illegible]

DATE	QETIC	QETIC	BHOC	HOC	IOSR	TSSM	TSSM	TSSM	TIM	TOM	MCIN	SE	RE	CDPSH	CDPM	FF	UF	SF	RSR	ESRN	EREN
15/10/86	1168	163392	21.25	24.37	7.1	17.7	16.9	10.3	15.6	5.4	0.0328	1.00	0.38	3.8	3.7	0.22	0.22	0.36	0.31	0.13	0.29
16/10/86	47311	155190	30.89	16.05	6.0	17.1	16.3	13.2	16.3	3.1	0.0114	0.86	1.00	9.7	2.8	0.27	0.16	0.57	0.25	0.33	0.24
17/10/86	113442	70994	95.06	42.07	5.1	17.6	16.7	12.2	17.1	4.9	0.0111	0.50	0.80	18.6	2.0	0.20	0.16	0.65	0.44	0.37	0.17
18/10/86	115295	16983	101.62	9.79	3.4	17.9	17.0	8.7	17.9	9.2	0.0072	0.46	0.37	11.8	1.8	0.17	0.18	0.65	0.72	0.28	0.14
23/10/86	39666	97074	0.63	53.41	4.7	15.7	15.7	16.1	16.4	6.3	0.0083	0.63	1.00	2.8	2.0	0.34	0.14	0.32	0.02	0.27	0.19
24/10/86	123521	0.02	95.33	3.5	16.6	16.2	17.5	17.7	17.7	0.2	0.0090	0.80	0.93	4.3	2.0	0.36	0.11	0.53	0.09	0.38	0.17
25/10/86	23744	23528	2.34	26.35	3.9	16.6	16.5	13.1	16.7	3.6	0.0065	0.74	0.20	4.9	1.7	0.21	0.15	0.65	0.42	0.25	0.17
26/10/86	462	89719	0.00	53.95	3.4	15.9	15.9	16.1	15.8	-0.3	MA	1.00	0.75	MA	1.7	0.71	0.16	0.13	0.13	0.12	0.16
27/10/86	0	0	0.00	0.00	4.1	15.3	15.4	12.5	15.8	3.3	MA	0.35	1.00	MA	MA	0.70	0.03	0.27	1.00	MA	MA
28/10/86	0	119	0.00	1.82	5.2	15.3	15.2	10.1	16.1	5.1	MA	0.49	0.01	MA	0.1	0.37	0.21	0.42	0.75	MA	0.09
29/10/86	17169	32	0.00	12.31	6.1	15.4	15.2	10.9	16.6	5.7	0.0059	0.83	0.01	MA	0.2	0.28	0.19	0.53	0.91	4.25	0.60
31/10/86	77262	42632	0.00	49.36	3.0	16.4	16.1	15.7	16.9	1.2	0.0090	0.51	0.21	5.0	2.1	0.39	0.11	0.50	0.21	0.30	0.19
01/11/86	22472	26676	1.15	9.78	3.7	15.5	15.5	12.3	15.8	3.5	0.0101	0.38	0.10	3.2	1.0	0.44	0.17	0.38	0.74	0.31	0.10
02/11/86	76803	106947	1.14	79.13	3.0	16.3	16.1	18.0	17.1	-1.0	0.0093	0.88	0.53	4.7	2.0	0.39	0.12	0.49	0.11	0.32	0.18
03/11/86	4054	2123	0.14	5.18	2.5	15.9	15.6	15.4	16.3	0.9	0.0036	0.35	0.06	2.6	1.0	0.42	0.13	0.45	0.67	0.20	0.13
04/11/86	58311	132106	0.00	99.01	2.3	16.4	16.0	21.5	16.6	-4.9	0.0098	1.00	0.95	3.5	2.1	0.76	0.10	0.14	0.14	0.36	0.10
05/11/86	66158	117948	0.00	95.51	1.4	16.5	16.2	21.0	16.7	-4.3	0.0094	0.64	0.48	5.3	2.0	0.49	0.11	0.41	0.13	0.27	0.17
07/11/86	26901	52703	0.30	34.27	1.4	15.8	15.5	14.9	16.5	1.6	0.0077	0.89	0.51	3.3	1.2	0.35	0.14	0.51	0.69	0.35	0.11
08/11/86	0	0	0.00	0.00	2.3	15.1	15.0	11.7	15.7	4.0	MA	0.28	0.01	MA	MA	0.33	0.22	0.45	1.00	MA	MA
10/11/86	65304	131849	0.01	90.05	2.2	16.1	15.8	20.3	16.8	-3.5	0.0085	1.00	0.65	4.0	2.2	0.48	0.11	0.40	0.66	0.29	0.19
11/11/86	0	21759	0.00	21.64	1.6	15.7	15.3	16.8	15.8	-1.0	MA	0.35	0.75	MA	1.0	1.00	0.13	0.60	0.13	MA	0.08
12/11/86	23769	19595	0.00	31.86	1.5	15.9	15.5	18.4	16.4	-2.0	0.0073	0.66	0.30	0.1	1.1	0.79	0.11	0.10	0.11	0.29	0.11
13/11/86	0	134811	0.00	75.95	0.9	15.6	15.4	27.3	15.7	-11.6	MA	0.23	1.00	MA	1.9	0.60	0.08	0.32	0.03	MA	0.16
14/11/86	421	42264	0.00	46.31	0.4	15.2	14.9	19.7	15.6	-4.1	MA	0.06	0.87	MA	0.9	0.31	0.08	0.41	0.00	0.20	0.08
15/11/86	5873	0	0.00	5.35	0.6	15.1	14.8	17.9	16.0	-2.0	0.0042	0.37	0.00	MA	MA	0.51	0.11	0.38	1.00	0.19	MA
17/11/86	0	0	0.00	0.00	0.5	14.8	14.4	15.5	15.7	0.2	MA	0.35	0.09	MA	MA	0.55	0.14	0.31	MA	MA	MA
18/11/86	42830	87878	0.09	90.19	0.3	15.1	14.9	26.3	16.0	-10.2	0.0084	0.91	1.00	1.3	1.3	0.58	0.08	0.33	0.00	0.31	0.12
19/11/86	0	93425	0.00	74.80	-0.8	15.3	15.1	28.3	15.5	-12.8	MA	0.18	1.00	MA	1.4	0.71	0.09	0.20	0.81	MA	0.12
21/11/86	44710	2046	0.09	9.34	-0.8	16.4	15.3	24.7	18.8	-5.9	0.0111	0.68	0.10	6.0	1.1	0.58	0.07	0.34	0.00	0.48	0.13
22/11/86	55878	0	0.25	11.31	-1.1	17.7	16.6	27.9	18.9	-9.0	0.0095	0.52	0.00	3.4	MA	0.82	0.09	0.09	MA	0.30	MA
23/11/86	3013	194	0.00	2.90	-0.8	17.4	16.0	21.7	18.7	-3.0	0.0039	0.18	0.00	MA	0.7	0.62	0.11	0.28	0.00	0.17	0.07
24/11/86	26092	477	0.00	14.18	-0.3	17.3	16.0	19.5	18.8	-0.7	0.0065	0.57	0.03	0.5	0.6	0.54	0.10	0.36	0.00	0.21	0.07
25/11/86	49302	129	0.00	16.24	-0.3	17.6	16.5	20.7	19.0	-1.7	0.0096	0.40	0.00	4.1	0.2	0.50	0.11	0.39	0.00	0.24	0.06
26/11/86	120384	0	0.00	16.79	-0.2	17.5	16.3	19.5	19.6	0.1	0.0118	0.98	0.00	6.9	MA	0.55	0.11	0.34	MA	0.46	MA
27/11/86	32762	0	0.24	13.97	-0.2	18.1	17.1	19.8	18.9	-0.8	0.0127	0.23	0.00	1.7	MA	0.47	0.09	0.44	1.00	0.26	MA
28/11/86	22911	0	0.00	6.56	-0.2	17.9	16.7	17.4	18.9	1.4	0.0122	0.97	0.01	1.2	MA	0.54	0.11	0.35	1.00	0.29	MA
29/12/86	94246	0	0.09	30.06	-2.0	18.3	17.5	29.1	19.2	-10.0	0.0127	0.63	0.81	2.3	MA	0.44	0.08	0.44	1.00	0.33	MA

APPENDIX E

Example of raw, treated
and compiled data

MONTH DAY # HR. MIN. SEC.
 04 18 05 18 12 26.143

LEGEND

: DAY OF THE WEEK
 VAR : VARIABLE
 MAX : MAXIMUM VALUE
 MIN : MINIMUM VALUE
 STD : STANDARD DEVIATION

REPORT SAMPLES PERIOD
 1 135 3599.79

VAR	MAX	MIN	MEAN	STD	SENSOR IDENTIFICATION
1	25.45	24.53	24.92	0.20	T1
2	22.63	21.90	22.15	0.07	T2
3	22.29	21.71	21.80	0.08	T3
4	21.58	20.70	21.05	0.20	T4
5	24.53	23.89	24.28	0.16	NOT USED
6	25.66	25.04	25.12	0.08	AIR INLET STATUS
7	21.77	20.50	21.11	0.38	T7
8	20.23	20.09	20.14	0.02	T8
9	19.76	18.59	19.29	0.09	T9
10	21.06	19.59	20.27	0.43	T10
11	25.22	24.13	24.57	0.23	T12
12	12.77	12.43	12.49	0.04	NOT USED
13	21.72	21.38	21.54	0.03	T14
14	9.49	0.00	9.18	0.38	ΔT INSIDE-OUTSIDE
15	20.60	20.49	20.53	0.02	T16
16	22.43	21.79	22.11	0.14	T17
17	17.43	16.80	17.00	0.07	T18
18	14.90	14.19	14.46	0.07	T19
19	3.64	2.47	2.94	0.23	T20
20	24.14	23.38	23.66	0.15	T21
21	20.30	19.89	20.06	0.03	T22
22	17.86	17.37	17.54	0.04	T23
23	21.80	21.03	21.39	0.22	T24
24	19.72	18.02	18.86	0.46	T25
25	20.50	19.75	20.08	0.19	T26
26	18.51	16.84	17.63	0.47	T27
27	22.17	18.44	19.89	0.98	T28
28	18.51	15.55	16.87	0.81	T29
29	22.98	22.36	22.60	0.12	T30
30	20.41	18.23	19.44	0.52	T31
31	18.46	16.63	17.54	0.50	T32
32	7.26	4.32	5.69	0.76	T33
33	23.84	23.15	23.37	0.12	T34
34	21.34	20.92	21.25	0.06	T35
35	23.30	22.71	22.85	0.06	T36
36	20.69	19.93	20.24	0.07	T37
37	19.46	19.30	19.35	0.02	T38
38	21.55	21.26	21.49	0.04	T39
39	21.39	21.05	21.21	0.03	T40
40	23.67	20.10	21.55	1.06	T41
41	21.97	21.22	21.48	0.08	T42
42	21.23	20.69	20.77	0.05	NOT USED
43	21.07	20.41	20.54	0.08	T44

44	14.24	14.15	14.19	0.02	T45
45	5.57	4.70	5.00	0.08	T46
46	3.83	3.20	3.46	0.06	T47
47	5.25	5.00	5.04	0.03	T48
48	24.11	23.27	23.49	0.08	T50 NOT USED
49	22.99	22.34	22.56	0.06	T51
50	22.90	22.61	22.75	0.03	T52
51	22.69	21.17	21.80	0.39	T53
52	25.77	25.52	25.66	0.04	T54 NOT USED
53	23.69	22.63	22.99	0.20	T55 NOT USED
54	21.15	12.92	16.54	2.81	T56
55	20.30	18.51	19.40	0.42	T57
56	21.33	19.30	20.26	0.52	T58
57	26.03	25.28	25.56	0.08	LIGHTING SYSTEM STATUS
58	26.38	25.62	25.94	0.16	T60 NOT USED
59	10.21	3.10	5.05	1.98	PYRANOMETRER
60	27.08	-0.03	9.53	4.07	ANEMOMETER
61	86.11	78.47	83.13	1.12	RH AMBIANT
62	75.98	68.88	73.98	0.96	RH PIPES INLET
63	82.79	74.80	78.79	2.00	RH PIPES OUTLET 1
64	83.85	73.64	78.92	2.49	RH PIPES OUTLET 2
65	4.58	0.07	2.75	1.12	δT RIDGE-SOIL
66	1.49	0.62	0.86	0.18	δT HEATER
67	93.67	93.27	93.38	0.05	LOUVER STATUS
68	94.02	25.09	40.68	28.72	CURTAIN STATUS
69	26.13	25.74	25.87	0.06	CIRCULATOR STATUS
70	88.88	88.26	88.33	0.06	VENTILATION STATUS
71	0.40	-2.61	-1.50	0.76	δT PIPES ROW 1
72	14.62	12.91	13.75	0.30	δT INSIDE-OUTSIDE
73	1.78	-1.40	-0.19	0.79	δT PIPES ROW 2

CO PERIOD

1	0	HEATING
2	3593.80	STORAGE/RECOVERY
3	0	AIR INLET
4	0	VENTILATION LOW SPEED
5	0	VENTILATION HIGH SPEED
6	0	VENTILATION HUMIDITY
7	2789.14	THERMAL CURTAIN
8	3593.80	SODIUM LAMPS

CO: COMMAND NO.

PERIOD: OPERATING TIME
IN SECONDES

RH: RELATIVE HUMIDITY

δT: TEMPERATURE
DIFFERENCE

V.: SPEED

APPENDIX F

Computater program

The program was written for the data management software "Lotus 1-2-3".

COMPUTATION FORMULA

A94: 'GENERAL COMPUTATION FORMULAS
 A96: TOS
 B96: (F1) (B26+B54+B53)/3
 A97: TSS1
 B97: (F1) ((B40+B18+B27)/3+B23)/2
 A98: TSS2
 B98: (F1) ((B56+B57+B9+B10+B48+B49)/6+(B23+B24)/2)/2
 A99: TSS3
 B99: (F1) ((B36+B41+B46+B15+B28+B29)/6+(B24+B25)/2)/2
 A100: TSSG
 B100: (F1) (TSS1*2+TSS2*2+TSS3*2)/6
 A101: TOHC
 B101: (F1) 0.9979+0.95284*B63
 A102: TFD
 B102: (F1) +TOHC-B62
 A103: TFM
 B103: (F1) (TOHC+B62)/2
 A104: TCSD
 B104: (F1) +B61-TSSG
 A105: TASD
 B105: (F1) +TSSG-B37
 A106: TIOM
 B106: (F1) (B39+B37)/2
 A107: TIOD
 B107: (F1) +B37-B39
 A108: TTD
 B108: (F1) (B30+B34)/2-B11
 A110: ESAF
 B110: (F2) 0.61078*@EXP((17.269*TFM)/(TFM+237.3))
 A111: HUF
 B111: (F4) (\$MW/\$MA*(@IF(B68>100,100,B68))/100*ESAF)/
 (\$P-((@IF(B68>100,100,B68))/100*ESAF))
 A112: DAF
 B112: (F2) +\$MA/\$R*(\$P-(1-0.622*HUF)*ESAF*(@IF(B68>100,
 100,B68))/100)/(TFM+273.15)
 A113: CPF
 B113: (F2) +\$CPA+\$CPV*HUF
 A114: PF
 B114: (F1) @IF(B82=0,0,(DAF*\$VAH*\$AI*CPF*TFD))
 A115: ESAV
 B115: (F2) 0.61078*@EXP((17.269*TIOM)/(TIOM+237.3))
 A116: HUV
 B116: (F4) (\$MW/\$MA*(@IF(B68>100,100,B68))/100*ESAV)/(\$P-
 ((@IF(B68>100,100,B68))/100*ESAV))
 A117: DAV
 B117: (F2) +\$MA/\$R*(\$P-(1-0.622*HUV)*ESAV*(@IF(B68>100,100,
 B68))/100)/(TIOM+273.15)
 A118: CPVH
 B118: (F2) +\$CPA+\$CPV*HUV
 A119: ESTI
 B119: (F2) 0.61078*@EXP((17.269*B34)/(B34+237.3))
 A120: HUTI

B120: (F4) ($\$MW/\$MA * (@IF(B69 > 100, 100, B69)) / 100 * ESTI$) /
 ($\$P + \$SPT - ((@IF(B69 > 100, 100, B69)) / 100 * ESTI)$)
 A121: ^DATI
 B121: (F2) $+\$MA/\$R * (\$P + \$SPT - (1 - 0.622 * HUTI) * ESTI * (@IF(B69 > 100, 100, B69)) / 100) / (B34 + 273.15)$
 A122: ^CPTI
 B122: (F2) $+\$CPA + \$CPV * HUTI$
 A123: ^HTI
 B123: (F1) $+CPTI * B34 + HUTI * \$LH$
 A124: ^AHUI
 B124: (F4) $+\$MW * ESTI * (@IF(B69 > 100, 100, B69)) / (100 * \$R * (B34 + 273.15))$
 A125: ^ESTO1
 B125: (F2) $0.61078 * @EXP((17.269 * B30) / (B30 + 237.3))$
 A126: ^HUTO1
 B126: (F4) ($\$MW/\$MA * (@IF(B70 > 100, 100, B70)) / 100 * ESTO1$) /
 ($\$P - ((@IF(B70 > 100, 100, B70)) / 100 * ESTO1)$)
 A127: ^DATO1
 B127: (F2) $+\$MA/\$R * (\$P - (1 - 0.622 * HUTO1) * ESTO1 * (@IF(B70 > 100, 100, B70)) / 100) / (B30 + 273.15)$
 A128: ^CPTO1
 B128: (F2) $+\$CPA + \$CPV * HUTO1$
 A129: ^HTO1
 B129: (F1) $+CPTO1 * B30 + HUTO1 * \$LH$
 A130: ^AHUO1
 B130: (F4) $+\$MW * ESTO1 * (@IF(B70 > 100, 100, B70)) / (100 * \$R * (B30 + 273.15))$
 A131: ^ESTO2
 B131: (F2) $0.61078 * @EXP((17.269 * B32) / (B32 + 237.3))$
 A132: ^HUTO2
 B132: (F4) ($\$MW/\$MA * (@IF(B71 > 100, 100, B71)) / 100 * ESTO2$) /
 ($\$P - ((@IF(B71 > 100, 100, B71)) / 100 * ESTO2)$)
 A133: ^DATO2
 B133: (F2) $+\$MA/\$R * (\$P - (1 - 0.622 * HUTO2) * ESTO1 * (@IF(B71 > 100, 100, B71)) / 100) / (B32 + 273.15)$
 A134: ^CPTO2
 B134: (F2) $+\$CPA + \$CPV * HUTO2$
 A135: ^HTO2
 B135: (F1) $+CPTO2 * B32 + HUTO2 * \$LH$
 A136: ^AHUO2
 B136: (F4) $+\$MW * ESTO2 * (@IF(B71 > 100, 100, B71)) / (100 * \$R * (B32 + 273.15))$
 A137: ^DATM1
 B137: (F2) $(DATI + DATO1) / 2$
 A138: ^DATM2
 B138: (F2) $(DATI + DATO2) / 2$
 A139: ^CPTM1
 B139: (F2) $(CPTI + CPTO1) / 2$
 A140: ^CPTM2
 B140: (F2) $(CPTI + CPTO2) / 2$
 A141: ^UG
 B141: (F4) $(B5 - B88) / B5 * @IF(B67 < 0.1, 0.0051, (5.556158 * 10^{-3} + 7.945778 * 10^{-5} * B67)) + B88 / B5 * \$ECC * @IF(B67 < 0.1, 0.0051, (5.556158 * 10^{-3} + 7.945778 * 10^{-5} * B67))$

```

10-5*B67))
A142: ^ESE
B142: (F2) @IF(B83=0#OR#(B34-(B30+B32)/2)<0,@NA,
      (B34-(B30+B32)/2)/(B34-$TSIF))
A143: ^ERE
B143: (F2) @IF(B83=0#OR#(B30+B32)/2-B34<0,@NA,((B30+B32)/
      2-B34)/($TSIC-B34))
A144: ^QL
B144: (F0) +$PL*B89
A145: ^QLE
B145: (F0) +QL*B84/B5
A146: ^QLU
B146: (F0) +QL-QLE
A147: ^QF
B147: (F0) +PF*B5
A148: ^QGH
B148: (F0) +UG*$AG*(B37-B39)*B5
A149: ^QSH
B149: (F0) +$USM*$PE*(TSSG-B39)*B5
A150: ^QVHL
B150: (F0) +DAV*$CPV*$FRH*(B37-B39)*B86
A151: ^QTHL
B151: (F0) +QGH+QSH+QVHL
A152: ^QSS1
B152: (F0) @NA
A153: ^QSS2
B153: (F0) @NA
A154: ^QSS3
B154: (F0) @NA
A155: ^QSST
B155: (F0) @NA
A156: ^QSSTG
B156: (F0) @IF(QSST<0,0,QSST)
A157: ^QSSTL
B157: (F0) @IF(QSST<0,@ABS(QSST),0)
A158: ^QEX1
B158: (F0) +DATI*$FRT1*(HTI-HTO1)*B83
A159: ^QEX2
B159: (F0) +DATI*$FRT2*(HTI-HTO2)*B$83
A160: ^QES1
B160: (F0) @IF(QEX1<0,0,QEX1)
A161: ^QES2
B161: (F0) @IF(QEX2<0,0,QEX2)
A162: ^QRT1
B162: (F0) @IF(QEX1<0,@ABS(QEX1),0)
A163: ^QRT2
B163: (F0) @IF(QEX2<0,@ABS(QEX2),0)
A164: ^QEST
B164: (F0) +QES1+QES2
A165: ^QRTT
B165: (F0) +QRT1+QRT2
A166: ^ISC
B166: (F0) @IF((B66-$COR)<0,0,(B66-$COR)*(B5-B88)/100)
A167: ^QRS

```

```

B167: (F0) +$A*$X*$SS*ISC
A168: ^QCF
B168: (F0) +$HCE*$AF*(TSS1-B$37)*B$5
A169: ^QRF
B169: (F0) @IF(QCF<0,0,QCF)
A170: (D1) ^QGF
B170: (F0) @IF(QCF<0,@ABS(QCF),0)
A171: ^QC
B171: (F0) +$VC*$IC*B83/1000
A172: ^QV
B172: (F0) @IF((B83-B85)<0,0,($VV*$IV*(B83-B85)/1000))
A173: ^QET1
B173: (F0) +DATM1*CPTM1*$FRT1*(B34-B30)*B83
A174: ^QET2
B174: (F0) +DATM2*CPTM2*$FRT2*(B34-B32)*B83
A175: ^QETS1
B175: (F0) @IF(QET1<0,0,QET1)
A176: ^QETS2
B176: (F0) @IF(QET2<0,0,QET2)
A177: ^QETR1
B177: (F0) @IF(QET1<0,@ABS(QET1),0)
A178: ^QETR2
B178: (F0) @IF(QET2<0,@ABS(QET2),0)
A179: ^QETST
B179: (F0) +QETS1+QETS2
A180: ^QETRT
B180: (F0) +QETR1+QETR2
A181: ^QBR
B181: (F0) +$VBR*$IBR*(QETRT+QRF)/$PFM/1000
A182: ^HCT
B182: (F4) @IF(@ABS(TTD)<=0.8#OR#B83=0,@NA,
    @IF((QET1/(13*$AT*TTD*B83))<0,@NA,(QET1/
    (13*$AT*TTD*B83))))
A183: ^AHU1
B183: (F2) (AHUI-AHU01)*$FRT1*B83
A184: ^AHU2
B184: (F2) (AHUI-AHU02)*$FRT2*B83
A185: ^DHU1
B185: (F2) @IF(AHU1<0,0,AHU1)
A186: ^DHU2
B186: (F2) @IF(AHU2<0,0,AHU2)
A187: ^HUM1
B187: (F2) @IF(AHU1<0,@ABS(AHU1),0)
A188: ^HUM2
B188: (F2) @IF(AHU2<0,@ABS(AHU2),0)
A189: ^DHU
B189: (F2) +DHU1+DHU2
A190: ^HUM
B190: (F2) +HUM1+HUM2
A192: ^COPS
B192: (F1) @IF((QC-QV)<=0#OR#QEST<=0,@NA,(QEST/(QC-QV)))
A193: ^COPR
B193: (F1) @IF((QC-QBR)<=0#OR#QETRT<=0,@NA,(QETRT/(QC-QBR)))
A195: 'ISNA ESE

```

B195: @ISNA(ESE)
 A196: 'ISNA ERE
 B196: @ISNA(ERE)
 A197: 'ISNA HCT
 B197: @ISNA(HCT)
 A198: 'ISNA COPS
 B198: @ISNA(COPS)
 A199: 'ISNA COPR
 B199: @ISNA(COPR)
 A200: 'SUM(HCT)=
 B200: @SUM(B197..Y197)
 A201: 'HCT C
 B201: @IF(@ISNA(HCT)=0,HCT,0)
 A202: 'COPS C
 B202: @IF(@ISNA(COPS)=0,COPS,0)
 A203: 'COPR C
 B203: @IF(@ISNA(COPR)=0,COPR,0)
 A204: 'ESE C
 B204: @IF(@ISNA(ESE)=0,ESE,0)
 A205: 'ERE C
 B205: @IF(@ISNA(A143)=0,A143,0)
 A206: 'MOY. HCT:
 B206: @IF(B200=24,@NA,@SUM(B201..Y201)/(24-B200))
 A209: "TFC =
 B209: (FO) @SUM(B\$82..Y\$82)
 A210: "TCC =
 B210: (FO) @SUM(B\$83..Y\$83)
 A211: "TPC =
 B211: (FO) @SUM(B\$84..Y\$84)
 A212: "TVGC =
 B212: (FO) @SUM(B\$86..Y\$86)
 A213: "TVBC =
 B213: (FO) @SUM(B\$85..Y\$86)-TVGC
 A214: "TVHC =
 B214: (FO) @SUM(B\$87..Y\$87)
 A215: "TCTC =
 B215: (FO) @SUM(B\$88..Y\$88)
 A216: "TLC =
 B216: (FO) @SUM(B\$89..Y\$89)
 A217: "QLC =
 B217: (FO) @SUM(B\$144..Y\$144)
 A218: "QLEC =
 B218: (FO) @SUM(B\$145..Y\$145)
 A219: "QLUC =
 B219: (FO) @SUM(B\$146..Y\$146)
 A220: "QFC =
 B220: (FO) @SUM(B\$147..Y\$147)
 A221: "QTHLC =
 B221: (FO) @SUM(B\$151..Y\$151)
 A222: "QSSTGC =
 B222: (FO) @SUM(C\$156..Y\$156)
 A223: "QSSTLC =
 B223: (FO) @SUM(C\$157..Y\$157)
 A224: "QESTC =

B224: (F0) @SUM(B\$164..Y\$164)
 A225: "QRTTC =
 B225: (F0) @SUM(B\$165..Y\$165)
 A226: "QRSC =
 B226: (F0) @SUM(B\$167..Y\$167)
 A227: "ISCC =
 B227: (F0) @SUM(B\$166..Y\$166)
 A228: "QRFC =
 B228: (F0) @SUM(B\$169..Y\$169)
 A229: "QGFC =
 B229: (F0) @SUM(B\$170..Y\$170)
 A230: "QCC =
 B230: (F0) @SUM(B\$171..Y\$171)
 A231: "QVC =
 B231: (F0) @SUM(B\$172..Y\$172)
 A232: "QBRC =
 B232: (F0) @SUM(B\$181..Y\$181)
 A233: "QETSTC =
 B233: (F0) @SUM(B\$179..Y\$179)
 A234: "QETRTC =
 B234: (F0) @SUM(B\$180..Y\$180)
 A235: "DHUC =
 B235: (F2) @SUM(B\$189..Y\$189)
 A236: "HUC =
 B236: (F2) @SUM(B\$190..Y\$190)
 A237: "TOSM =
 B237: (F1) @AVG(B\$96..Y\$96)
 A238: "TSS1M =
 B238: (F1) @AVG(B\$97..Y\$97)
 A239: "TSSGM =
 B239: (F1) @AVG(B\$100..Y\$100)
 A240: "TIODM =
 B240: (F1) @AVG(B\$107..Y\$107)
 A241: "TIM =
 B241: (F1) @AVG(\$B\$37..\$Y\$37)
 A242: "TOM =
 B242: (F1) @AVG(\$B\$39..\$Y\$39)
 A243: "HCTM =
 B243: (F4) +\$B\$206
 A244: "SE =
 B244: (F2) @IF(QSSTGC/(QESTC+QRSC+QGFC)>1,1,QSSTGC/
 (QESTC+QRSC+QGFC))
 A245: "RE =
 B245: (F2) @IF((QETRTC+QRFC)/QSSTLC>1,1,(QETRTC+QRFC)/
 QSSTLC)
 A246: "COPSM =
 B246: (F1) +\$D\$206
 A247: "COPRM =
 B247: (F1) +\$F\$206
 A248: "FF =
 B248: (F2) @IF(QFC/QTHLC>1,1,+QFC/QTHLC)
 A249: "LF =
 B249: (F2) @IF(+QLUC/QTHLC>1,1,QLUC/QTHLC)
 A250: "SF =

B250: (F2) @IF(1-FF-LF<0,0,1-FF-LF)
A251: "RSR =
B251: (F2) +\$QRFC/(\$QETRTC+\$QRFC)
A252: "ESEM =
B252: (F2) @IF(H206>1,1,H206)
A253: "EREM =
B253: (F2) @IF(J206>1,1,J206)